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# FOREWORD

This service manual has instructions and procedures for the subject on the front cover. The information, specifications, and illustrations used in this manual are based on information that was current at the time this issue was written.

Correct servicing will give this engine a long productive life. Before attempting to start a test, repair or rebuild job, be sure that you have studied the respective sections of this manual, and know all the components you will work on.

Safety is not only your concern but everybody's concern. Safe working habits cannot be bought or manufactured; they must be learned through the job you do. By learning what CAUTION or WARNING symbol emphasizes, know what is safe — what is not safe. Consult your foreman, if necessary, for specific instructions on a job, and the safety equipment required.



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# MAINTÉNANCE STANDARDS

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# SERVICE GUIDE

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O refers to engines in regular service. © refers to new engines being broken in.

				Servicing intervals (hrs)						
V	/hat to service	What to do	Criteria	Daily (10)	75	150	300	600	1200	Remarks
		Check oil level and replenish.		0						
ication	Oil pan	Change oil.	7.0 liters (1.8 gallons)		0			0		
-ubr yste	Oil filter	Replace filter element.						0		
	Oil circuit	Clean by flushing.							0	
	P. 1.01.	Drain out sludge and condensate.						0		
	Fuel filter	Replace filter element.							0	
	Feed pump	Clean inlet connection.						0		-
tem		Check injection pressure.	100.01.1.2						0	
zuel sys	nozzles	De-carbon and clean nozzles and seats.	(1706 ± 71 psi)						0	
-	Fuel tank	Drain out condensate and sludge.	60 liters (16 gallons)			0				
		Clean by flushing.						0		
		Check water level and replenish.	-	0						
system	Radiator	Change coolant.	16 liters (4.2 gallons)					0		
		Clean radiator fins.			0					
coling	Fan belt	Check and adjust tension.	12 mm (1/2 in.) of deflection					0		
0	Thermostat	Check for performance.						0		
	Coolant circuit	Clean by flushing.			1				0	
Ē	A.L	Clean element.					0			Standard dut
ster	Air cleaner	Replace element.						0		conditions
Intake sy	Bolts and nuts on muffler and air cleaner	Check for tightness and retighten.			0			0		
al ent	Starter	Inspect brushes and commutator for wear; recondition as necessary.						0		
pme	D.44	Check acid level and replenish.			0					
Elec	Battery	Check electrolyte S.G.			0					
	Glow plugs	Inspect for condition.						0		
roper	Valve clearance	Check and adjust.	0.25 mm(0.0098 in.) for both intake and exhaust valves						0	
ine p	Major bolts	Retighten.							0	
Engi	Packings and seals	Inspect for leakage and repair.		0						

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# CONSTRUCTION AND FUNCTION



Description Exterior views

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Left-side view



1-Water pump	6-Oil pipe	11-Fuel inlet
2-Fan	7-Fuel feed pipe	12-Adjusting lever
3-Fan belt	8-Fuel filter	13-Starter
4-Fuel injection pump	9-Rocker cover	
5-Fuel feed pump	10-Governor	

Right-side view

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1-Cylinder head	7-Oil filler	13-Timing gear case
2-Crankcase	8-Thermostat	14-Mounting bracket
3-Service meter	9-Oil pipe	15-Oil level gauge
4-Flywheel housing	10-Fan	16-Drain plug
5-Exhaust manifold	11-Alternator	17-Oil pan
6-Rocker cover	12-Alternator bracket	

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### Longitudinal cross section



1-Thermostat 2-Water pump 3-Piston 4-Camshaft gear 5-Fan 6-Connecting rod 7-Crankshaft pulley

- 8-Crankshaft gear 9-Timing gear case 10-Oil pan 11-Oil strainer 12-Elbow 13-Rocker cover 14-Exhaust valve
- 15-Intake valve 16-Rocker shaft 17-Rocker bracket 18-Valve push rod 19-Cylinder head 20-Crankcase 21-Tappet
- 22-Flywheel housing 23-Flywheel ring gear 24-Flywheel 25-Camshaft 26-Crankshaft

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Transverse cross section



- 1-Exhaust manifold 2-Piston 3-Valve push rod 4-Tappet
- 5-Camshaft 6-Connecting rod 7-Oil pump 8-Oil strainer

9-Rocker arm 10-Oil filler 11-Glow plug 12-Oil injection nozzle 13-Fuel filter 14-Fuel injection pipe 15-Fuel injection pump 16-Fuel feed pump

## Specifications

	Model designation Type		S4E Water-cooled, 4-stroke cycle, in-line, turbulence-chamber type diesel
	Number of cylinders Bore x stroke, mm (in.) Piston displacement, cc ( Compression ratio Firing order Direction of rotation Fuel Crankcase lubricating oil	(cu in.)	4 94 x 94 (3.701 x 3.701) 2609 (159) 19 : 1 1 · 3 · 4 · 2 Clockwise as viewed from timing gear case side ASTM No. 2-D diesel fuel API service classification CC · class
	Dimensions	Overall length, mm (in.) Overall width, mm (in.) Overall height, mm (in.)	813 (32) 512 (20-1/8) 741 (29-1/8)
	Weight (dry), kg (lb)		290 (639)
neral data	Maximum output, PS/rpm Maximum torque, kg-m (ft-lb)/rpm		DD model         DPS model           35/2400         37/2400           14.5 (104.9)/1200         15.5 (112.1)/1200
Ge	<ul> <li>Compression pressure, kg/cm<sup>2</sup> (psi) (rpm)</li> <li>Minimum speed, rpm</li> <li>Maximum speed, rpm</li> <li>Cylinder sleeve</li> </ul>		20 (284.4) (150 ~ 200), minimum 650 ~ 700 2640 Dry type, special cast iron
	Number of piston rings	Compression rings Oil	2 1 (with spring expander)
	Valve arrangement		Overhead type
	Valve timing	Intake valves Open Close	30° before TDC 50° after BDC
	Exhaust valves Open Close		74° before BDC 30° after TDC
	Valve clearance	Intake, mm (in.) Exhaust, mm (in.)	0.25 (0.0098) (cold) 0.25 (0.0098) (cold)
	Starting system		Starting motor
6	Fuel feed pump	Model Manufacturer Cam lift, mm (in.)	ND - EP/KS22A Nippon Denso 6 (0.236)
el systen		Model Manufacturer	PES4A65B Nippon Denso
Fu	Fuel injection pump	Plunger diameter, mm (in.)	6.5 (0.256)
		Plunger lead Cam lift, mm (in.)	Right hand 8 (0.315)

	Injection timing		25 ± 1° before TDC					
	Governor	Type Model Manufacturer	Centrifugal, flyweight, all-speed type RSV Nippon Denso					
Fuel system – cont.	Manufacturer Nozzle holder Nozzle tip Type Injection nozzles Spray hole diameter, mm (in.) Spray cone angle Injection pressure, kg/cm <sup>2</sup> (psi)		Nippon Denso Bosch type, KCA17SD Bosch type, ND - DN0SD Throttle type 1 (0.039) 0° 120 (1706) ± 5 (71)					
	Fuel filter	Filter element Manufacturer	Paper-element type Nippon Rokaki					
	Capacity	Oil pan, liter (gal) Oil filter, liter (gal)	7.0 (1.8) 0.67 (0.18)					
E	Oil pressure	When operating, kg/cm <sup>2</sup> (psi) When idling, kg/cm <sup>2</sup> (psi)	3 ~ 4 (42.7 ~ 56.9) 1.0 (14.2), minimum					
-ubrication syste	Type Speed ratio to crankshaft Displacement, liter (cu in.)/min/rpm		Trochoid type 1/2 19.2 (1172)/2400 (engine speed), minimum Oil temperature: 50°C (122°F) Discharge pressure: 3 kg/cm <sup>2</sup> (42.7 psi)					
	Oil filter	Туре	Paper element type					
	Relief valve	Type Relief pressure kg/cm <sup>2</sup> (psi)	Piston-valve type 3 ± 0.3 (42.7 ± 4.3)					
	Capacity (excl. radi	ntor), liter (gal)	4.6 (1.2)					
	Water pump	Type Speed ratio to crankshaft Displacement, liter (cu in.)/min/rpm	Centrifugal type 1.2 : 1 100 (6103)/2520 (pump speed)					
Ystem	Belt	Type Manufacturer	Low-edge cog B-type "V" belt Mitsuboshi Belt					
Cooling s	Thermostat	Type Manufacturer Valve opening temperature, °C (°F)	Wax type Fuji Seiko 76.5 ± 2° (169.7 ± 3.6°F) (Fully opens at 90°C (194°F))					
	Fan	Type Number of blades Outside diameter, mm (in.) x pitch Speed ratio to crankshaft	Steel blade, pusher type 6 380 (14.96) x 30° 1.2 : 1					

	Voltage		24 V	
	Polarity		Negative	
	Glow plugs	Type Rated voltage-current Resistance value (at normal temperature)	Sheathed type 22.5 volts - 4.8 amperes 4.5 ± 0.5 ohms	
	Starter	Model Manufacturer Type Voltage - output Pinion/ring gear (No. of teeth)	M005T27671 Mitsubishi Electric Pinion-shift type 24 volts - 3 kilowatts 11/132	
Electrical equipment	Alternator	Model Manufacturer Type Voltage - current Rated voltage gener- ating speed, rpm Rated output gener- ating speed, rpm Maximum permissible, rpm Speed ratio to crank- shaft	AP4012B <sub>1</sub> Mitsubishi Electric 3-phase, enclosed type 24 volts - 12.5 amperes 1100 1900 7000 1.68	
	Regulator unit	Model Manufacturer Type Voltage regulator cut-in voltage Safety relay cut-in voltage	RMS4227 C9 Mitsubishi Electric Voltage regulator 27.5 ~ 30.5 volts at 3000 rpm (alternator speed) 5 volts, max.	

### Performance curves

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### Location of engine serial number

The engine serial number is embossed on the upper left hand side, directly above the starting motor, as shown below.



Engine serial number

### Crankcase



a special cast iron are press-fitted. The sleeves are of "dry" type: they are not in direct contact with engine coolant.

The crankcase as a whole is designed for high rigidity; the strength built into it is calculated to withstand, with an ample margin, the severe cyclic stress of complex nature imparted by the internal running parts.

These stresses vary from one part of the crankcase to another as in any diesel engine; the needed rigidity and durability are secured by the skirt section, which is sized more substantial and extending far below the level at which conventional-design crankcases are terminated to mate with the oil pan.

Each main bearing cap is fitted into and held by the crankcase, thus providing a more reliable support for the crankshaft at five places along its length.

### Main, thrust and big-end bearings

The five main bearings are of shell type, each consisting of two half shells, there being no distinction between the two: each may be located on top or bottom at the time of bearing installation in engine reassembly.

Crankshaft thrust is taken up at No. 5 journal. Three bearing plates, each in the form of half-ring, are used: two on the rear and one on the bottom front side of No. 5 bearing. These thrust plates are held in place by spring pins.

Connecting-rod bearings too are of shell type, each shell being of tri-metal design. The shell is essentially a kelmet metal fused to a steel backing base and covered by an overlay of a lead-tin alloy deposited by plating.



1-Main bearing 2-Thrust bearing

### Crankcase

The crankcase and cylinder block are in one, shaped as a single casting, into which the cylinder sleeves made of

### Air breather

The breather for letting fresh air into and letting out fumy air from the crankcase is located on the right-hand side of the engine. It prevents the pressure inside the crankcase from building up and thus minimizes the amount of lube oil getting into the combustion chambers.





Cylinder head



Cylinder head

### Cylinder head

The cylinder head is a single-piece casting, elaborately jacketed for improved cooling and also for greater structural rigidity. Exhaust ports, intake ports and precombustion chambers, each numbering four, are cast out inside the head. The exhaust ports open out to the right, and the intake ports to the left.

A total of 17 bolts secure the cylinder head to the cylinder block. The positions of these bolts are sequentially referred to in the tightening procedure, and are numbered sequentially, starting with "1" and ending with "17" as shown.



### Precombustion chambers

The precombustion chamber is formed with a cast-out space and an orifice piece – precombustion chamber jet – fitted into the cylinder head from its gasketed side. The shapes of the piston crown and the communicating orifice are such as to produce good turbulence even when the engine is running slowly. The glow plug, sticking out into the ante chamber – precombustion chamber – right beside the injection nozzle is a starting aid in cold weather.

### Cylinder head gasket

The gasket for sealing the joint between cylinder head and cylinder block is essentially an asbestos insulator sandwiched between two thin steel sheets, with its combustion chamber holes being edged in apron fashion by stainless steel grommets. The bottom surface in contact with the cylinder block is coated with a special sealing compound for improved sealing effect.

### Rocker cover

The rocker cover is aluminum alloy in material

### Valves, valve seats and springs

The intake valve has its disc sized as large as possible for increased suction efficiency.

The material of valves is heat-resistant steel having good hot-hardness. This steel used in exhaust valves is of a special kind having extra high resistance to high-temperature creeping, burning and oxidation, fatigue and thermal shock.

Each valve guide is fitted with a stem seal to prevent lube oil from entering the combustion chamber.

A special heat-resistant material of high-durability type is used in the valve seats of both intake and exhaust valves. Each valve is loaded by a single coil spring, whose turns are spaced apart with equal pitch.

The valve stem end is capped; the cap is in contact with the rocker arm and serves as the wearing member.



### Rocker arms, shaft and brackets

The rocker arm is a forging. Its tip for pressing down the valve stem is induction-hardened for increased resistance to wear. Its bore for admitting the rocker shaft is bushed; the bushing is lead bronze in material.



Rocker arm

The rocker shaft is hollow; one end of it is plugged while the other end admits lube oil for lubricating the rocker arms. The external surface of this shaft is hardened by nitriding at low temperatures.

The four rocker brackets are alluminum alloy castings shaped identically.

### Main moving parts

### Crankshaft

The crankshaft is a single-piece forging complete with balance weights. Its journals and crankpins are sized large to increase its rigidity, and are case-hardened by induction heating.

### Flywheel

The cast-iron flywheel carries the pilot ball bearing for holding the forward end of clutch shaft. The flywheel is doweled to the crankshaft and secured by four bolts.

### Ring gear

The ring gear is of a high carbon steel, its gears are hardened by induction heating. The ring itself is shrinkfitted to the flywheel. There are 132 gear teeth, each tooth being chamfered at its end facing the starter drive pinion in order to facilitate the meshing action of the pinion.



Main moving parts

### Pistons

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The pistons are of an aluminum alloy. The piston crown is recessed in such a way as to promote turbulence in both precombustion chamber and main chamber. As seen in side view, the piston is slightly tapered to present an increasingly large diameter toward its skirt; and, as seen in plan view, it is slightly oval. These two features are calculated to compensate its roundness for unequal thermal expansion at the operating temperature.

The piston pin hole has its axis slightly offset toward anti-thrust side in order to minimize piston noise. Each piston has its weight indicated on its crown by punching. All four pistons are required to have the same weight within a given tolerance so that the vibration of the main moving parts during operation will be minimized.

6-Connecting rod bearing cap 7-Connecting rod bearing (shells)

Front of engine Piston weight



Piston weight marking and pin hole offset

### Piston rings

Each piston is fitted with three rings: two compression rings (Nos. 1 and 2 as counted from top) and one oil ring. No. 1 ring and oil ring have their faces chromeplated. No. 2 ring is tapered. An expander coil is provided inside the oil ring to augment its elastic strength.



Piston pins

at each end.

The piston pin is hollow. Its sliding surface is case-

hardened by carburization. It is fitted to the piston and

carries the small end of the connecting rod in "full

floating" manner. The pin inserted into the piston is

retained in place by a snap ring fitted into the pin hole

Timing gear case

To the front end face of crankcase is attached a large mounting plate called the front plate. To this plate is bolted the timing gear case, in which the timing gears are housed.

The front plate is doweled to the crankcase by two pins.

The fuel injection pump is mounted on the front plate. Thus, the position of the injection pump relative to the engine is determined by these two dowel pins.

The forward end of crankshaft extends through the timing gear case to drive the cooling fan through crankshaft pulley. An oil seal is provided in the timing gear case to prevent oil from leaking along this part of crankshaft. The oil seal is doweled to take a given position.

### Timing gears

Helical gears made of high carbon steel are used to drive injection pump and camshaft from crankshaft through an idler. The teeth of these gears are finished by shaving for increased durability and high machining accuracy. Because of helical mesh, these gears run quietly and assure accurate timing action.



<sup>1-</sup>Camshaft gear (No. of teeth: 38) 2-Idler (No. of teeth: 43) 3-Grankshaft gear (No. of teeth: 19) 4-Injection pump gear (No. of teeth: 38) Timing gear configuration

### Camshaft

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The camshaft is a high carbon steel in material, and its cam surfaces are chill hardened.

The front journal has an oil hole, through which the lube oil under pressure flows from crankcase toward the valve mechanism over the cylinder head. A part of this oil lubricates the thrust face of camshaft.

### Tappets

The tappet is of flat type and shaped pot-like to admit the push rod into its hollow. It is cast iron in material; its cam-riding face is hardened by chilling. This design provides a lightweight tappet, resistant to wear and strong and thus suited to high-speed operation. All tappets, regardless of whether they are for intake valves or exhaust valves, are identical and, therefore, identified by the same part number.

The axis of the push rod is slightly offset from the center of the cam. This offset is calculated to cause the tappet to rotate during operation and thus to prevent its camriding face from wearing unevenly.



### Valve push rods

Made from carbon-steel pipe stock, the push rods has a steel ball welded to its bottom end and a caved-in piece welded to its top end. By the steel ball, the push rods stands on the spherical seat provided in the tappet and, by the caved-in top end, it bears against the adjusting screw threaded in the rocker arm. These contacting ends are hardened by carburization.

### Valve timing and valve lash

Valve lash is prescribed to be 0.25 mm (0.0098 in.) (cold) for both intake and exhaust valves, and the valve mechanism is timed to actuate the valve as follows:

INTAKE VALVES	open at 30° B.T.D.C.
	close at 50° A.B.D.C.
EXHAUST VALVES	open at 74° B.B.D.C.
	close at 30° A.T.D.C.



### Lubrication system



1-Piston 2-Oil filter 3-Crankshaft 4-Oil strainer 5-Rocker arm 6-Rocker shaft 7-Oil pressure alarm switch 8-Oil pump 9-Fuel injection pump 10-Water pump

Lubrication oil circuit

### Lube oil circulation

A trochoid rotary pump draws oil in the oil pan and delivers it under pressure to a full-flow oil filter, from which the cleaned oil is forwarded into the oil gallery inside the crankcase. From the gallery, the oil is distributed to the various parts of the engine. The pump is driven from the camshaft.

The oil filter is of a cartridge type containing a replaceable element through which the oil is forced. The element becomes increasingly dirty as the solid particles accumulate on and in its texture, thereby increasing the difference in pressure between inlet side and outlet side. The element is to be replaced before the differential pressure rises to a level at which the valve located in the bypass passage opens to allow the oil to bypass the element and flow directly into the oil gallery.

The bypass valve is an emergency means; it opens to avoid any critically reduced supply of lube oil to the running parts of the engine.

### Oil pan

The oil pan is a sheetmetal vessel shaped deeper in its front part to provide an oil sump. The oil sump is so located because of its position in the machine. The oil level gauge is located at its right-hand side.

The gasket, through which the oil pan is attached to the crankcase, is of rubberized cork.

### Oil strainer

The strainer is a metal screen fitted to the suction side of the oil pump. It serves the purpose of preventing any large-size solid particles from entering the pump.

### Oil pump

The pump is located inside the crankcase at its righthand rear portion. Its main shaft is driven from the skew gear formed of the camshaft. Being a trochoidal rotary pump, it has two rotors, inner and outer. Inner rotor is mounted on the shaft and drives outer rotor inside the pump case.



It is a positive displacement pump with its rotors in trochoidal mesh. This mesh is relatively free of abrasive action and enables the rotors to serve long and keeps up its pumping performance. Its design performance is as follows:

Pump speed	Displacement	Discharge pressure
1200 rpm	19.2 liters (1172 cu in.)/minute	3 kg/cm <sup>2</sup> (43 psi) at 50°C (122°F)

### Oil filter

The filter is mounted on the right-hand side of crankcase at its center part. The valve mentioned above for letting the oil bypass the element is actually a relief valve located in the center portion of the element. This valve is set to open when the differential pressure across the element rises to  $1.0 \pm 0.2$  kg/cm<sup>2</sup> (14.2 ± 2.8 psi); when the valve opens, the oil flows directly from inlet side to outlet side. The filter element must be serviced regularly or before the element becomes so dirty as to actuate this bypass valve.



The oil filter head has a built-in relief valve operating in response to the oil pump discharge pressure. This valve starts relieving when the pressure rises to  $3 \pm 0.3 \text{ kg/cm}^2$  ( $43 \pm 4.3 \text{ psi}$ ), thereby bleeding the excess oil to the oil pan and limiting the pressure of oil reaching the engine oil gallery to a constant level.



### Fuel system

### Fuel circuit

The fuel feed pump, mounted on the fuel injection pump body and forming a part of the injection pump unit, draws fuel from the fuel tank and delivers it through the fuel filter to the gallery inside the injection pump.

The injection pump is of individual plunger type, consisting of four plunger pump elements which are driven from a common camshaft. Each pump element delivers, intermittently, a shot of high-pressure fuel oil to its injection nozzle through its own injection pipe. These shots are synchronized to the diesel cycle in each cylinder and timed by the setting of the timing mechanism.

"Injection quantity," or the amount of fuel delivered uniformly by the four pump elements to the engine through their injection nozzles, is controlled from the accelerator through a linkage and automatically adjusted by the injection pump governor on the basis of engine speed and load requirements.

Each injection nozzle is spring-loaded to spray fuel at pressures not lower than 120 kg/cm<sup>2</sup> (1706 psi). A part of each shot of fuel reaching the nozzle returns to fuel feed pump through a leak-off pipe common to all four nozzles. The injection nozzle is of throttle type (as distinguished from standard type), and sprays fuel in atomized form into the precombustion chamber.

The governor built in the injection pump body is a mechanical all-speed governor, which limits the maximum and minimum engine speeds and actuates the control rack of the injection pump to maintain a constant engine speed under varying load condition at a speed level proportional to the position of the accelerator.



Fuel circuit

Fuel filter



Fuel filter - Sectional view

The fuel filter is located forward of the intake manifold. Its filtering element is made of a special paper designed to provide high filtering performance and large capacity.

Total area of filtration	850 cm <sup>2</sup> (132 sq in.)
Filtering element mesh	2 microns (µ)

### Fuel feed pump

The camshaft in the bottom section of the injection pump has an eccentric cam besides the cams for actuating the individual pump elements. By this eccentric cam, the pumping plunger of the feed pump is actuated to draw fuel through the inlet strainer and forward it with a discharge pressure limited to 2 kg/cm<sup>2</sup> (28.4 psi) to the injection pump.

A means of manually priming the fuel circuit ahead of the feed pump is provided in this pump. It consists of a plunger and a knob. Pushing the knob in rapid repetition sends the fuel forward. The fuel circuit from the feed pump through the fuel filter to the injection pump can be primed in this manner. This feature is utilized also in bleeding air out of the fuel circuit.



Fuel injection pump

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(1) Description

The pump body is an aluminum alloy casting and houses all the moving parts of pump elements and the camshaft. The governor housing is attached to one end of the pump body.

The camshaft is supported by two tapered roller bearings. Like the engine camshaft, it has four cams, one for each pump element, and is driven from the crankshaft through a train of gears arranged for a gear ratio of 2 to 1. For each two rotations of crankshaft, the injection pump camshaft rotates once.

The pump element consists of a plunger, barrel (cylinder), tappet, plunger spring, control pinion and spring-loaded delivery valve. The tappet rides on the cam and -pushes the plunger upward for each rotation of camshaft. As the plunger rises, the fuel in the barrel becomes compressed and is forced out through the delivery valve into the injection pipe. The upward plunger stroke, effective in compressing or pressurizing the fuel, is variable, and is varied by means of the control rack and pinion in the manner to be explained later.

The delivery valve, through which a shot of fuel is forced out into the injection pipe by each upward motion of the plunger, is essentially a check valve having a special function of quickly reducing the line pressure the moment the plunger begins to descend. This quick relief of line pressure is necessary to prevent the injection nozzle from dribbling at the end of each injection. How this is accomplished will become clear.

Cam lift	8 mm (0.315 in.)
Plunger diameter	6.5 mm (0.256 in.)
Delivery valve dia.	6 mm (0.236 in.); retraction volume 51 mm <sup>3</sup> (0.003 cu in.)/stroke
Injection order	1-3-4-2
Injection interval	90° ± 30′

Injection pump data

### (2) Pumping action

a. Pump element construction

The principal parts of the pump element are the cylinder (barrel) and plunger, as shown in this perspective view. Both are machined to extremely close tolerances; the plunger slides up and down in the bore of the cylinder with such a small radial clearance as to make the fit virtually oil-tight. The two - cylinder and plunger - are selectively combined during manufacture and must be handled as an inseparable pair.



A helical slot is milled in the top portion of the plunger. Called the control groove, this slot is communicated to the space above the plunger through a center hole (or a vertical groove in other designs).

The cylinder has a feed hole, through which the internal space is communicated to the fuel chamber or gallery outside. Fuel (under pressure) flows through this hole when the plunger is down. As the plunger rises, its top portion covers up the feed hole and, from this moment on, the plunger compresses the fuel above it until the control groove meets the hole. Effective stroke refers to that length of the plunger that keeps the feed hole covered during the upward stroke. This length or stroke can be increased or decreased by angularly displacing the plunger.

b. Pump element operation

The following description is referenced to the four cutaway views below:

- BOTTOM OF STROKE: Fuel flows into the inside space – delivery chamber.
- BEGINNING OF PRESSURIZATION: The cam pushes up the plunger and, as it rises, its top portion covers the feed hole.
- FUEL DELIVERY: Fuel is compressed; it forces the delivery valve against its spring to unseat the valve. From this moment, the fuel in the line from delivery valve to injection nozzle is pushed by the plunger.
- 4. END OF EFFECTIVE STROKE: Pressurization ceases and the delivery valve seats itself under the force of its spring. This valve has an annular recess. As the valve comes down, a small amount of fuel becomes trapped and is extracted from the injection line, so that the pressure ahead of the valve drops very sharply to enable the injection nozzle to snap into closed position. The amount of fuel so drawn back is called "extraction volume," an important factor of fuel injection.



Pump element operation

- (3) Injection quantity control
- a. Plunger rotating mechanism

The control sleeve, around which the control pinion is fastened, surrounds the lower portion of the cylinder (barrel). The sleeve has two nocks or slots in its bottom end; the driving face or flange formed of the plunger is engaged with these slots, so that the plunger rotates as the sleeve is turned.

The slots are long enough to permit the drive face or flange to slide vertically for full plunger stroke. The control pinion is engaged with the teeth of control rack.

b. Control action

The amount of fuel delivery, or injection quantity, per stroke is determined primarily by the effective pumping stroke of the plunger. The control groove milled out in the plunger being slanted, turning the plunger around its axis changes its effective stroke, and this turning is effected by moving the

three views of the plunger, cylinder, control pinion



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The seat of the delivery valve takes its position right above the barrel and is held down tight by the screw-in pipe connection. The valve has its guide portion fitted into the bore of the seat, and is capable of moving vertically. A coil spring urges the valve downward to keep the valve in contact with the seat face by its conical face.

It should be noted that a land is formed of the valve, a little above its guide portion, forming an annular recess between it and the cone. This recess assumes importance in regard to "extraction volume," mentioned previously.



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### a. Check valve action

During normal operation, the valve spring keeps the valve seated when the plunger is in non-injection position or, if it is in injecting position, when it is moving on downward stroke. As the plunger pressurizes the fuel in delivery chamber to overcome the force of valve spring, the valve unseats and lets out the high-pressure fuel into the injection pipe.

### . b. Retracting action

Consider the downward movement of the delivery valve following the end of fuel pressurization. The land enters the bore of the seat as the valve goes down, so that the delivery chamber becomes isolated from the injection pipe. The further downward movement of the land draws a small amount of fuel from the pipe and, when the valve cone has seated fully, this fuel is in the annular recess (called "extraction volume").

By this extraction, which occurs within a flash of moment, the injection pipe becomes instantly de-pressurized, thereby enabling the injection nozzle to snap into closed position and thus preventing the secondary injection or dribbling from occurring after each fuel injection.



(1) Beginning of extracting action (2) End of extracting action

### Type RSV governor

(1) Description

By type, the governor is a mechanical flyweight governor; by function, it is an all-speed governor operating in response to changes in engine speed to actuate the control rack in order to maintain engine speed at a constant level proportional to the set position of the accelerator. This governor function is in sharp contrast to that of a minimumspeed maximum-speed governor, whose control action is to limit the lowest and highest speeds of the engine, leaving the control of intermediate speeds to the operator.

The RSV governor too limits the lowest and highest speeds to provide a speed range over which it performs the governing action mentioned above. These limits, as well as its speed regulation, can be changed by means of adjusting lever and screws.



1-Adjusting lever 3-Idling set screw 2-Maximum speed stop screw Governor

To make full use of the advantages inherent in this governor, it is well to know its characteristics, which may be summarized as follows:

- (a) Compact and lightweight
- (b) Automatic supply of excess fuel for starting
- (c) Adjustable speed range and regulation for adapting the engine to each type of duty
- (d) Maximum injection quantity for each speed level can be adjusted to suit what the engine demands, by adding an adaptor spring.
- (2) Basic rules on governor setting

The governor is factory-sealed. Do not break the seal in an attempt to change the settings of critical parts unless you are qualified to do so.

- Maximum speed stopper is set to supply the right amount of fuel to the engine at the upper limit of the speed range. Disturbing this setting is likely to result in lack of output power or in overspeeding.
- Full load stopper is set to supply the right amount of fuel for full-load operation. Disturbing this setting is likely to result in lack of output power or in dirty or black exhaust smoke.

Adjusting screw for the swiveling lever is set at the position to which it has been backed away by 24 notches (6 rotations) or less from fully run-in position. Never try to back it away more than 24 notches or the adjusting screw will come off eventually to create a hazardous condition. (Refer to the part dealing with the adjustment of speed regulation.)

Unless you have overhauled RSV governors many times and can remember the overhauling procedure, be sure to refer to the disassembling and assembling procedures outlined in the latter section of this manual if you are to overhaul them.

Never re-use circlips, "E" rings and "O" rings removed in disassembly. Use new parts in reassembly.

### (3) Operating principles

The fundamental principles of a flyweight governor are schematically illustrated here. Arms (A) of flyweights, pivoting around point (C), push on the spring-backed block, whose key point is indicated as (B). The push is due to the centrifugal force of revolving flyweights.



When the revolving speed is constant, the push is in balance with the counter-force exerted by the compressed spring. This is an equilibrium condition. When the speed increases, for instance, the whole system seeks a new equilibrium, relocating point (B) and block to the dot-line position.

In the injection pump, point (B) is connected through a linkage to the control rack; the rack is pulled or pushed to vary the injection quantity, thereby lowering or raising the engine speed.

### (4) Control spring

It will be recalled that the spring rate (or constant) is the force required to stretch or compress it by unit length. Of course, this force is in the axial direction. For the tension lever, that part of the spring rate of the control spring, effective for pulling this lever, can be changed by angling the spring. Swiveling lever is the means of angling.



Note that hook hole (E) is in the arm of swiveling lever, and that this arm can be turned down (to reduce the tensile preload on control spring) or up (to increase the preload) by means of the adjusting screw. Thus, that component of the spring force acting on point (D) to turn tension lever can be set initially by positioning swiveling lever and also its adjusting screw. Our interest is not in how much force control spring exerts to tension lever but rather in that part of this force effective in turning the lever around its pivot point up above.

This arrangement of control spring (1) relative to tension lever (2) explains why, in the RSV governor, there is no need of using more than one control spring to change the governed speed (the speed which the governor operates to maintain) and to increase or decrease the speed regulation (or speed droop).

Note, also, that the block with its point (B) exerts push to tension lever in the direction of the arrow; this push is opposed by the pull of control spring. This opposing pull can be increased or decreased by turning the swiveling lever around its pivot (F). If tension lever happens to be off and away from full-load stopper, the increase or decrease of this pull (against a given push of the block) causes the control rack to move inward or outward, thus varying the rate of fuel injection to raise or lower the engine speed; consequently the push increases or decreases to introduce a new equilibrium.

- (5) Construction details
- a. Flywheight device

The two flyweights are mounted on bushing keyed to camshaft and secured by round nut. Since each flyweight can turn around the shaft, and because its inner tip has a roller, these two symmetrically arranged flyweights are capable of pushing on the flanged face of sleeve by their rollers through rolling contact.



Flyweights and related parts

Sleeve is arranged to slide axially along bushing and rotate around control block, there being provided a ball bearing between sleeve and block. The outer end of block is pinned to guide lever.

Thus, flyweights spread apart more or less depending on the running speed of camshaft, and push control block (toward the right in the illustration) more or less through sleeve. In other words, the rotating speed is translated into a linear force and hence a resultant linear movement of block by the medium of centrifugal force in opposition to the force of springs.

b. Levers and springs

We are now to see how the speed-dependent movement of guide lever is transmitted to the control rack of the injection pump. To do so, we must take note of levers and springs intervening in this transmission. To be examined are these levers and springs: adjusting screw related to torque control lever, torque spring on adjusting screw; guide lever; tension lever; control lever; idling spring; adaptor spring; control spring; swiveling lever; start spring.



2-Torque spring	12-Flyweights
3-Guide lever	13-Control spring
4-Tension lever	14-Swiveling lever
5-Torque control lever	15-Shackle
6-Pin A	16-Control rack
7-Control lever	17-Start spring
8-Idling spring	18-Governor housing
9-Tension lever pin	19-Adeptor
10-Adaptor spring	20-Lock nut
Governor - C	cross section

All these levers are movable, each being pivoted to the stationary part of the governor at its top or bottom end and pinned to another lever at its other end or intermediate point.

Tension lever and guide lever are pendent from a common pivot shaft (lever supporting shaft). Control lever pivots on a fork joint (stationary) by its bottom end, and its intermediate point is pinned to a halfway point of guide lever, whose bottom end is pinned to the control block, as mentioned before.

The top end of control lever is linked to control rack through shackle. Control spring is hooked between swiveling lever and tension lever. Start spring is hooked between the top end of control lever and a stationary anchor point. Adaptor spring is mounted inside the tension lever, and opposes the control block. Idling spring is mounted in the governor housing as if it were a cushion for the tension lever.

The shaft by which the swiveling lever turns extends through the housing and, outside the housing, is gripped by the adjusting lever. Turning this lever turns the swiveling lever inside. It is to this adjusting lever that the accelerator (lever or pedal) is linked; and it is by turning this adjusting lever that the governed speed is manually raised
or lowered. Once the adjusting lever is set, the governor operates to maintain a constant speed corresponding to the position of the adjusting lever.

The angular range of the adjusting lever is limited by the maximum speed stopper at the upper end of the range and by the stop adjust screw.

How these levers and spring cooperate will become clear in the subsequent description of the governor operation in three parts: ENGINE STARTING, IDLING CONTROL and MAXIMUM SPEED CON-TROL. Before we consider the operation, we shall discuss a special device - STOP DEVICE.

#### c. Stop device

In the standard RSV governor, which is not equipped with the stop device, turning the swiveling lever all the way to reduce the preload (by control spring) to zero causes the control lever to pull the control rack outward, thereby reducing the fuel injection to zero. This is the way a running engine is stopped.

The stop device, if provided, makes it possible to pull the control rack directly and independently of the adjusting lever (outside) and swiveling lever (inside). This device consists of a stop lever, a supporting lever, two springs and a screw, all associated with the control lever connected to the control rack through the shackle.



1-Start spring 2-Control lever 3-Stop lever 4-Screw 5-Supporting lever 6-Shaft 7-Pressure spring 8-Return spring 9-Control rack 10-Shackie A-Stop position

Pushing down the stop lever to its stop position (A) tilts the control lever outward and thus pulls the control rack to its non-injection position. This actuation is direct and fast.

In the standard-specification RSV governor, the adjusting lever is turned to bear against the stop adjust screw (both being located outside the governor housing) to stop the engine. Where the stop device is fitted to the governor, the stop adjust screw may be so positioned as to limit the lowest idling speed (beyond which the engine should stall). How the stop device operates will become clear later in the paragraph explaining this device again in reference to a schematic side view of the governor.

- (6) Governor operation
  - a. Engine starting

Suppose the accelerator is a pedal. Depressing the pedal turns down adjusting lever (shown as integral part of the swiveling lever in the schematic side view) toward the left, and pulls tension lever up against full-load stopper, pushing control block and governor sleeve.





7-Control block and govern 8-Flyweights 9-Swiveling lever 10-Control spring 11-Control rack

#### Engine starting

By this movement, guide lever and control lever tilt to the left, pushing the control rack to its starting position. This pushing action is assisted by start spring; this spring is designed to urge the control lever toward the left with a relatively small force.

When control rack is in its starting position, the injection pump delivers more fuel than is needed for full-load operation. The excess fuel is needed to help the engine fire up more easily.

b. Idling control

As the engine fires up, the operator would release the pedal; this makes the swiveling lever and adjusting lever move back to idling position to reduce the pull on tension lever and allow control lever to be tilted back by the push exerted by revolving flyweights upon control block. Control rack is therefore pulled out to idling position, at which the pump delivers fuel at a rate sufficient for keeping the engine idling.

Under the conditions indicated, an idling equilibrium is established between control block on the one hand and the total force of idling subspring and control spring, plus start spring.

responding automatically to any tendency of the speed to rise or fall and acting to cancel off this tendency by moving the control rack. Suppose, now, that some load is put on the engine.

The engine will then slow down and the flyweights contract, reducing the push of control block to allow the control rack to be pushed in. Consequently, the rate of fuel injection increases to raise the speed, and this increases the push exerted by the control block. In no time, the governor reaches an equilibrium state and the speed settles again at a constant level. This new level, however, is slightly below the previous one (because of the speed regulation) and, if it is too low, could cause the engine to stall.



#### c: Maximum speed control

Let us assume that the engine is idling with the adjusting lever bearing against the idling set screw: the adjusting lever is in idling position. If the lever is moved gradually toward the maximum speed stopper, the pull by the control spring increases gradually and, through the process of action and reaction involving the tension lever and control block, the control rack moves inward, increasing the fuel injection gradually to raise the engine speed. As the adjusting lever meets the full load stopper, the control block will be pushing the tension lever with a greater force, keeping the lever off the full load stopper. Thus, the control rack is prevented from moving too far inward beyond its "idling" position.



#### (7) Speed regulation and adaptor spring action

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One way of considering the effect of full engine load on its speed is to see what would happen when the load is increased gradually under the last-mentioned condition of the governor; namely, the adjusting lever is up against the maximum speed stopper and the tension lever is off the full-load stopper (with the flyweights spread wide apart). As the load increases, the engine slows down, and the tension lever closes in on the fullload stopper, causing the control rack to move in the direction for increasing fuel injection quantity. As the engine slows down still further because of the increasing load, the push by the control block against the tension lever diminishes further and, finally, the tension lever touches the full-load stopper.

The final speed, it must be noted, is lower than the original no-load speed by several percent despite the fact that the adjusting lever has been kept at the position limited by the maximum speed stopper. This difference in speed between no-load condition and full-load condition is due to the speed droop (or speed regulation) characteristic inherent in the governor of this type. "Speed droop" is desirable for the stability of an engine working under variable load condition.



After the tension lever touches the full-load stopper, what if the load on the engine increases to lower its speed and causes the flyweights to contract? The tension lever can no longer push back the control block; the control lever would be unable to push the control rack inward to increase the rate of fuel injection. This condition is avoided by means of the adaptor spring built in the tension lever.



Under the full-load condition, the tension lever behaves as if it were rigid and the control block is opposed by the adaptor spring. In other words, the state of equilibrium is produced by the adaptor spring and the control block. If the speed falls due to a rise in load, then the adaptor spring pushes the block to the left, causing the control rack to move inward, thus increasing the delivery of fuel to the engine.

Consider the reverse case: the load is decreased on the engine running slow with full load. In this case, the control block keeps on pushing the adaptor spring to prevent the speed from rising and, after compressing this spring fully, touches the tension lever. From this point onward, the block pushes the lever away from the full-load stopper as the load keeps decreasing.

(8) Adapting injection quantity to engine

The true function of the adaptor spring can be appreciated when the two important characteristics of a diesel engine and also of a plunger-type injection pump are recalled.

The amount of fuel delivered per stroke by the plunger in the pumping element is theoretically constant when the plunger is in a given angular position (with the control rack held in one position), regardless of its reciprocating speed (dependent on engine speed). Actually, this amount decreases as the speed rises. The reason is that the leakage of fuel, though extremely small, through the sliding clearance around the plunger decreases as the speed rises.

On the other hand, the amount of intake air drawn into each cylinder of a diesel engine is theoretically constant and equal to the "swept volume" of the cylinder; actually, this amount increases as the speed of the engine rises. The reason is that air has mass and takes a definite time to flow.

As long as the amount of air drawn into the cylinder is sufficiently large for the amount of fuel sprayed into it, there is practically no problem: the fuel will burn completely and the exhaust smoke will be clean. However, under full-load condition and, consequently, with a large amount of per-stroke fuel injected, a question has to be asked: is there a sufficient amount of excess air in the drawn-in air?

To summarize, where the control rack is held steady and the speed is increased, fuel injection quantity increases but intake air decreases. Under full-load condition, the "smoke limit" would be exceeded to result in a dirty exhaust smoke. To avoid this situation, the control rack must be pulled outward to decrease injection quantity, that is, the control block must be allowed to move toward the tension lever instead of being stopped by this lever. This requirement is met by the adaptor spring.

The graph shown here explains how the adaptor spring adapts injection quantity to the available air in the engine:



This graph assumes that the control rack of the injection pump is set for maximum injection quantity. Curve a - b' represents one setting, and curve a' - b another. With curve a - b', injection quantity would too much at speed N<sup>2</sup> but just right at speed N<sup>1</sup>. With curve a' - b, the quantity would be just right at speed at N<sup>2</sup> but too little at N<sup>1</sup>. What is desired for the air curve A - B is the modified curve a - b, which can be produced by causing the control rack to be pulled out by a small amount as the speed rises under full-load condition. The adaptor spring makes this possible.

#### (9) Governor characteristic



(10) Stopping the engine with stop lever

The stop device, mentioned in CONSTRUCTION DETAILS, is shown schematically, as associated with the bottom end of the control lever. With the stop lever in normal position, the control lever has its bottom end at the position for normal governor operation. Pushing down the stop lever



tilts the control lever to pull the control rack all the way out to the non-injection position, thereby causing the engine to stop.

#### Injection nozzle and nozzle holder

Referring to the cross section of the nozzle, the internal space of the nozzle and holder is filled with fuel. The leakoff line for passing the fuel back to feed pump is connected to nozzle holder. The leakoff passage drilled out in the holder is communicated to the space above distance piece, in which pressure spring is contained to load upon pressure pin.

The fuel inlet, to which the injection pipe (not shown) is connected, is provided in the holder. The inlet passage extends through the nozzle holder and opens out at the pressure chamber formed in the tip of injection nozzle. The needle valve has its conical face exposed to the fuel in the pressure chamber.

In operation, a shot of high-pressure fuel reaches the pressure chamber in the form of a pressure rise, causing the needle to unseat so that the fuel is forced out through the orifice into the precombustion chamber.

The pressure at which the needle unseats itself is determined by the compressed state of pressure spring. This preload can be varied for adjustment by changing the thickness of washer. The internal mating faces as well as the threaded portions are finished to extremely close tolerances to ensure the high oil-tightness required of this injecting unit.

Injection nozzle tip

Туре	NP-DNOSD	
Opening pressure	120 ± 5 kg/cm <sup>2</sup> (1706 ± 71 psi)	
Angle of fuel spray	0 deg.	



1-Retaining nut 2-Nozzle tip 3-Distance piece 4-Pressure pin 5-Pressure spring 6-Washer 7-Nozzle holder 8-Gasket 9-Nut

Injection nozzle - Cross section

Cooling system



Referring to the diagram, above, the coolant is set in forced recirculation by the water pump, which is a centrifugal pump driven by cooling-fan belt. The pump draws coolant from the lower tank section of radiator (4) and forwards it to the water inlet of crankcase (7).

Upon entering the middle section of the crankcase, coolant flows in the jacket to cool the cylinders; then it rises into jacket (5) of cylinder head (6) to cool the combustion chambers and areas around the intake and exhaust valves. From the forward end of the cylinder head, the coolant, now hot because it has taken as much heat as it can, flows into the inlet of thermostat (1).

The thermostat, responding to coolant temperature, controls the flow of coolant toward the radiator upper tank. When coolant temperature is low as when the engine has just been started up from cold state, the thermostat valve remains closed and all of the coolant is diverted back to the water pump inlet through bypass hose (3): under this condition, radiator (4) is bypassed by the coolant.

As the rising coolant temperature reaches 76.5°C (169.7°F), the thermostat valve begins to open increasingly wide and the coolant begins to flow to radiator (4)

at a rising rate of flow, with a corresponding decreases in the amount of coolant being bypassed. As the temperature reaches 90°C (194°F), the valve becomes full open, shutting off the bypass passage.

The probe for temperature gauge (2) is installed in the coolant outlet of cylinder head. It is with the signal produced by this probe that the gauge (located at the control station) operates.

#### Water pump

Pump case (6) is secured to cylinder block through cover plate (10). Pump shaft (3) is supported by two ball bearings (5) having a large bearing capacity. Lube oil is supplied under pressure from engine main oil gallery into space (7) formed around shaft between the two bearings.

Two oil seals (2) contain the oil so admitted to lubricate the ball bearings. Unit seal (9) prevents coolant from leaking out along the shaft. Impeller (8) is threadedly mounted on the inner end of the shaft, and pulley (1) is keyed to the outer end.

Crankshaft pulley and pump pulley (1) are in the speed ratio of 1 to 1.2. The pump capacity is 100 liters (6103 cu in.)/minute at 2520 pump rpm.



Water pump - Cross section

#### Thermostat

The thermostat is of wax type, designed to start opening its value at 76.5  $\pm$  2°C (169.7  $\pm$  3.6°F) of rising temperature and open it fully at 90°C (194°F), lifting it off the seat by 9 mm (0.35 in.) (maximum lift).



1-Pulley 2-Oil seal

3-Shaft 4-Spacer 5-Bell bearings 6-Case

8-Impeller 9-Unit seal

10-Cover plate

7-Space filled with lube oil

#### Cooling fan

The cooling fan has 6 blades and drives air against the core of the radiator. It is secured to the front end of the water pump pulley. Its outside diameter is 380 mm (14.96 in.); the pitch angle of its blade is 30 deg.

#### Fan belt

A single low-edge cog belt of Type B is used to transmit drive from crankshaft pulley to pump pulley. Its length is 41 inches (1047 mm).

### Electrical equipment Major equipment specifications

Equipment	Туре	Make
Starter	M005T27671	Mitsubishi Electric
Alternator	AP4012B1	Mitsubishi Electric
Regulator unit	RMS4227C9	Mitsubishi Electric
Glow plugs	Sheathed type	Hiyoshi Denso



#### Starter

(1) Specifications

Item	Specification
Motor type	Lever-shift pinion type with over- running clutch, built on totally- enclosed water-proof DC motor, M005T27671, compound wound
Voltage	24 volts
Output	3 kW
Yoke dia,	118 mm (4.646 in.)
Rating	30 seconds
Rotation	Clockwise as viewed from pinion side
Weight	Approximately 12.5 kg (27.6 lb)
No-load characteristic	4500 rpm, drawing not more than 50 amperes, at 23 volts
Locked-rotor characteristic	Developing 4.0 kg-m (29 ft-lb) and drawing not more than 700 am- peres at 9 volts
Switch-on voltage	16 volts, maximum

#### (2) Construction

The motor enclosure is of totally-enclosed type, designed tight against oil and water. The following cross section shows that the starter motor consists of three components: DC motor, engaging mechanism comprising an overrunning clutch, a shift lever and a pinion, and magnetic switch for actuating the lever.



Starter



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The inner race is integral with pinion (3), and the outer race presenting five cams is integral with splined sleeve (1). The sleeve is engaged with the splined part of the shaft, there being 10 splines. Five clutch rollers (2) are distributed around the inner race, each being pressed against the cam by a spring.

The splined sleeve is capable of sliding axially along the shaft and, when it does slide, the whole clutch moves axially. As the motor shaft rotates, the sleeve revolves with the shaft (when the sleeve is prevented from advancing any further with the pinion meeting the stopper) to drive the pinion in mesh with the ring gear of the flywheel. Under this condition, the rollers (2) are seized between inner race and cams. If the flywheel ring gear drives the pinion (after the firing up of the engine), rollers (2) become released and "freewheel" between inner race and outer race: under this condition, no drive is transmitted from engine side to the starter. The shift lever, extending from the magnetic switch, embraces the splined sleeve (1) by its forked end. The top end of this lever is held by the magnetic switch plunger; and the middle part is pinned. As the plunger jumps inward upon energization of the switch coil, the lever tilts to push the clutch toward the ring gear. When this lever-shifting action occurs in actual starting up of the engine, the motor will be rotating rather slowly to advance the clutch by the screw action due to the helical splines. Thus, the pinion advances rotatingly to mesh into the ring gear.

#### (4) Starter operation

How the starter is operated to crank the engine will be explained sequentially in reference to this schematic diagram of the starter circuit:



- a. Turning on the switch energizes the two coils of the magnetic switch. The initial current from the battery flows in these two coils, one of which is connected in series with the motor, so that the motor begins to run but slowly because the initial current is rather small. In the meantime, the two coils pull in the plunger to push the overrunning clutch toward the ring gear. The clutch slides along the helical splines and, for the reason already stated, advances smoothly to mesh its pinion with the ring gear.
- b. As the pinion meshes into the ring gear fully, the plunger is allowed to move in all the way, making its contactor to close, thereby permitting full current to flow through terminals (M) (B) into the motor. Consequently, the motor runs with full force to crank the engine. Under this condition, the coil in series with the motor is shunted so that practically no current flows in this coil, but the other coil (connected between terminal (S) and ground) remains energized to hold the plunger in pulled-in position.
- c. Turning off the switch (key switch) upon firing up of the engine de-energizes the holding coil, so that, by the force of the return spring, the plunger snaps back to the original position, thus disrupting the motor current and pulling the pinion away from the ring gear. The motor will

coast before coming to a halt: the counter-electromotive force (reverse voltage) occurring in the motor during this coasting helps the plunger move outward.

Alternator and regulator unit

(1) Alternator specifications and data

The alternator is complete with a rectifier.

Item	Specification	
Туре	Enclosed-type alternator, AP4012B	
Rated output	24 volts, 12.5 amperes	
Ground	Negative ground	
Outside diameter	128 mm (5.039 in.)	
Rotating direction	Clockwise as viewed from pulley side	
Weight	6.4 kg (14.1 lb)	

(2) Regulator specifications and data

Item	Specification	
Туре	Two-element type RMS4227 C9	
Elements	Voltage regulator, and safety relay (w/lamp)	
Weight	0.45 kg (1 lb)	

#### (3) Alternator construction

The alternator enclosure is of enclosed type. The field is a single coil mounted on the shaft and surrounded by two multi-pole magnets; excitation current is supplied through slip rings to the coil.

The armature coils are in three groups connected to provide a three-phase armature, and are mounted in the laminated core secured to the casing. Thus, the armature is stationary and the field is rotary. The rotor shaft is driven from the engine through the belt and pulleys.

The three-phase output leads of the armature are tied to the six-diode rectifier mounted inside the casing: three diodes are soldered to the positive heat sink and the other three to the negative heat sink. Cooling is made by the fan from outside.







#### (4) Charging system operation

The first of the two circuit diagrams to follow shows how the current flows from the battery when the key switch is turned on for starting up the engine. The second shows the flow of current for charging the battery. In these diagrams, attention should be directed to voltage coils VC<sub>1</sub> and VC<sub>2</sub> and current coils CC<sub>1</sub> and CC<sub>2</sub>, the four coils of the regulator unit.  $VC_1$  and  $CC_1$  actuate points  $P_1$ ;  $VC_2$  and  $CC_2$ actuate points  $P_2$  and  $P_3$ . The energizing current of a voltage coil is dependent on voltage; that of a current coil (which is connected as a shunt coil) is depending on current.

In these diagrams, the alternator unit (not shown) is represented by its three terminals (A) (F) (N). DC output voltage is available between terminal (A) and ground. Another voltage occurs between terminal (N) – neutral point of the rectifier – and ground. Terminal (F) is for receiving field excitation current from the terminal (A) of the alternator unit itself or, at engine starting, from the battery.

a. Engine starting

With the battery switch closed, turning on the key switch (by moving it to position 1) allows current to flow from the battery to the alternator field and also to the lamp. Field current at this time is small because of resistor R4. The lamp burns to tell that the alternator is not generating power.

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Turning the key switch to position 2 connects the battery to the starter to crank the engine through the sequence of actions already described.



Flow of current for starting the engine

Under this condition, points  $P_1$ ,  $P_2$  and  $P_3$  remain in the indicated state by their springs. Current flows in VC<sub>1</sub> but it is too small to be of any consequence.

b. Normal charging operation

As the engine starts up, the alternator unit begins to develop output voltage, so that VC<sub>2</sub> becomes energized (by the voltage between terminal (N) and ground (E), as mentioned before) to open points P<sub>2</sub> and put out the lamp.

The output current of the alternator unit flows through point P3 toward the battery and toward the alternator field through points P1 when the output voltage is up at the normal level.

If this voltage rises above the predetermined level, the current in  $VC_1$  increases to open points  $P_1$ 

against its spring, so that field current has to flow through resistor  $\mathbb{R}_s$  and is therefore smaller than before: this reduces the output voltage of the alternator unit. Actually, points  $P_1$  open and close in rapid succession to regulate the voltage at a relatively constant level.



Flow of current for charging the battery

c. Auxiliary circuits

Capacitor C and resistor R<sub>2</sub> are for absorbing the surge that occurs when points P<sub>1</sub> open. They prevent arcing from jumping between the contacting faces of points P<sub>1</sub>.

Current coil CC1 assists VC1 in closing and opening points P1 sharply so that the output voltage will be free from excessive ripples.

Current coil CC<sub>2</sub> and resistor Rs pass some current for the alternator field when points P<sub>1</sub> are open: they prevent the voltage from fluctuating so widely as to cause the lamp to flicker.

#### Glow plugs

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Each precombustion chamber of the engine has a glow plug. The plug is a starting aid and serves to warm up the chamber by "glowing" red with electricity supplied from the battery. It is of a sheathed type in construction.

The four glow plugs, one for each cylinder, are connected in parallel between the preheating line and ground. Failure of one plug, therefore, does not cut out the other three,



Glow plug

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# MAINTENANCE AND ADJUSTMENT

Inspection and adjustment of engine proper

#### Valve clearance adjustment

The valve clearance specification for this engine is 0.25 mm (0.0098 in.) for both intake and exhaust valves. This value assumes that the engine is at normal temperature, there being no temperature difference throughout the body of the engine. The checking and adjusting procedure is as follows:

- (1) Rotate the crankshaft slowly to bring the piston in No. 1 cylinder to Top Dead Center (TDC). This can be accomplished by observing rocker arms of No. 4 cylinder. As you turn the crankshaft, exhaust-valve rocker arm of this cylinder rises: stop turning the crankshaft just when intake-valve rocker arm begins to go down after exhaust valve rocker arm has come up all the way. Under this condition, adjust valve clearance in the usual manner on intake and exhaust valves of No. 1 cylinder, intake valve of No. 2 cylinder, and exhaust valve of No. 3 cylinder.
- (2) Turn the crankshaft one complete rotation (360°), and hold it there. Adjust the clearance on intake and exhaust valves of No. 4 cylinder, exhaust valve of No. 2 cylinder, and intake valve of No. 3 cylinder.



Adjusting valve clearance

Fan belt tesion adjustment

Give a thumb pressure to the middle section of the belt between alternator pulley and water pump pulley, and see how much this portion of the belt deflects by measuring with a rule. The deflection should be 12 mm (1/2 in.): if not, loosen the mounting bolts of the alternator holder to displace the holder in order to tighten or slacken the belt. After obtaining the prescribed amount of deflection, be sure to tighten the bolts good and hard.





#### Crankcase

Crankcase inspection

 Inspect the outside and inside surfaces for evidence of cracking. Visually examine the cylinder bores for scuffing, rusting, erosion or any abnormal wear. Using a straightedge, check the top face (for mating with cylinder head), front face (for mating with front plate) and rear face (for mating with rear plate) for flatness.



Checking crankcase top for flatness

(2) Make sure that the top face of the crankcase is flat within the repair limit specified below. If the limit is found to be exceeded, reface the top by using a surface grinder to make it flat within the specified flatness. Be careful not to remove any more stock than is necessary; if a stock of more than 1 mm (0.039 in.) has to be ground off, then the crankcase is done for.

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Item	Standard	Repair limit
Flatness of crankcase	0.05, max.	0.2 mm
top face	(0.002)	(0.008)

Cylinder sleeve inspection

 Using a cylinder gauge, take I.D. measurements in two directions (parallel and transverse to crankshaft axis) on each cylinder sleeve, at three places indicated below; and, from the six measurements taken, determine the amounts of wear (in comparsion with the specifications, listed below) and of uneven wear to see if the repair is exceeded; if so, rebore the sleeve to the next oversize.



Taking I.D. measurements on cylinder sleeves





Unit: mm (in.)

Positions for checking sleeve bore diameter

Item	Standard	Repair limit	Service limit
Cylinder sleeve I.D.	94 <sup>+0.035</sup> -0 (3.701 <sup>+0.0014</sup> )	+0.20 (+0.008)	+1.20 (+0.047)
Out-of-round	0.015 (0.0006), max.		
Taper	0.015 (0.0006), max.		

NOTE: "Taper" refers to the parallelness of bore wall.

- (2) Two oversizes are provided for: +0.25 and +0.5 mm (0.0098 and 0.0197 in.). After reboring, be sure to hone the bore to the specified oversize accurate within plus 0.035 mm (0.0014 in.) or minus 0 mm. Machining the bores of all four sleeves to the same oversize is preferred. (Pistons and piston rings are available for the two oversizes.)
- (3) If any sleeve bore is unevenly worn, determine the oversize, to which the sleeves are to be rebored, on the basis of the maximum wear noted. This will ensure perfect roundness in the oversized bores.

## NOTE

If the cylinder sleeves are found in good condition, with the wear far less than the repair limit, it is permissible rebuild the engine with replacement piston rings. In such a case, be sure to ream off the "ridge" and, as necessary, hone the bore.



Removing ridge with ridge reamer

#### Cylinder sleeve replacement

A cylinder sleeve badly scuffed or worn down beyond the service limit must be replaced by a new one and, in such a case, it is not necessary to replace the other sleeves.

If, however, the cylinder hole becomes damaged in the process of removing a sleeve, then the hole must be rebored for repair and, only in such a case, all the four sleeves must be replaced. The procedure of replacing the cylinder sleeve is as follows:

- Set the boring machine on the crankcase, and center it on the sleeve by referring to the lower part of the sleeve which is least subject to uneven wear.
- (2) Operate the machine to cut the sleeve until its wall thickness decreases to about 0.5 mm (0.0197 in.)
- (3) Taking care not to damage the cylinder hole,

break the sleeve and take it out of the hole.

- (4) Take measurements on the diameter of the cylinder hole and also on the O.D. of the replacement sleeve; and, from these measurements, see if an interference anywhere between 0.08 and 0.145 mm (0.00315 and 0.00571 in.) is available in the fit to be made; if not, try another sleeve to meet this interference requirement.
- (5) With a proper replacement sleeve having been selected, heat the crankcase in a bath of oil to about 300°C (572°F). Using the sleeve installer and hydraulic press, push the sleeve into the crankcase in one stroke, making sure that the top end of sleeve becomes flush with the gasketed surface (top) of crankcase.
- (6) Hone the installed sleeve to the standard I.D., that is, 94 plus 0.035 mm or minus 0 mm (3.701 plus 0.0014 in. or minus 0 in.).



Replacing sleeve

U	nit:	mm	(in.)

Item	Standard diameter	
Cylinder hole diameter	98 -0.010 -0.045 (3.858 -0.00039 -0.00177)	

For replacement sleeves, be sure to use the parts with this part number:

1. 1			24 1
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Part number	0.D.	1.D.
34407-00300	98 +0.10 +0.07 (3.858 +0.0039 +0.0028)	93 +0 -0.2 (3.66 +0 -0.0079)

Main bearing inspection

 Inspect each main bearing for evidence of wiping or fatigue failure, for scratches by dirt particles imbedded and for improper seating on the bore (bearing cap). On the basis of findings, determine whether the bearing should be replaced or not.

(2) Check each main bearing to be used in engine reassembly to see whether it will provide the specified radial clearance. This can be accomplished in this manner.

Install the main bearings on the crankcase, less the crankshaft, securing each bearing cap by tightening its bolts to 10.4 kg-m (75.2 ft-lb), and read the diameter in the two directions (A) (B), indicated below. Mike the journal and, from these readings, compute the radial clearance.

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Item	Standard	Repair limit
Radial clearance between main bearing and journal	0.05 ~ 0.115 (0.0020 ~ 0.0045)	0.20 (0.008)

If the computed clearance exceeds the limit, replace the bearing or regrind the journal and use the next undersize bearing. Two undersizes are available for this purpose: 0.25 and 0.50 mm (0.0098 and 0.0197 in.).



Measuring main bearing I.D.



Positions for miking main bearing

(3) Check each main-bearing shell for "crush." Shells found to be loose in the bore or have an excessive crush must be replaced. A crush of up to 0.04 mm (0.00157 in.), which will yield to a load of 500 kg (1102.5 lb), is prescribed.



#### Tappet and tappet hole inspection

- Inspect the riding face of each tappet for wear, contact pattern and crack. Replace defective tappets.
- (2) Check the radial clearance of the tappet in the hole against the repair limit, indicated below. If the limit is exceeded, then replace the tappet. If the hole is worn down so much as to provide an excessive radial clearance even with a new tappet, the crankcase must be replaced.

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Item	Standard	Repair limit	Service limit
Tappet- to-hole clearance	0.035~0.086 (0.0014~ 0.0034)	0.12 (0.0047)	+0.1 (hole) (+0.004)
Tappet hole diameter	22 <u>+0.021</u> _0 (0.8661+0.00083) _0		+0,1 (+0.004)

Camshaft hole inspection

- Inspect the inside surface of each hole for wear and scratch.
- (2) Mike the I.D. of respective holes and also the camshaft journals and, from the readings taken, compute the radial clearance available on each journal. If the clearance exceeds the limit, insert bushing or replace camshaft to reduce the clearance to the specification.



Miking camshaft hole

Unit: mm (in.)

Iter	'n	Standard	Repair limit	Service limit
Clearanc journal i hole	e of n	0.04 ~0.09 (0.0016 ~ 0.0035)	0.15 (0.0059)	
Ruching	Nos. 1 and 2	54+0.060 -0 (2.126-0)		
FIOIE	No. 3	53 <sup>+0.060</sup> -0 (2.087 <sup>+0.00236</sup> )		
Journal	Nos.1 and 2	54-0.04 -0.06 (2.126-0.00157, (2.126-0.00236)		-0.1 (-0.0039)
O.D.	No. 3	53-0.04 -0.06 (2.087-0.00157) (2.087-0.00236)		-0.1 (-0.0039)

(3) To install the camshaft bushings, use a group of drivers (puller, 30091-07300, adaptors, 30891-04500 and 30891-04600) after boring the ID of camshaft holes in the crankcase up to 57 mm ± 0 (2,244 ± 0 in.).



#### Cylinder head

#### Cylinder head inspection

Check the gasketed surface of the cylinder head for flatness by using a straightedge and thickness gauge as in the case of checking the crankcase surfaces. This check is to be made with the precombustion jets removed.

Use a surface grinder to reface the cylinder head, as necessary, to the specified flatness.



Checking cylinder head face for flatness

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Item	Standard	Repair limit
Flatness of gasketed surface of cylinder head	0.05 (0.002), max.	0.2 (0.008)

Valves and valve seat

(1) Inspection

De-carbon valve stems and seats; inspect both for wear and evidence of burning. Provided that the wear is within the service limit, grind smooth the seating face of each valve, removing the wear groove, if any, and finishing it to the specified angle of 45 deg. For this service, use a valve refacer.

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lter	n	Standard	Service limit
Valve stem	Intake	8 -0.045 -0.060 (0.315 -0.00177) (0.315 -0.00236)	-0.1 (-0.004)
diameter	Exhaust	8 -0.060 -0.075 (0.315 -0.00236) (0.315 -0.00295)	0.15 (0.0059)
Valve head t	hickness	1,5 (0.059)	1.2(0.0472) after refacing



Valve head thickness

- (2) Valve replacement
  - (a) Replace valves whose stems are found to have worn down to the service limit or head thickness is down to 1.2 mm (0.0472 in.) or under after refacing.
  - (b) Any valve showing evidence of cracking particularly in the head part must be replaced.
  - (c) "Valve sinkage" refers to a head face being below the combustion chamber surface, as shown, and is prescribed to be not greater than 1.3 mm (0.051 in.), the standard sinkage being 0.7 ± 0.2 mm (0.028 ± 0.008 in.). If the limit is reached, replace the valve or seat.
  - (d) Replace valve caps found excessively worn at the top face.



(3) Valve guide replacement

		Un	it: mm (in
Iten	1	Standard	Service limit
Valve stem	Intake	0.055 ~ 0.085 (0.0022 ~ 0.0033)	0,15 (0.0059)
clearance in valve guide	Exhaust	0.070 ~ 0.100 (0.0028 ~ 0.0039)	0.20 (0.0079)
Guide length hole	outside	17 ± 0.3 (0.669 ± 0.012)	

Where the stem-to-guide clearance is found to have exceeded the service limit, both valve and guide must be replaced. Apart from this clearance, check each guide to see if its I.D. near each end has enlarged and, if so, replace it.

Valve guides are press-fitted. To remove them, use a press and a drift, which is a special tool called the guide remover (31391-10500); to install, use the installer (34491-00400), another special-tool drift.





Installing valve guide

#### (4) Valve stem seal replacement

The valve stem seals should be replaced if the engine disassembled shows evidence of lube oil leaking into the combustion chambers along valve stems. The seal can be readily removed. When putting on a new seal, make sure that it fits snugly into the annular groove provided in the valve guide end,

If a valve has to be drawn out for one reason or another in the engine in regular use, be sure to have a replacement stem seal on hand for that valve. This is because the seal lip is certain to get scarred by the sharp-edged stem end.



1.Stern seal 3-Cylinder head 2-Guide Valve stem seal replacement

(5) Valve seat refacing

A valve seat badly worn or coarsened must be refaced by grinding in place. Use a valve seat grinder or a seat cutter and 400-grit emery cloth. Care must be exercised in using the seat cutter so that the cut will be even all around. After cutting, pinch the 400-grit emery cloth between the cutter and the seat and grind the seat face smooth.

Before installing the valve, lap the valve and seat, using the lapping compound. Check for contact pattern after lapping, using a paste of red lead to visualize the pattern. The pattern should be uniform and continuous.

(6) Valve seat removal and installation

To remove the valve seat, thin it in place by cutting with a rotary cutter, and break it loose with a chisel, taking care not to nick the counterbore in which the seat is seized by expansion fitting.

To insert the replacement seat, chill it first to about  $-80^{\circ}C$  ( $-112^{\circ}F$ ). This low temperature can be reached by immersing the seat in a pool of either or alcohol and by placing dry ice in the pool. Force the chilled seat into the counterbore, which has been trimmed clean and smooth, and calk around the seat with the calking tool (31391-13010 for intake valve or 31391-13020 for exhaust valve).



Installing valve seat by using calking tool

#### Valve spring inspection

Inspect each spring for cracks, and check it for squareness, free length and as-installed length against these specifications:

Unit: mm (in.)

Item	Standard	Repair limit
Valve spring	48.85	47.6
free length	(1.923)	(1.874)
Valve spring squareness	0.4/25, max. (0.016/0.98), max.	
As-installed	43	44
length	(1.693)	(1.732)



Checking valve spring for squareness

Rocker arm and rocker shaft inspection

- The valve-actuating tip of the rocker arm is subject to wear. If the tip face is excessively worn, replace the rocker arm.
- (2) Take diameter readings on the bushings and the rocker arm shaft, and compute the radial clearance from these readings. If the limit is exceeded, reduce the clearance by replacing the bushings or rocker arm shaft.

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Item	Standard	Repair limit
Bushing bore diameter	$\begin{array}{c} 20 \begin{array}{c} +0.021 \\ -0 \\ (0.787 \begin{array}{c} +0.00083 \\ -0 \end{array}) \end{array}$	
Rocker shaft diameter	20 -0.016 -0.034 (0.787 -0.00063 -0.00134)	
Shaft clearance in bushing	0.016 ~ 0.055 (0.0006 ~ 0.0022)	0.07 (0.0028)



Rocker arm

(3) Check to be sure that the oil hole drilled out in the rocker arm shaft is clear. When installing replacement bushings, be sure to align the oil holes.

#### Valve-clearance adjusting screw inspection

Examine each adjusting screw to see if its end face for contacting with the push rod is worn down excessively or if its threads are showing signs of failure; if so found, replace it by a new one.

#### Push rod inspection

Check push rods for deflection, and inspect them for wear at the end faces for contacting with the tappet and adjusting screw. "Deflection" refers to the runout exhibited by the push rod being rotated with its ends supported by such as "V" blocks.

Unit: mm (in.)

ltem	Standard	
Push rod distortion	0,4 (0.016), max.	

#### Exhaust manifold inspection

Inspect the manifold flange for cracks and distortion. If the flange faces are warped by more than 0.2 mm (0.0079 in.) when checked as shown, grind them smooth and flat. If any flange is found cracked, replace the manifold.



Checking exhaust manifold flange faces for flatness

Crankshaft inspection

(1) Crankshaft distortion

Support the crankshaft as shown and roll it to measure its deflection with a dial gauge. "Distortion" is one-half of the deflection (dial gauge reading); if it exceeds the repair limit, reduce it by bending the crankshaft in a press.

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Item	Standard	Repair limit
Crankshaft distortion	0.02 (0.0008), max	0.05 (0.0020)



Checking crankshaft for distortion

#### (2) Journal inspection

- (a) Inspect each journal for surface flaws such as roughing, scratches, pitting and burns, and, as necessary, repair the journals by grinding to the next undersize or replace the crankshaft.
- (b) Mike each journal to take a total of four readings to determine the wear, out-of-round and taper (cylindricity). If any of the limits is exceeded, repair by grinding to the next undersize or replace the crankshaft.



Miking crankshaft journals

#### MAINTENANCE AND ADJUSTMENT



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Positions for miking journal

It	em	Standard	Repair limit	Service limit
Crank-	Diame- ter	75 <sup>-0.030</sup> -0.050 (2.953 <sup>-0.00118</sup> ) -0.00197)	-0.15 (-0.0059)	-0.9 (-0.035)
shaft jour-	Out-of- round	0.01 (0.0004), max.	0.03 (0.0012)	
nals	Taper	0,01 (0.0004), max.	0.03 (0.0012)	

(c) Journal undersizes For the two undersize main bearings available, the journals are to be ground to these sizes:

Unit:	mm	(in.)

Journal undersize	Journals to be ground to:	
0.25 (0.0098)	74.75 -0.030 (2.9429 -0.00118) -0.050 (2.9429 -0.00197)	
0.50 (0.0197)	74.5 -0.030 (2.9331 -0.00118 -0.00197)	

(3) Crankpin inspection

(a) Inspect each crankpin for surface flaws such as roughing, scratches, pitting and burns, and, as necessary, repair the crankpins by grinding to the next undersize or replace the crankshaft.



Miking crankshaft crankpins



Positions for miking crankpin

(b) Mike each crankpin to take a total of four readings to determine the wear, out-of-round and taper. If any of the limits is exceeded, repair by grinding to the next undersize or replace the crankshaft.

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WILL'S STREET	1111111	1.45 1.43

Item		Standard	Repair limit
	Diameter	58 -0.035 -0.055 (2.283-0.00138) (2.283-0.00217)	0.20 (0.00787)
Crankpins	Out-of-round	0.01 (0.0004), max.	
	Taper	0.01 (0.0004), max.	

(c) Crankpin undersizes

When grinding the crankpins to the next undersize, be sure to finish each crankpin to the tolerance prescribed for the undersize, which is 0.25mm (0.00984 in.) or 0.50mm (0.01969 in.).

Crankpin undersize	Crankpins to be ground to:
0.25 (0.0098)	57.75 <sup>-0,035</sup> <sub>-0.055</sub> (2.2736 <sup>-0,00138</sup> )
0.50 (0.0197)	57.50 <sup>-0.035</sup> (2.2638 <sup>-0.00138</sup> -0.00217)

Unit: mm (in.)

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Try to keep the center-to-center distance between journal and crankpin within  $\pm 0.05$  mm (0.00197 in.) of 47 mm (1.850 in.). When grinding the crankpins to an undersize, be sure to size the corner radius (fillet) to 3 mm (0.118 in.). This applies also to the fillets of journals.



Crankshaft corner radius (fillet radius)

(4) Crankshaft end play

Check the crankshaft for end play, as shown, by using a thickness gauge at the thrust bearing. If a play of 0.3 mm (0.0118 in.) or more is noted, replace the thrust bearing.



Checking crankshaft end play

Item	Standard	Repair limit
Crankshaft end play	0.1 ~ 0.264 (0.00394 ~ 0.01039)	0.3 (0.012)

Unit: mm (in.)

The end play is due to the difference between the width of thrust bearing and the dimension (A) indicated below:



Journal width for thrust bearing

(5) Oil seal inspection

Inspect each oil seal, and replace it if it is badly worn, damaged or thermally fatigued at the lip surface. An oil seal suspected of poor sealing action evidenced by signs of leakage (noted upon engine disassembly) must be inspected more closely.

(6) Oil seal sleeve inspection

The outside surface of the oil seal sleeve is precision-machined and chrome-plated for greater wear resistance. Be sure to handle the sleeves carefully and protect this surface against damage.

Even a slightest scratch mark, not to mention of a dent or groovy wear, on this surface could result in oil leakage, and a sleeve with such a surface flaw must be replaced.

(7) Replacement of rear oil seal sleeve (for crankshaft gear)

To remove the sleeve, put a chisel to the outside surface of the sleeve and drive it in axial direction to stretch it. This will loosen the sleeve, making it ready to be drawn out. When driving, be careful not to damage the gear. To install the replacement sleeve, oil its bore and the crankshaft gear, using clean, fresh engine oil; hold the sleeve squarely and drive it into its position, keeping it trued up accurately.

(8) Inspection of crankshaft keyway and screw threads The forward end of the crankshaft is threaded and has a keyway. Visually examine the threads and keyway and, as necessary, repair them.

#### Pistons and piston rings

(1) Piston inspection

Inspect each piston for any abnormal wear of its sliding surface, for cracks at the crown and for evidence of melting or fusion. Examine the ring grooves for stepped wear and sloped wear. Replace pistons found in bad condition.

(2) Piston clearance in the bore

Mike each piston at the positions listed below; and by referring to the bore diameter, previously determined, of its sleeve, compute the radial Unit: mm (in.)

	Item	Standard	Service limit
Piston diame- ter (at skirt)	Standard size	93.86±0.015 (3.6953±0.00059)	
	0.25-mm (0.0098 in.) oversize	94.11±0.015 (3.7051±0.00059)	-0.2 (-0,008)
	0.50-mm (0.0197 in.) oversize	94.36±0.015 (3.7150±0.00059)	
	At piston crown	0.615~0.680 (0.02421~0.02677)	
	At No.1 land	0.465~0.530 (0.01831~0.02087)	
Piston	At No.2 land	0.415~0.480 (0.01634~0.01890)	
clear- ance	Just below oil ring	0.275~0.340 (0.01083~0.01339)	0.2
in bore	17 mm (0.669 in.) below oil ring	0.195~0.260 (0.00768~0.01024)	(0.000)
	37.5 mm (1.476 in.) below oil ring	0.155~0.220 (0.00610~0.00866)	
	At skirt	0.125~0.190 (0.00492~0.00748)	

clearance at each position. If the piston is worn down so much as to exceed the limit [-0.2 mm (-0.0079 in.)] at any of these positions, replace it.



Piston weight marking and pin hole offset

NOTE

Before reassembling the engine, make sure that the four pistons do not differ by plus or minus 3 grams (0.1 oz) from the weight indicated on the crown. If any of the pistons has to be replaced by an oversize piston, replace the other three, too, by those of the same oversize.

(3) Piston ring gap

Be sure that each piston ring has its gap within the service limit. Measure the ring gap with a thickness gauge, holding the ring fitted in a new sleeve.



Checking piston ring gap

	U	nit: mm (ir
Item	Standard	Service limit
Piston ring gap	0.3 ~ 0.5 (0.0118 ~ 0.0197)	1.5 (0.059)

#### (4) Piston ring clearance in groove

The clearance between a piston ring and its groove is specified for each. This clearance is dependent on the condition of the ring or the groove, or both. If the reading taken exceeds the repair limit, replace the ring and, if the replacement ring still provides an excessive reading, it means that the groove is worn so much as to require piston replacement.



Checking piston ring clearance

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Unit: mm (in.)

item		Standard	Repair Ilmit
Piston ring	No. 1	0.04 ~ 0.08 (0.0016 ~ 0.0031)	0.2 (0.0079)
clearance in the groove	No. 2	0.025 ~ 0.060 (0.0010 ~ 0.00236)	0.15 (0.0059)
	No. 3	0.025 ~ 0.060 (0.0010 ~ 0.00236)	0.15 (0.0059)

(5) Pin clearance in piston

Replace the piston or piston pin if the pin clearance, as computed from diameter readings taken on pin hole and pin, exceeds the service limit.

Unit: mm	(in )
WING HILL	111144

ltem	Standard	Repair Iimit
Piston pin diameter	$\begin{array}{r} 28 \begin{array}{c} +0 \\ -0.006 \\ (1.102 \begin{array}{c} +0 \\ -0.00024 \end{array}) \end{array}$	
Pin clearance in the piston	0 ~ 0.016 (0 ~ 0.0006)	0.05 (0.002)

Connecting rods

(1) Pin clearance in small end

Replace the piston pin or bushing if the pin clearance in the bushing, as computed from diameter readings taken on pin and bushing, exceeds the service limit.



Miking piston pin



To remove the bushing from and install it in the small end, the special tool must be used. Before installing the bushing, be sure to have the oil holes aligned.

ltem	Standard	Repair limit
Piston pin diameter	28 +0 -0.006 (1.102 +0 -0.00024)	
I.D. of bushing in ' small end	28 +0.045 +0.020 (1.102 +0.00177 +0.00079)	
Pin clearance in bushing	0.020 ~ 0.051 (0.0008 ~ 0.0020)	0.08 (0.003)

(2) Connecting rod alignment and big-end bearings

(a) Inspect each connecting rod for evidence of cracking, particularly at the corner radius parts of the "I" shank next to the big and small ends and also at the oil hole in the small end. Connecting rods found cracked or suspected of cracking must be replaced.

Check each connecting rod for alignment by fitting it to the aligner, as shown, and repair it by cold-working with a press, as necessary. The aligner tells whether the connecting rod is twisted or bent beyond the limit.



Checking connecting rod for bend

Unit:	mm	(in )
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Item	Standard	Repair limit
Connecting rod align- ment (twist and bend)	0.05 (0.002), max.	0.15 (0.006)



Checking connecting rod for twist

- (b) Inspect each big-end bearing for evidence of wiping or fatigue failure, for scratches by dirt particles imbedded in and for improper seating on the bore. Determine whether the bearing should be re-used or replaced on the basis of findings.
- (c) Check the radial clearance between crankpin and bearing; if the repair limit specified below is exceeded by the checked clearance, replace the bearing. Where the crankpin is to be ground to the next undersize, use a replacement bearing of that undersize.

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ltem	Standard	Repair limit
Crankpin diameter	58 -0.035 -0.055 (2.283-0.0014) (2.0022)	
Radial clearance between bearing and crankpin	0.035 ~ 0.100 (0.0014 ~ 0.0039)	0.20 (0.008)

The two bearing undersizes are 0.25 mm (0.00984 in.) and 0.50 mm (0.01969 in.).

As in the case of the main bearing, the clearance is to be determined from the crankpin diameter (determined as described in CRANKPIN IN-SPECTION) and also the diameter readings taken on the big-end bearing at the positions indicated and in the manner illustrated.



Measuring big-end

The big-end bearing fitted to the connecting rod must be secured by tightening the cap bolts  $8.5 \pm 0.5$  kg-m (61.5  $\pm 3.6$  ft-lb).



Positions for miking big-end bearing

(d) Check the contact pattern of big-end bearing on crankpin by fitting the big end in the normal manner to the crankpin, with the crankshaft laid out on the bench, and by using a paste of red lead or Prussian blue to visualize the contact. Be sure to tighten the cap bolts to the specified torque, that is, 8.5 kg-m (61.5 ft-lb). The contact should occur over at least 75% of the entire surface; if not, replace the bearing.

## NOTE

The above job of checking the contact pattern may be eliminated where the crankpin is ground to the specified tolerance and the bearing has been replaced. This is because a replacement bearing is precision-finished to ensure the specified extent of contact.

(e) Check each bearing shell for "crush." Shells found to be loose in the bore or have an excessive crush must be replaced. A crush of up to 0.04 mm (0.0016 in.), which will yield to a load of 350 kg (772 lb), is prescribed. As in the case of the main bearing shells, some "crush" is needed for securing a proper fit, without which the bearing might roll or jump in place, resulting in localized overloading and consequent flaking, burning or fatigue failure.

Check to be sure that the "crush" disappears to allow the bearing cap to mate the big end positively when the cap bolts are tightened to 8.5 kg-m (61.5 ft-lb).



(f) Check each connecting rod for end play in the manner illustrated, with the cap bolts tightened to 8.5 ± 0.5 kg·m (61.5 ± 3.6 ft-lb). Use a thickness gauge to measure the end play (which is the clearance between big end and crank arm). If the clearance measured exceeds the service limit, replace the connecting rod or big-end bearing.

Unit: mm (in.)

ltem	Standard	Service limit
Connecting rod	$0.15 \sim 0.35$	0.50
end play	(0.006 $\sim 0.014$ )	(0.020)



Checking end play of connecting rod

## NOTE

Before reassembling the engine, make sure that the four connecting rods are equal in weight within  $\pm 5$  grams ( $\pm 0.18$  oz) of the specification weight.

Flywheel inspection

- Repair the friction surface of the flywheel if it is found burnt, stepped, or grooved by rivet heads. If this surface is badly worn or damaged, the flywheel must be replaced.
- (2) Using a dial indicator, check the friction surface for flatness and face runout. A perfectly flat surface is no guarantee that the surface will not "run out" when it rotates.

Flatness can be checked, as shown, with the flywheel laid on a surface plate. To take the runout reading, secure the flywheel to the crankshaft in the normal manner and roll the crankshaft, with the spindle of the dial indicator put to the surface near its edge.



Checking flywheel for flatness



Checking flywheel for face runout

ltem	Standard	Repair limit
Flatness of friction surface	0.15 (0.006), max.	0.5 (0.020)
Face runout of friction surface	0.15 (0.006), max.	0.5 (0.020)

(3) Make sure that the flywheel securing bolts are in good condition, with their screw threads showing no signs of stripping. The ring gear with broken or badly worn teeth must be replaced. Clean the bushing for pilot bearing; if the bushing is abnormally worn or showing defective contact pattern, replace it.

Timing mechanism

Camshaft inspection

 Support the camshaft at No. 1 and No. 3 journals by "V" blocks, with the spindle of the dial gauge put to No. 2 journal, and roll the camshaft to measure its distortion (which is one-half of the deflection, that is, the dial gauge indication). Straighten the camshaft in a press or replace it, as necessary.

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Item	Standard	Service limit
Camshaft distortion	0.02 (0.0008), max.	0.05 (0.0020)



Checking camshaft deflection

(2) Mike each cam of the camshaft to read D<sub>1</sub> (cam height) and D<sub>2</sub> (diameter), and compute the difference between D<sub>1</sub> and D<sub>2</sub>. If this difference is less than the service limit, replace the camshaft.

Ite	m	Standard	Service limit
Cam height wear (D1 - D2)	Intake cam	$\begin{array}{c} D_1: 46.916 \substack{+0.1 \\ -0.3} \\ (1.847 \substack{+0.00394 \\ -0.01181}) \\ D_1 - D_2 = 6.684 \\ (0.26315) \end{array}$	D1 - D2 = 6.184 (0.24346)
	Exhaust cam	$\begin{array}{c} D_1: 45.944 \substack{+0.1 \\ -0.3} \\ (1.8088 \substack{+0.00394 \\ -0.01181}) \\ D_1 - D_2 = 7.344 \\ (0.28913) \end{array}$	D1 - D2 = 6.844 (0.26945)

Unit: mm (in.)

- (3) Inspect the camshaft journals for abnormal wear and damage; the camshaft must be replaced if any of its three journals is found in bad condition beyond repair.
- (4) Mike each journal of the camshaft in two directions, one being at right angles with another, at two places, front and rear. Measure each camshaft hole in the crankcase. Compute the clearance between the two; if the repair limit is exceeded, bore the hole up to 57H6<sup>+0.019</sup> mm (2.24<sup>+0.00015</sup> in.) and insert a bushing to bring it into the standard clearance range.

Ite	em -	Standard	Repair limit
Camshaft journal O.D. No. 3	54 -0.04 -0.06		
	and 2	$(2.126 \substack{-0.0016\\-0.0024})$	
	No. 3	53 -0.04 -0.06 (2.087-0.0016) (2.087-0.0024)	
Camshaft hole clear:	journal-to- ance	0.04~0.09 (0.0016~0.0035)	0.15 (0.0059)



Miking camshaft journals

Timing gear inspection

(1) Be sure that the backlash in each mesh is within the repair limit. If the limit is exceeded, reduce the backlash by replacing the worn gear. To measure backlash, use a thickness gauge: put the gauge squarely into between two gear teeth.



Checking timing gear backlash

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		nit: mm (in
ltem	Standard	Repair limit
Backlash	0.07 ~ 0.20 (0.0029 ~ 0.0079)	0.25 (0.0098)

(2) Check the radial clearance between idler bushing and shaft by miking. Compute the clearance from the readings taken and, if the repair limit is exceeded, replace the bushing.

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Item	Standard	Repair limit
Idler bushing I.D.	$36 \begin{array}{c} +0.025 \\ -0 \\ (1.417 \begin{array}{c} +0.00098 \\ -0 \end{array})$	
Idler shaft O.D.	36 -0.025 -0.050 (1.417 -0.00098) -0.00197)	
Bushing-to-shaft clearance	0.025~0.075 (0.00098~0.00295)	0.1

(3) Check the idler end play with a thickness gauge. Replace the thrust plate to reduce the play if the thickness gauge reading exceeds the repair limit.

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ltem	Standard	Repair limit
Idler end play	$0 \sim 0.1$ (0 ~ 0.004)	0.35 (0.0138)



Checking idler end play

(4) If the idler shaft has to be replaced, use the idler shaft puller to remove it, as shown. When installing the replacement shaft, check to be sure that the oil holes are aligned.



(5) Inspect the timing gears as follows:

(a) Camshaft gear

Replace the gear if its teeth show evidence of flaking or excessive wear, or if its keyway is galled, worn or otherwise disfigured. Make sure that the camshaft gear as mounted on the camshaft has no more end play than 0.4 mm (0.0157 in.): to check the end play, use a dial gauge. If the reading exceeds the repair limit, replace the thrust plate. (Remember, this gear is shrink-fitted to the camshaft.)

Item	Standard	Repair limiτ
Camshaft end play	0.05 ~ 0.112 (0.00197 ~ 0.00441)	0.3 (0.0118)

(b) Injection pump gear Inspect the gear teeth for damage and also the mounting bolt holes for malcondition. Replace the gear if found in badly damaged condition.

(c) Crankshaft gear Replace the gear if its teeth show signs of defective tooth contact, or are excessively worn or otherwise defective.

- (d) Idler gear Inspect the idler gear teeth and, as necessary, replace the gear.
- (6) Inspect the gear case for cracks, and for evidence of oil leakage at the part ahead of the crankshaft. A cracked case must be replaced. Inspect the crank pulley, too, examining the condition of the surface in contact with the oil seal and checking the keyway and key for wear. Replace the pulley if found in defective condition.

#### Lubrication system

#### Oil level check

The oil level gauge is located at the right-hand side of the crankcase, and carries two level marks, upper and lower. The oil pan contains about 7 liters (1.8 gallons) when the oil is up to the upper mark, and about 5.5 liters (1.5 gallons) when it is up to the lower mark. The oil pan should be kept filled to the upper mark.

#### Oil pump inspection

 Running clearance between outer rotor and inner rotor

Using a thickness gauge, check the clearance at various positions. If the reading exceeds the service limit, replace both rotors.



Checking rotor-to-rotor clearance

Item	Standard	Service limit
Clearance between inner rotor and outer rotor	0.013~0.15 (0.0005~0.0059)	0.25 (0.0098)

#### (2) Sliding clearance between rotors and cover

This clearance is required to be not greater than 0.15 mm (0.00591 in.). If this limit is exceeded, grind off the mating face of the body to reduce the clearance.

Unit	mm	(in )
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ltem	Standard	Repair limit
Clearance between rotors and cover	$0.04 \sim 0.09$ (0.0016 $\sim 0.0035$ )	0.15 (0.0059)



Checking rotor-to-cover clearance

(3) Radial clearance between outer rotor and pump body

Insert a thickness gauge into between outer rotor and body. If the clearance checked is greater than the limit, replace the worn part.



Checking rotor-to-body clearance

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Item	Standard	Service limit	
Clearance between outer rotor and body	0.2~0.28 (0.0079~0.0110)	0.5 (0.0197)	

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#### (4) Rotor shaft diameter

Inspect the shaft for damage, and check it for wear by miking. Determine the available clearance of the shaft in the pump body from the mike readings; if the service limit in terms of clearance value is exceeded or if the shaft is in badly damaged condition, replacement is necessary.

Unit: mm		nit: mm (in
Item	Standard	Service limit
Rotor shaft diameter	<sup>13</sup> +0 -0.015 (0.512+0 -0.00059)	
Shaft to body clearance	0.032~0.074 (0.00126~0.00291)	0.15 (0.0059)

#### Oil filter inspection

The filtering element is prescribed to be replaced after each 600 hours of operation or whenever its filtering performance is noted to have deteriorated. Inspect the element to see if it has any signs of rupture or fissure; and if so, replace it by a new one. Visually examine the filter case for distortion and cracks.

#### Relief valve inspection

First, inspect the valve and valve seat for seating contact, and check its spring for condition. Poor seating can be corrected by lapping. A broken or fatigued spring must be replaced.

Next, check the relief valve for relieving pressure in reference to the specification. Increase or decrease the spring preload, as necessary, to obtain a relieving (opening) pressure within  $\pm 0.3 \text{ kg/cm}^2 (\pm 4.27 \text{ psi})$  of  $3 \text{ kg/cm}^2 (\pm 2.7 \text{ psi})$ .

Unit: kg	/cm² (psi)	þ
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Item	Standard
Relief valve opening	3 ± 0.3
pressure	(42.7 ± 4.27)

#### Fuel system

#### Fuel filter inspection

Inspect the filter case and cover for cracks, distortion and damage and also for stripped threads at its threaded part. Replace the case and cover if found in defective condition.

Replace the connector bolts and plug if found with defective threads. The packings removed in disassembly must be discarded: be sure to use new packings in reassembly.

The filtering element is prescribed to be replaced at intervals of 1200 hours, and the filter itself be made free of sludges and condensation by draining at intervals of 300 hours.

## NOTE

The filtering element is of paper type not meant to be cleaned by washing. Use a softhair brush and compressed air to clean it.

Fuel feed pump inspection

(1) Check valve

Inspect the seating faces of the check valve for wear, and replace parts found in abnormally or excessively worn condition.

(2) Tappet wear

Mike the tappet and tappet hole to determine their diametral wear. Replacement is necessary if the amount of wear noted in comparison with the standard diameter is 0.1 mm (0.004 in.) or more.

Item	Standard	Service limit
Tappet diameter	7 (0.276)	0.1 (0.004)
Tappet hole diameter	7 (0.276)	0.1 (0.004)

Unit: mm (in.)

The overall play of tappet roller pin, involving the pin hole and roller, is prescribed to be not greater than 0.3 mm (0.012 in.). If this limit is exceeded, the whole tappet sub-assembly must be replaced.

Replace the roller if its diameter has worn down to the service limit.

ltem	Standard	Service limit
Tappet roller O.D.	<sup>15</sup> <sup>+0</sup> _0.027 (0.591 <sup>+0</sup> _0.00106)	-0.075 (-0.00295)

Unit: mm (in.)

#### (3) Pump housing damage

Inspect the housing for cracks, broken screw threads and other types of damage and repair or replace it as necessary.

#### (4) Priming pump wear

Inspect the piston and cylinder for scratch marks, wear and rusting. Check the seating faces of its valve for wear. An excessively worn or damaged seating face must be corrected by replacement.

#### Fuel feed pump testing

The feed pump is in satisfactory condition when it meets all of the test requirements hereunder enumerated;

#### (1) Suction performance

The pump should be capable of starting to deliver fuel in 45 seconds of its starting at 150 rpm.

#### (2) Discharge pressure

Run the feed pump at 600 injection-pump rpm, with the discharge side of the feed pump fully closed. Under this condition, the discharge pressure should be anywhere between 1.8 kg/cm<sup>2</sup> (25.6 psi) and 2.2 kg/cm<sup>2</sup> (31.3 psi).

Unit: ka/cm<sup>2</sup> (psi)

ltem	Standard	
Feed pump discharge pressure	1.8 ~ 2.2 (25.6 ~ 31.3) at 600 rpm	

#### (3) Capacity test

Run the pump at the speed specified below and open the discharge side more or less to hold the discharge pressure at 1.5 kg/cm2 (21.3 psi) (as read on the test gauge), with a measuring glass cylinder set up to receive the discharged fuel. Under this condition, the pump should deliver at least 900 cc (54.9 cu in.)/minute.

Item	Standard	Repair limit
Feed pump capacity	900 cc (55 cu in.)/minute, minimum at 1000 rpm	600 cc (37 cu in.)/minute, at 1000 rpm

#### (4) Air-tightness

Immerse the feed pump in a pool of fuel, with its discharge side plugged up. Apply an air pressure of 2 kg/cm<sup>2</sup> (28 psi) to its suction to see if any bubbles come out of the pump. Some air, however, will leak out through the clearance between its pushrod and housing. The pump is sufficiently air-tight if the amount of this leakage (with no leakage from any other part of the pump) is not greater than the value specified.



If a greater leakage than the specified value occurs, rework the pushrod hole with a burnishing broach to the oversize and replace the pushrod by a new oversize one.

Injection pump inspection

 Wear of pumping elements (plunger and barrel) Mike the width of plunger flange. If this flange is worn down to give a reading less than 6.95 mm (0.2736 in.) [= 7-0.05 mm (0.2756-0.0020 in.)], replace the plunger.



Width of plunger flange

Measure the width of the slot provided in the control sleeve. If this slot is worn down to give a reading greater than 7.02 mm (0.2764 in.), replace the sleeve.



Width of slot in control sleeve

Inspect the sliding surfaces of plunger and barrel for wear, scratch marks and evidence of pitting due to burning. If any defect is noted, replace the whole pumping element (plunger and barrel).

Whether a plunger is worn or not can be told from its appearance. A worn plunger has no surface luster. If any of the four plungers is in defective condition to require replacement, then all four pumping elements (plunger and barrel) should be replaced to ensure the uniform pumping performance for the four injection nozzles. This replacement is necessary also when any of the pumping elements fails to meet the following test requirement on the sliding clearance between plunger and barrel:

Fuel-tightness test on pumping element

After assembling the injection pump, install a test pressure gauge on the delivery valve holder, and run the pump at 200 rpm, with the control rack held in a position for low-load engine operation. The pressure gauge should be capable of indicating up to 300 kg/cm<sup>2</sup> (4266 psi) or 400 kg/cm<sup>2</sup> (5688 psi). When operated under these test condition, the pressure gauge should register a pressure not lower then 150 kg/cm<sup>2</sup> (2133 psi).

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Item	Standard	Repair limit
Fuel-tightness of plunger in barrel	150 ~ 200 (2133 ~ 2844) at 200 rpm	150 (2133)

In addition to the above requirement, each pumping element has to meet the following requirement as proof of a proper fit of the plunger in its barrel: Into the barrel removed upon injection pump disassembly, insert its plunger about two-thirds of the way, leaving a third of its length outside the
barrel, while holding the barrel horizontal; then angle up the barrel slowly by about 60°. This should cause the plunger to slide in all the way by its own weight to evidence a proper fit. If the plunger goes inward in a free-falling manner or becomes stuck on the way, then the pumping element must be replaced.



Checking pumping element for fit

(2) Delivery valve test

Each delivery valve must be tested for seating tightness. Before testing it, inspect its piston, valve seat and other parts for wear. If any part is excessively worn or if the seating contact is defective, replace the valve piston and seat as a unit.

Set up the assembled injection pump on the test stand, with a test pressure gauge installed on the delivery valve holder, as in the fuel-tightness test outlined above. [The gauge should be capable of indicating up to 300 kg/cm<sup>2</sup> (4266 psi) or 400 kg/cm<sup>2</sup> (5688 psi)]. Run the pump until the pressure gauge reads slightly above 150 kg/cm<sup>2</sup> (2133 psi). With a stop watch in the hand, pull the control rack to non-injection position and, at the same time, start clocking the time required by the fuel pressure (ahead of the delivery valve) to fall (due to leakage past the seated valve) 10 kg/cm<sup>2</sup> (142 psi). If this duration is not less than 5 seconds, then the delivery valve is satisfactorily tight.

Item	Service limit
Seating tightness of delivery valve in terms of duration for a drop of 10 kg/cm <sup>2</sup> (142 psi) from 150 kg/cm <sup>2</sup> (2133 psi)	5 seconds, minimum

When the injection pump is in disassembled state, check the fit of the delivery valve piston in the bore by holding the valve with fingers as shown. First, let down the valve all the way into the bore, and give thumb pressure to the bore. This should raise the valve a little and release of thumb pressure should allow it to settle into seated condition; if the valve remains seated without responding to the thumb pressure, its fit in the bore is too loose.



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Checking delivery valve for fit

(3) Wear of control rack and pinions

The control rack must be replaced if it is found distorted or bent or its rack teeth are excessively worn. After assembling the injection pump, check each pinion for backlash, and replace pinions found to exceed the service limit on backlash.

Unit: mm (in.)

ltem	Standard	Service limit
Pinion-to-rack backlash	0.15 (0.0059)	0.25 (0.0098)

Using a spring scale, check the force necessary to set the control rack in sliding motion. Repair or replace the control rack if it offers a resistance requiring a greater force to overcome than 150 grams (5.25 oz).



Checking sliding resistance of control rack

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Item Standard	
Sliding resistance of control rack	Not more than 150 (5.3) as starting pull

(4) Wear of tappets and rollers

The tappet roller consists of roller, roller bushing and pin. The overall wear of these three parts is to be checked by measuring the radial play of the roller with a dial gauge as illustrated. With the tappet sub-assembly held as shown, move the roller. up and down with a rod. If a reading greater than 0.3 mm (0.0118 in.) is obtained, replace the whole tappet sub-assembly.



Checking tapper roller for wear

Mike each tappet and tappet bore to determine the amount of radial clearance between the two. Replacement of worn parts is necessary if the clearance computed from micrometer readings exceeds the service limit.

ltem	Standard	Service limit
Radial clearance between tappet and tappet bore	0.02~0.062 (0.0008~0.00244)	0.25 (0.0098)

Check the outside diameter of each roller; if the reading as referred to the standard O.D. reveals a wear of 0.075 mm (0.00295 in.) or more, replace the whole tappet sub-assembly.

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Unit: mm (in.)

Item	Standard	Service limit
Tappet roller O.D.	$17_{-0.007}^{0}$ (0.669_0_00011)	-0.075 (-0.00295)

(5) Delivery-valve springs, plunger springs and seats Springs showing evidence of cracking must be replaced. Measure the free length of each spring; if the spring is found to be shorter by -0.5 mm(-0.020 in.) (plunger spring) or by -1 mm (-0.039 in.) (delivery-valve spring) than the smallest standard length, replace it.

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Item		Standard	Service limit
Free length	Plunger spring	49 <sup>+1</sup> <sub>-0</sub> (1.929 <sup>+0.039</sup> <sub>-0</sub> )	-0.5 (-0.020)
of spring	Delivery- valve spring	32±0.5 (1.26±0.020)	-1 (-0.04)

Inspect each plunger-spring seat; if the seat is found to have a recess (due to the seating end of the spring) of 0.1 mm (0.0039 in.) or more in depth, replace it.

(6) Wear of camshaft and its tappet roller bearings

Replace the camshaft if any of its cam surfaces is badly worn, grooved or otherwise damaged or if its keyway or threaded end portions are found defective. Mike the major diameter (cam height) of each cam to determine its wear in reference to the standard size and if the amount of wear is noted to exceed the service limit, replace the camshaft.

Un	代1	mm	(III.)

ltem	Standard	Service limit
Cam height	$32^{+0.1}_{-0}$	-0.2
(major dia.)	(1.26 <sup>+0.0039</sup> )	(-0.0079)

Inspect the tapered roller bearings for wear. A loose, rattling or otherwise defective bearing must be replaced.

(7) Condition of pump housing

Inspect the housing for cracks and examine the threaded parts for damage. If the housing is in cracked or damaged condition or if any of its tappet bore is found to have worn down excessively as a result of the check on the radial clearance of the tappet (in (4) above), replace the housing.

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ftem	Service limit
Pump housing tappet bore	24 - 0,15 (0,945 - 0,0059)

#### Injection pump testing and adjustment

A diesel engine cannot give the full performance it should be capable of even if it were in its best operable condition, unless the injection pump and governor serving it are equally in best condition. The following testing and adjusting instructions are for making sure that the pump is in such a condition and must be carried out with utmost care. Instructions on the governor will be given separately in the subsequent section.

The tests and adjustments, set forth under three headings, presupposes the use of special servicing equipment - the injection pump tester. It should be borne in mind that the pump and its governor constitute an inseprable set and must be tested and adjusted to meet the specified criteria before installing them on the engine.

- (1) Preparation
  - (a) Mount the reassembled injection pump on the tester.



Setting up the pump on the tester

(b) Attach the rack position measure. Remove from the governor these parts: maximum speed stopper, stop adjusting screw, idling spring, torque spring and adaptor spring. Push in the control rack toward the governor side as far as it will go, and set the rack position measure (95904-01060) at the starting position (zero mm).



Setting rack measure at 0-mm position

(c) Bleed air out of the injection pump, as follows: First, move the selector lever of the pump tester into "injection" position and turn on the motor switch to start up the motor.

Next, produce the prescribed delivery pressure by means of the pressure adjusting valve. Loosen the air bleeder screw on the pump to let out the trapped air, if any.



It is not necessary to "run" the pump in order to bleed air out.



Air bleeding

The pump is now ready for these operations: CONTROL RACK SLIDING RESISTANCE TEST, INJECTION TIMING ADJUSTMENT, and INJECTION QUANTITY ADJUSTMENT.

(2) Control rack sliding resistance test

Run the pump at 1000 rpm and measure the resistance with the hand spring balancer. The control rack should be capable of sliding without offering any resistance greater than 50 grams (1.8 ounces).



Checking rack sliding resistance

- (3) Injection timing adjustment
  - (a) Pre-stroke adjustment

"Pre-stroke" refers to the upward movement of the plunger (and hence of the tappet) and is measured as the distance the tappet travels from its lowest position until the feed hole (in the barrel) becomes closed by the plunger.

Loosen the nozzle holder overflow valve; move the selector lever of the tester to "injection" position; and start running the high-pressure pump of the tester by engaging the clutch; and, with the tappet clearance measuring device (95904-02100) securely installed, turn the camshaft to bring No. 1 plunger to bottom dead center (lowest) position.

From this position of camshaft, rotate it gently in normal direction to raise the tappet (and hence the plunger). In the meantime, fuel will be overflowing. Stop turning the camshaft just when the fuel ceases to overflow, and read the distance traversed by the tappet from its lowest position.



Measuring pre-stroke



Adjusting pre-stroke

	Unit: mm (in.)
Item	Standard
Plunger pre-stroke	1.95 ~ 2.0 (0.077 ~ 0.079)

If the reading is not within the standard range, adjust the pre-stroke by repositioning the tappet adjusting bolt vertically with the tappet wrench. Turning this bolt by about a half rotation varies the pre-stroke by about 0.5 mm (0.020 in.). After making this adjustment, be sure to have the lock nut tightened fully.

Check and adjust the other plungers for the prescribed pre-stroke by repeating the foregoing procedure.

(b) Checking and adjusting the angular position of the beginning of injection

The end of pre-stroke corresponds to the beginning of injection: the plunger begins to pressurize the fuel in the barrel when it has just closed the feed hole.

Take the position of No. 1 plunger at its beginning of injection as the reference angular position (0 deg.), and check the angular position at which each of the rest of the plunger begins to inject. Make sure that the beginning of injection comes within the 1 deg. tolerance of the angular value prescribed:

Item	Standard beginning of injection				
Angular	No. 1	No. 2	No. 3	No. 4	
spacing of injection timing	0°	89°30' ~90°30'	179°30' ~180°30'	269°30' ~270°30'	

Increase or decrease the pre-stroke to bring the beginning of injection, as necessary, into the tolerance allowed. Turning the tappet adjusting



Checking interval between injections

bolt by about a one-fifth (1/5) rotation changes the beginning of injection by about 1 degree.

(c) Tappet clearance measurement

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By the term "tappet clearance" is meant the marginal upward stroke of the tappet from its top dead center position, through which the tappet can be moved by forcing with a lever. This clearance is prescribed to be not less than the value specified below, and normally should not require any adjustment. Thus, the purpose of this measurement just for making sure that some clearance greater than the specified minimum is available.

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	Unit: mm (in.	
Item	Standard	
Tappet clearance	0.2 (0.0079), minimum	

Use the tappet clearance measuring device (95904-02100), as shown, and lever up the tappet which is pushed up all the way by the cam. If the reading happens to be less than the minimum, increase it by means of the tappet adjusting bolt, without causing the injection timing (beginning of injection) to deviate from the 1-deg. tolerance. Increasing the tappet clearance will increase the pre-stroke: be sure not to exceed the upper limit [2.0 mm (0.0787 in.)].



Measuring tappet clearance

The injection is properly timed in the injection pump when all four pumping elements have their pre-stroke and tappet clearance uniformly set to the specifications, with the four angular intervals between successive beginnings of injection are equalized within the given tolerance.

# (4) Injection quantity adjustment

"Injection quantity" is expressed in terms of cubic centimeters (cc) of fuel delivered by each pumping element for many strokes of its plunger. This quantity is measured as follows:

Close the nozzle holder overflow valve, so that the injection nozzle will spray out the fuel delivered by its pumping element. Have the high-pressure pump of the tester taken out of service by disengaging its clutch. Keep the selector lever in "injection" position.

# NOTES

- a) Keep the fuel supply pressure at 2.0 kg/cm<sup>2</sup> (28.4 psi).
- b) Be sure to use a measuring cylinder for each pumping element.
- c) To empty a measuring cylinder, in to which fuel has been sprayed, be sure to invert the cylinder and keep it in that position for at least 30 seconds before using it for the subsequent measurement.

Take three measurements on each pumping element, one measurement for each set of conditions, namely, pump speed, rack position and number of strokes, and compare the measurements taken with the specifications:

Pump speed rpm	Rack position mm (in.)	Strokes	Injection quantity cc (cu in.)	Difference cc (cu in.)
1000	8,5 (0,335)	200	7.2 ~ 7.8 (0.44 ~ 0.48)	0,4 (0.02)
1000	8.0 (0.315)	200	6.4 ~ 7.0 (0.39 ~ 0.43)	0.4 (0.02)
200	6.0 (0.236)	500	$5.0 \sim 8.0$ (0.31 ~ 0.49)	1.0 (0.06)

Injection quantity specifications

If any pumping element is noted to deliver too much or too little fuel, adjust it to bring its injection quantity into the range specified by displacing the control sleeve relative to the pinion. Loosening the pinion clamp screw allows the sleeve to be rotated in the pinion; turning the sleeve toward the governor side increases the injection quantity, and vice versa.

Be sure to set the pinion and sleeve accurately so that all four pumping elements will deliver the same amount of fuel without exceeding the limit, indicated above, on difference between the largest and the smallest measurement. Be sure to tighten the pinion clamp screw good and hard after adjusting the control sleeve.



Adjusting injection quantity

#### Governor inspection

Upon disassembling the governor, visually inspect the ball bearing (which is between control block and sleeve as a means of transmitting the push and pull between the flywheel device and the lever mechanism) for wear and damage. Examine the balls and the raceways carefully and, if any abnormal or excessive wear is noted, replace the bearing. Make sure that this bearing is in perfectly good condition: any rattle or abnormal noise is not permitted.

There are a total of five items to be checked and serviced in the governor during the process of reassembly. A repair limit is specified for each and, if the limit is reached, then the part or parts responsible must be repaired or replaced to bring the item (dimension) into the standard dimensional range.

# (1) Flyweight inspection

(a) Measure the clearance between the flyweight roller and roller pin. If the limit is reached, replace the flyweight assembly.

Unit: mm (in.)

ltem	Nominal diameter	Standard	Service limit
Flyweight roller and roller pin clearance	8 (0.315)	0.025 ~ 0.062 (0.00098 ~ 0.00244)	0.10 (0.00394)

(b) Check the contact surfaces of the flyweight roller and governor sleeve. If any excessive wear or damage is found, replace the flyweight assembly.

Item	Nominal diameter	Standard	Service limit
Flyweight roller O.D.	16 (0.63)	0 0.11 (0.0043)	0.25 (0.0098)

- (2) Inspection of parts related to control lever
  - (a) If the control block is worn down at its end and chrome plating is off, replace the control block or guide lever.
  - (b) Measure the clearance between the holes of the tension lever and guide lever, and supporting lever shaft. If the limit is reached, replace them. If any excessive uneven wear in holes and stepped wear on the shaft are found, replace the supporting lever shaft and levers.

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Item	Nominal diameter	Standard	Service limit
Tension lever and guide lever holes, and sup- porting lever shaft clearance	8 (0.315)	0.013 ~ 0.05 (0.0005 ~ 0.0020)	0.10 (0.00394)

(c) Measure the clearance between the schackle pin and control rack hole. If the limit is reached, replace the schackle or control rack.

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~		N. M.	111111	711141

Item	Nominal diameter	Standard	Service limit
Schackle pin and control rack hole clearance	5 (0.197)	0.015 ~ 0.0056 (0.0006 ~ 0.00022)	0.08 (0.0031)

# (3) Swiveling lever inspection

Measure the clearance between the swiveling lever shaft and bushing. If the limit is reached, replace the bushing, When replacing the bushing, replace "O" ring and oil seal, too.

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Item	Nominal diameter	Standard	Service limit
Swiveling lever shaft and bushing clear- ance	11 (0.433)	0.016 ~ 0.07 (0.0006 ~ 0.0028)	0.15 (0.0059)

(4) Torque control lever inspection

Measure the clearance between the torque control lever bushing and supporting pin. If the limit is reached, replace the lever.

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ltem	Nominal diameter	Standard	Service limit
Torque con- trol lever bushing and supporting pin clearance	8 (0.315)	0.026 ~ 0.056 (0.00102 ~ 0.00220)	0.10 (0.00394)

## (5) Spring inspection

When the governor is adjusted, check various springs and determine if they are defective or not. Expecially at disassembly, check them for bent, damage, fatigue and rusting. Check the control spring at its hook part, too. Replace springs found in abnormal condition.

#### Governor testing and adjustment

As to the meanings of various technical terms used in the following instructions regarding governor performance and adjustments, reference must be had to the governor characteristic curves, below, and to the table of standard adjustment data, carried at the end of this part.

The procedures which follow assume that the injection pump has been properly set for injection timing and injection quantity as outlined in the preceding part, and that the injection pump unit (complete with its governor less the adaptor spring and idling spring) is set on the pump tester.

(1) Governor adjustments

After servicing the governor, securing the prescribed clearances in the various running parts, make four adjustments: adaptor adjustment, maximum-speed control adjustment, low-speed control adjustment and torque spring adjustment. Each adjustment will be explained in reference to the characteristic curves.

(a) Preparation

Install the angular scale plate (protractor) on the governor housing to read the angular position of the adjusting lever. (vertical =  $40^{\circ}$ )



Protractor on governor housing

- (b) Maximum-speed adjustment
- 1. Setting the full-load stopper
  - Run the injection pump at the speed corresponding to "Nc," which is indicated in the characteristic curve diagram. Turn the adjusting lever in the direction for raising the speed: this tensions the control spring, making the control rack move inward. Stop moving the adjusting lever when the rack comes to the position corresponding to "Rc" [=  $8.8 \pm 0.1$  mm (0.346  $\pm 0.004$  in.)], and secure the adjusting lever there tenstatively. Rack position "Rc" is for maximum injection quantity. Bring the full-load stopper into contact with the tension lever under this condition.



Governor characteristic curves



Setting full-load stopper

 Setting the maximum-speed stopper Slowly raise the speed from "Nc" to see when the control rack begins to come out (in the direction for decreasing injection quantity). The speed at which this should occur is prescribed to be 1200 rpm (= Nd). This requirement can be met by unlocking the adjusting lever (which was secured in the preceding step), by turning the lever, and by slowly raising the speed. Upon locating that position of adjusting lever at which the control rack begins to come out at 1200 rpm of rising speed, bring the maximumspeed stopper into contact with the lever, thereby setting the maximum-speed position of adjusting lever. The lever angle for this position is prescribed to be  $47^{\circ} \pm 5^{\circ}$ .

3. Speed regulation adjustment

What "speed regulation" signifies was explained previously: it refers to the difference between two governed speeds: no-load speed and fullload speed for a given position of the adjusting lever. It is expressed as a percentage of full-load speed:

Speed regulation

Generally speaking, the smaller the speed regulation, the better is the engine control; but some regulation is necessary for the sake of running stability and the smallest regulation for the type of all-speed governor as the present one is limited by the governor mechanism. Moreover, each engine runs best when the governor is set to provide the regulation specified for the engine.

For the maximum speed position of adjusting lever, speed regulation is represented by that portion of the characteristic curve from point "d" to point "e" for the differential speed from "Nd" to "Ne."

In the present governor, the regulation can be changed by means of the adjusting screw provided in the swivel lever. Tightening this screw increases the tension of control spring to reduces the value of regulation, and vice versa. With a small regulation, the curve "d"  $\sim$  "e" is sharper and, if it should be too sharp, the governor would become too sensitive.

It is important to note that the speed regulation of an engine unit (complete with its injection pump and governor) is determined not solely by the governor setting but by the characteristics of the engine (which were explained earlier in the discussion of the torque spring). For the present engine, a proper regulation will obtain when the adjusting screw (on the swivel lever) is set as follows: Drive in the adjusting screw as far as it will go in, and then back it away by four (4) rotations. Four notches of this screw correspond to one (1) rotation.

Backing away the screw increases the regulation (making the curve "d"  $\sim$  "e" less sharp). Never back it away by more than 24 notches (6 rotations) or the threaded engagement of the screw will be so small as to invite a hazardous condition.



Setting adjusting screw

 Re-adjustment of maximum-speed stopper setting

Changing the speed regulation by tightening or loosening the adjusting screw is, in substance, changing the tension of the control spring. For this reason, after each repositioning of the adjusting screw, the position of maximum-speed stopper for determining "Nd" (the speed at which the governor begins to perform its highspeed control action) must be changed to raise or lower "Nd" to a proper level by repeating the process described above.

# NOTE

In the table of standard adjustment data, the angular position of the adjusting lever assumes that  $40^{\circ}$  is vertical.

- (c) Adaptor adjustment
- Move the adjusting lever to make it bear against the maximum-speed stopper, and install the adaptor in its normal position in the tension lever.
- Run the pump at "Na" (= 900 rpm). This should move the control rack inward to the 9.2 ± 0.1 mm (0.362 ± 0.004 in.) position (= Ra); if not, change the thickness of shim plate (3) indicated in this illustration
- 3. Raise the pump speed from "Na" to "Nb." This

should pull the control rack out to the position "Rc" [=  $8.8 \pm 0.1 \text{ mm} (0.346 \pm 0.004 \text{ in.})$ ]; if not, tighten or loosen adaptor screw (4).

# NOTE

The rack movement from "Ra" to "Rc" corresponds to the amount of compression of adaptor spring (5), which is referred to as "adaptor stroke." Tightening (or driving inward) adjusting screw (4) elongates this spring to increase the stroke, and vice versa.



1-Tension lever 2-Lock nut 3-Shim plate 4-Adaptor screw 5-Adaptor spring

Adaptor adjustment

(d) Low-speed control adjustment

This adjustment is related to the low-speed control curve in the diagram given above, and is effected as follows:

- Run the pump at "Ni" (= 275 rpm), with the adjusting lever bearing against the maximumspeed stopper; turn back the adjusting lever until the control rack comes to "Rh" [= 5.5 ± 0.1 mm (0.217 ± 0.004 in.)]; and secure the adjusting lever there.
- Install the idling spring. Drive in the adjusting screw of this screw, as shown, until the control rack moves in and comes to "Ri." Be careful not to set this adjusting screw too far inward or the no-load maximum speed will rise too high in operation.
- Lower the speed to "Ng" (= 200 rpm); this should cause the control rack to move in and comes to "Rg" [= 11.0 mm (0.433 in.) minimum] owing to the action of the idling spring.



Setting idling spring for low-speed control

- 4. After completing the foregoing adjustments, stop running the pump and turn the adjusting lever in the stopping direction until the control rack comes to 1 mm (0.0394 in.) position. Set the stop adjusting screw to limit the stopping end of adjusting lever stroke, and secure the screw by tightening its lock nut.
- (3) Matching the pump to the engine

After adjusting the governor according to the procedures set forth in (2) above, check the injection quantity by running the injection pump as outlined in (4) INJECTION QUANTITY ADJUSTMENT, INJECTION PUMP TESTING AND ADJUSTMENT. Use a 500-cc (30.5-cu in.) measuring cylinder to receive and collect the fuel delivered by the four pumping elements, with the adjusting lever set at  $47^{\circ} \pm 5^{\circ}$ . For this test, take two readings, one by running the pump at 900 rpm and the other at 1200 rpm. If the readings differ from the values indicated in the table of standard adjustment data, adjust the pumping elements.



Measuring injection quantity

## Table of standard adjustment data

APPLICABLE TO: Injection pump, 090000-9721 (ND-PES4A65B320RND972) Governor, 090800-4030

(ND-EP/RSV250-1750A2/302ND403)

## 1. Injection timing adjustment

- Direction of rotation: Clockwise as viewed from drive side
- (2) Injection order: 1 − 3 − 4 − 2
- (3) Interval between successive injections: 90° ± 30'
- (4) Pre-stroke: 1.95 ± 0.05 mm (0.077 ± 0.0020 in.)
- (5) Tappet clearance: 0.2 mm (0.0079 in.) minimum

#### 2. Injection quantity adjustment

The listed values are based on these conditions: 1) injection nozzles, 093400-0090, 2) nozzle opening (injecting) pressure, 120 kg/cm<sup>2</sup> (1706 psi), 3) fuel supply pressure, 2.0 kg/cm<sup>2</sup> (28.4 psi), and 4) highpressure tester pump, 1.6 mm (0.0630 in.) dia. x 6 mm (0.2362 in.) dia. x 600 mm (23.622 in.).

Pump speed rpm	Rack position mm (in.)	Pumping strokes	Individual injection cc (cu in.)	Permissible difference cc (cu in.)
1000	8.5 (0.335)	200	7.2~7.8 (0.44~0.48)	0.4 (0.02)
1000	8.0 (0.315)	200	6.4~7.0 (0.390~0.43)	0.4 (0.02)
200	6.0 (0.236)	500	5.0~8.0 (0.30~0.49)	1.0 (0.06)

#### 3. Governor adjustment

The listed values are based on these conditions: 1) governed speed range,  $275 \sim 1200$  rpm, and 2) swivel-lever adjusting screw setting, backed away about four (4) rotations from the fully tightened position.

# (1) High-speed control

Lever angle	Speed rpm	Rack position mm (in.)
47° ± 5°	1100	8.2 ± 0.1 (0.3228 ± 0.0039)
	1230	7.0 ± 0.1 (0.2756 ± 0.0039)
	1290	≦ 4.0 (0.1575)

Lever angle	Speed rpm	Rack position mm (in.)
High-speed control lever angle MINUS $26^\circ \pm 3^\circ$	275	5.5 ± 0.1 (0.2165 ± 0.0039)
	330	5.0 ± 0.1 (0.1969 ± 0.0039) with sub-spring
	200	≧ 11.0 (0.4331)

NOTE: Figures in box \_\_\_\_\_ are for initial lever setting.

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(3) Control by adaptor action

Lever angle	Speed rpm	Rack position mm (in.)
51° ± 5°	400	10.4 ± 0.1 (0.4094 ± 0.0039)
	650	8.8 ± 0.1 (0.3465 ± 0.0039)

(4) Match between injection pump and engine

Lever angle	Pump speed rpm	Total injection qt. cc (cu in.)/500 strokes, 4 cyl.	Remarks
600	81 ± 2 (3.1890 ± 0.0787)	With adaptor spring	
51 = 5	51° ± 5° 1200	62 ± 2 (2,4409 ± 0.0787)	With torque spring

Injection nozzle services

- (1) Needle valve and nozzle body
  - (a) Immerse needle valve and nozzle body in a pool of clean kerosene, insert the valve into the body, and move the valve back and forth to be sure that the sliding contact is smooth without evidencing any excessive clearance. The injection nozzle as a whole must be replaced if the fit is found defective.
  - (b) Visually examine the nozzle body with a magnifying glass having a power of 4 or 5.
  - (c) Inspect the needle valve for distortion or damage at its seating part and for wear of its end face in contact with the pressure pin.
  - (d) Poor seating contact may be corrected, if the defective condition is not advanced too far, by lapping the valve against the seat with a coat of clean lube oil applied to the seating faces. If this does not help, the injection nozzle must be replaced.
- (2) Nozzle holder and distance piece

Check the fit between nozzle holder and distance piece and between distance piece and nozzle holder. Determine the quality of the fit from contact patterns obtained with the use of red lead paste: defective fit will be evidenced by an abnormally high rate of return oil (lead-off) flow.

- (3) Pressure spring and pressure pin
  - (a) Replace any pressure spring broken, cracked or otherwise defective, or out of square. Inspect each spring for these defects.
  - (b) Inspect each pressure pin for wear at its end faces, one for pressure spring and the other for needle valve.
- (4) Leak-off pipe packing

If the packing is found in deteriorated condition, replace it.

Injection nozzle testing and adjustment

(1) Injection pressure

The pressure at which the needle valve unseats itself against the force of the pressure spring is referred to as "valve opening pressure" or "beginning-of-injection pressure," but will be called here "injection pressure" for short. The value of this pressure is specified; it is checked and adjusted as follows:

(a) Install the injection nozzle in the nozzle tester, and operate the manual pumping handle of the tester several strokes to prime the nozzle. (b) Move the lever up and down slowly, completing each up-and-down cycle in about a second, to pressurize the injection nozzle, while observing the indication of the test pressure gauge. As the nozzle begins to spray, the indicating pointer of the gauge being deflected will start perceptively oscillating: read the pressure right then as the injection pressure.

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Item	Standard	Repair limit
Injection pressure	120 ± 5 (1706.4 ± 71.1)	110 (1564.2), minimum

(c) If the reading taken is below the limit, increase the thickness of the shim used on the pressure spring. Increasing the shim thickness by 0.1 mm (0.0039 in.) raises the injection pressure by about 10 kg/cm<sup>3</sup> (142 psi). Adjusting shim stock for this purpose is available in 20 sizes, from 1.0 mm (0.0394 in.) up to 1.95 mm (0.0768 in.) in increments of 0.05 mm (0.0020 in.) each.



Checking injection pressure on nozzle tester

(2) Spray pattern

The injection nozzles used in the present engine are of throttle type. Some throttling action takes place when the needle valve begins to unseat, thereby limiting the amount of fuel being sprayed out during the initial stage of each fuel injection. Thus, each slug of fuel sprayed out may be regarded as consisting of two parts: initial throttled spray and terminating main spray.

When tested on the nozzle tester, the injection nozzle can be made to produce these two kinds of spray for visual inspection. Initial throttled spray comes about when the tester lever is operated at a rate of 60 cycles per minute (up and down in one second); terminating main spray occurs when the lever is operated rapidly at a rate of, say, 4 to 6 cycles per second.



(a) Initial throttled spray

When the nozzle is producing only this spray, atomization is generally poor and the pattern is rather straight than conical, there being more or less after-dribble, that is, fuel dribbling after injection. All these are due to the fact that the fuel being injected is being throttled by the pintle protruding from the valve.

While the nozzle is making this spray, see if the needle valve chatters in synchronism with the cyclic motion of the lever; if so, then the needle valve is free from any sticking or hitching tendency and, if not, the nozzle and needle valve must be cleaned by washing and re-tested.

Off-center spray or directionally erratic spray, if noted, should be taken to mean that the injection nozzle needs thorough cleaning.

(b) Terminating main spray

With the tester lever operated at a rate of 4 to 6 cycles per second, the initial throttle spray is hardly visible. The spray under this condition may be regarded as main spray.

The main spray should be a good straight cone, about  $0^{\circ}$  in angle, consisting of finely atomized fuel particles without any large droplets, and should terminate with no dribble at the tip, not to mention of any fuel dripping.

(3) Seating tightness

An injection nozzle tested and adjusted as above, and found to produce a good spray pattern may be re-used in the engine provided that it passes this final test — seating tightness test.

With the injection nozzle mounted on the nozzle

tester, raise the pressure slowly to 100 or 110 kg/cm<sup>2</sup> (1422 or 1564 psi) (without exceeding the set pressure of 120 kg/cm<sup>2</sup> (1706 psi), so that the needle valve will not unseat). Hold the pressure and observe the nozzle tip: there should be no evidence of fuel oozing out to form a dribble. If such evidence is noted, then the contacting faces of the needle valve and seat must be repaired by lapping in the manner already suggested or the injection nozzle as a whole must be replaced.

## Cooling system

#### Flushing service

Even drinking water contains one or more substances in extremely small quantities as impurities. In the engine in use, the cooling water deteriorates gradually: the concentrations of impurities increase progressively to form sludges, scales or rust on wet walls inside the engine jackets and in the radiator core to interfere with smooth heat conduction.

Overheating tendency of the engine is often due to its cooling water circuits fouled with scale, sludge or rust formation. For this reason, it is necessary to periodically change the cooling water and, at the same time, flush the cooling system clean. For the cooling water, be sure to use a soft water (drinking water is usually soft).

#### Water pump services

- Check to see if the bearing rattles or develops hitches when the pump shaft is spun by hand; if it does, replacement is necessary.
- (2) Inspect the pump impeller for pitting, erosion and breakage; replace the impeller if it is found in bad condition. An impeller found with its front or rear edges showing evidence of rubbing against pump case or rear cover means that the impeller together with the bearing need replacement.



(3) The unit seal must be replaced as a whole if water leakage from it has been complained by the user. Referring to the sectional view of the unit seal, examine floating seat (carbon) (1) and seal ring (ceramic) (2) particularly carefully for wear. If the wear is found excessive, replace the unit seal.

Item	Standard	Wear limit
Free-state height of unit seal	21.8 ± 1 (0.858 ± 0.04)	
Floating seat height	1.5 (0.059)	0

Unit: mm (in.)

Unity mm (in )

(4) Check the fit of pump shaft in the bearing inner race for tightness. If the fit is loose or if the mating faces are badly damaged, replace shaft or bearing or both.

	Quite mun (ma)
Item	Standard
Fit of pump shaft in bearing inner race	0.001T ~ 0.017T (0.00004T ~ 0.00067T) <sup>(front)</sup>
	$0.001T \sim 0.017T$ (0.00004T ~ 0.00067T) <sup>(rear)</sup>

(5) Inspect the bore provided in the water pump case for receiving the bearing outer races to see if the bore is damaged; if so, replace the case or the whole pump assembly. Be sure that the bore admits the bearing races with a tight fit.

onter min p			
Item	Standard		
Fit of bearing outer races in pump case bore	0.011L ~ 0.025T (0.0004L ~ 0.0010T)	(front)	
	0.011L ~ 0.025T (0.0004L ~ 0.0010T)	(rear)	

(6) Inspect the threaded portion of the impeller for damage.

# Thermostat inspection

Test the removed thermostat to see if it starts opening at 76.5°C  $\pm$  2°C (169.7°F  $\pm$  3.6°F) of rising temperature and becomes fully open at 90°C  $\pm$  2°C (194°F  $\pm$  3.6°F). If the difference between these temperature levels is too large, replace the thermostat.

The test is carried out by immersing the thermostat in water contained in an appropriately sized pan and by heating the water with such as an electric stove. Check the rising temperature with a thermometer. Inspection of fan belt and fan

- Inspect the belt for signs of deterioration such as cracks, and check it for permanent stretch. An excessively stretched or cracked belt must be replaced.
- (2) Inspect the fan blades for distortion and cracks and replace the fan as necessary.

# Electrical equipment

#### Starter services

- (1) Inspection before disassembly
  - (a) Checking the starting circuit for operation

With the starter in place, check to be sure that -

- The battery is in good condition, with its cell plates showing no evidence of "sulfation" or any other faulty condition, and is in fully charged state.
- The battery terminal connections are clean and tight.
- 3. The starter terminal connections are tight.
- The wires are securely connected to terminals, and are free of any insulation stripping due to fraying, there being no signs of grounding or breaking.
- The starter switch closes and opens the circuit positively at each position.

Do not jump into a conclusion that the starter is in trouble when the engine refuses to fire up upon cranking: the engine could be in trouble.

(b) No-load test

If the starter is suspected of trouble, take it down from the engine and run a no-load test on it to find out if it is really in trouble.

When removing the starter, be sure to have the battery switch turned off.



No-load test circuit

Here's how to carry out the no-load test: Form a test circuit with a voltmeter and an ammeter, as shown, using a fully charged 24-volt battery; close the switch to run the starter until its speed rises to and above 4500 rpm; and then read the voltmeter and ammeter when the starter is spinning. The ammeter should show that the starter is drawing not more than 50 amperes, with the voltmeter indicating at least 23 volts (at the speed of at least 4500 rpm); if not, estimate the cause of the trouble by consulting the troubleshooting guide, which follows:

Starter troubleshooting guide

Symptom	Possible cause	
Large current and low speed	<ol> <li>Bearings are dirty, or need lubrication.</li> </ol>	
	<ol> <li>Rotor (armature core) is rubbing the pole pieces.</li> </ol>	
	<ol> <li>Grounded coil in the armature or in the field.</li> </ol>	
	<ol> <li>Short-circuit in the armature coils.</li> </ol>	
Large current but no speed	<ol> <li>Magnetic switch is grounded and is not working.</li> </ol>	
	<ol><li>Grounded coil in the armature or in the field.</li></ol>	
	3. Seized bearing.	
No current and no speed	<ol> <li>Open-circuited coil in the armature or in the field.</li> </ol>	
	2. Broken brush pigtail.	
	<ol> <li>No conduction between brushes and commutator because of "high mica" condition or dirty commutator surface.</li> </ol>	
Small current and low speed	Loose coil connection in the field	
Very large cur- rent and very high speed	Short-circuited field coil.	

The best way of testing the starter is to run it under loaded condition, but that requires special testing equipment. For ordinary servicing purposes, the no-load test and troubleshooting guide will do.

- (2) Inspection after disassembly
  - (a) Check the armature coils and commutator for ground, open and short. A circuit tester will serve the purpose of checking the coils and commutator for ground and open. To check for short, however, the "growler" must be used.



Testing armature coils and commutator for ground and open



Testing armature for short with growler

(b) Inspect the commutator surface for burning and "high mica" condition. Surface burns can be removed by turning the commutator in a lathe provided that there is enough stock to be removed by machining without exceeding the limit diameter [43.2 mm (1.701 in.)]. "High mica" must be corrected by undercutting the mica between commutator segments. Inspect the risers and, if their solder is found melted, repair them by re-soldering.

Check the commutator for radial runout and, as necessary, repair it by turning in a lathe.

Unit: mm (in.)

Item	Standard	Repair limit	Limit dia.
Commutator runout	0.03 (0.00118)	0.1 (0.0039)	43.2 (1.701)



Checking commutator for runout

- (c) One end of the shunt field coil is soldered to the yoke. Undo this soldered connection, and check the positive (+) brush holder for ground by putting one testing prod of a circuit tester (with its selector knob set in the ohmic zone) to the "M" terminal of the starter and the other prod to the field coil: the tester should indicate "continuity." Shift the latter prod to the yoke: the tester should indicate "infinity." This check is for finding whether or not the positive brush holder is satisfactorily insulated. If the insulation is found defective, repair or replace the holder.
- (d) Check the brushes for wear. The brush worn down to 13 mm (0.512 in.) in length must be replaced.

		manual course formal
ltem	Standard	Service limit
Brush length	19 (0.748)	13 (0.512)

(e) Replace the rotor if the armature shaft is worn down at any of the three places indicated below; and also replace the bearing if it is so worn that the radial clearance exceeds the upper limit of the range indicated:

-		2.1				
SI	121	π	C	ea	19	nce

I. Los Parz.		20. 1
Unit:	mm	(ID.)

Unit: mm (in )

Item	Nominal diameter	Standard	Repair limit
Commutator	14.2	0.034~0.104	0.2
side	(0.559)	(0.00134~0.00409)	(0.0079)
Pinion side	12.2	$0.034 \sim 0.104$	0.2
	(0.480)	(0.00134 ~ 0.00409)	(0.0079)
Center-	20.3	0.020~0.353	
bracket side	(0.799)	(0.00079~0.00139)	

within I was ambind exceeded

Upon reasonabling the starter, conduct a no-load test in the manner already outlined, and check the magnetic switch and its lever mechanism for piniorshifting action.

The place is an if the minimal angle forward and the electrometer mere such has the same han the test entering arranged as shown below's researchiv The pinion should move back and forth smoothly through its full stroke, without any hitches. If any faulty movement is noted of the pinion, disassemble the shift mechanism and repair it.



Testing circuit for checking pinion shifting action

Having made sure that the pinion moves satisfactorily, close the switch (K<sub>1</sub>) to advance the pinion all the way out and hold the switch closed. Under this condition, move back the pinion by giving a light push to it with a fingertip to take up its play, and measure the clearance between pinion and stopper. This clearance should be between 0.5 and 2 mm (0.0197 and 0.0787 in.); if not, increase or decrease the number of washers used in the magnetic switch mounting.

	Unit: mm (in.)
Item	Standard
Clearance between	0.5~2
pinion and stopper	(0.0197~0.0787)

After mounting the starter in place, test it by cranking the engine a second or so about 10 times just to make sure that the pinion meshes with the ring gear properly.

Generating system services

- (1) Cause of poor charging operation
  - If the user complains that the battery tends to become overcharged or discharged, check the fan belt for tension and, if the belt is in proper tension, check the wiring connections for tightness. Be sure that there is no open in the charging circuit.

possible causes listed in this guide

		Voltage regulator is set to maintain too high a voltage	1
to version services		sousce de norme es suited.	
	G	The batterv is internally short-circuited.	 
		ounces or existing timely.	:
innenna e	з. ,	The alternator is brouking	1

Hator and its didde rectimer must be tested ascentation what is preventing the alternator in-

co, as the internator that on the test bonch and concern use alternator shart to the variable-spee." Show source theorem of the bench. Make electisource to the test circuit arranged as shown:



With switch in crosser (to energize the arternator field from the battery), drive the alternator by turning on the drive motor, increasing the speed of drive gradually, and observe the indications of the voltmeter and ammeter. Stop increasing the speed when the ammeter indication reaches its zero mark, and open switch K1:

Jora the DC output terminal of its rectiment "The putton voltage is now men entering to Alexand Sets a second and all This spore is the one of which the alternaunit develops the rated voltage, and should be about 1100 rpm. Next, close switch K2. With the variable than ..... ance equally eradually and read the ammeter of respective levels of mine maan - ----ülésé readines i ourput current ann speens with the performance specifications to determine wnetner or not the alternator unit is capable of the specified output performance.

the rectifier diodes and breaking the seli-exc-

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ing condition of the alternator uni-

(b) Charling the motifier dioder for unen and more

"OR NEW IN CONTINUE PROCESSION OF THE SECOND (TO/Ward) OF anows.



It is obvious from the above schematic diagram

1999 (1) 199 (1) 199 (1) 199 (1) 199 (1)

A lamp (2 to 10 watts, 24 volts) and a 24-volt battery with two lead wires will serve the purpose.



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Alternatively, a circuit tester may be used. In this case, bear in mind that the (+) terminal of the tester is connected to the (-) side of its built-in battery and the (-) terminal to the (+) side.



Checking diodes with a circuit tester

the foregoing check reveals that the rectifier



Checking diodes for ground

Diodes found short-circuited, open-circuited or ground-circuited by the foregoing methods must be replaced.

(c) Checking the field coils for short through layers Check the ohmic resistance of field circuit by using the circuit tester, with its testing prods put to the two slip rings. The field coils are in sound condition if a resistance reading of approximately 27 ohms [at 20°C (68°F)] is obtained. A lower reading than this calls for rotor replacement.



Checking field coils for layer-short

(d) Checking the armature windings for open and ground

mindings and the core. ....



Checking armature windings for open and short

(e) Each brush may be left in service until it wears down to the limit line marked on it. Be sure to use genuine replacement brushes.



Wear limit mark on brush

(3) Regulator tests and services

With the alternator unit mounted on the test bench and coupled to the variable-speed drive source (motor), connect the regulator unit, as shown below, with two ammeter A<sub>1</sub> and A<sub>2</sub> and a voltmeter. Be sure to use a fully charged 24-V battery. Before starting the test operation, check to be sure that the resistance inside the regulator unit between terminal IG and terminal F is anywhere between about 35 and 41 ohms. This ohmic value is important: this much resistance means that resistor R<sub>4</sub> is in sound condition and that the point faces of contact P<sub>1</sub> are smooth and closing positively to pass field current properly.



Regulator test circuit

(a) Checking for regulated voltage

Close switches K1 and K2 and start driving the alternator. Just when ammeter A1 indicates zero to signify that the alternator is now in selfexciting condition, open switch K2. Raise the speed to 3000 rpm and read the voltmeter. The regulator is properly set and working satisfactorily if this reading is between 28.0 and 31.0 volts.

- (b) If the indication of ammeter A2 remains zero, it means 1) contact P2 is not working properly, or 2) voltage coil VC2 is open-circuited.
- (c) If the voltage read on the voltmeter is outside the range stated above, despite ammeter A<sub>2</sub> showing some field current being supplied to the field, then it means 1) voltage coil VC<sub>1</sub> is open-circuited, 2) resistor R<sub>3</sub> is open-circuited, or 3) contact P<sub>1</sub> is stuck closed with its point faces fused together.

# Bench tests

An overhauled engine should be operated on the test bench in order to correctly break it in, to adjust the engine and injection pump and governor for best engine performance, and to quantitatively determine the output power the overhauled engine is capable of. The test bench is an apparatus complete with a dynamometer. In the following procedures, a standard-type test bench is assumed.

# (1) Breaking in

Have the engine set up on the bench, and aligned to the dynamometer. Operate the engine for a total of 2 hours according to the schedule indicated below. During this operation, observe the running condition carefully and, if any malcondition is noted to be developing, shut down the engine and take steps to correct it.

-	
U	30
7.5	30
15	60
20	60
	0 7.5 15 20

Breaking-in	schedule
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Run the engine with (a) lube oil pressure held between 3 and 4 kg/cm<sup>2</sup> (42.7 and 56.9 psi), (b) cooling water temperature held between 75° and  $85^{\circ}C$  (167° and 185°F), (c) lube oil temperature, as measured in the oil pan complete with a vacuum pump, held between 80° and 90°C (176° and 194°F). While the engine is running, check to be sure that there is no leakage of oil, water or combustion gases, and listen into the engine now and then for abnormal noise.

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For this breaking-in run, start up the engine as follows:

- (a) Fill up the cooling system and the oil pan, and make sure there is enough fuel. Prime the fuel system, letting out all trapped air, if any.
- (b) Use the preheating system to make sure that the glow pilot lamp works. The lamp should glow in about 20 seconds.
- (c) With the adjusting lever of the governor moved to the starting position, crank the engine with the starter and, after starting up the engine, move the adjusting lever to its idling position.
- (d) With the engine kept in idling condition, inspect for leakage of oil or water, observe the color of exhaust smoke, listen into the engine for abnormal noise, and check lube oil pressure and cooling water temperature.

Until the cooling water temperature rises to its normal operating range, the engine might develop sharp knocking-like sound, but this is no cause for alarm because it will disappear as the temperature rises. (2) Engine performance tests

1

Have the air cleaner, vacuum pump, alternator unit and other auxiliary devices mounted on the engine, and test the engine for (a) no-load maximum speed (governor setting), (b) fuel injection (rack setting), and (c) no-load minimum speed (idling setting). Have the dynamometer turned off for these three tests.

 (a) Setting the governor (no-load maximum speed test)

Immediately after the breaking-in operation, set the governor to limit the highest speed to  $2640 \pm 20$  rpm. With the governor so set, the engine speed will, as it should, fall to 2400 rpm when full load (rated load) is put on the engine by means of the dynamometer.

- (b) Setting the rack (injection quantity test) Set the rack so that fuel injection quantity will be between 7.6 and 7.8 liters (464 and 476 cu in.) per hour at 1600 engine rpm.
- (c) Setting the idling stop screw (no-load minimum speed test)

Set the stop screw so that, when the adjusting lever is turned to bear against this screw, the governor will allow the engine to run at 600 rpm.

(3) Engine output test

Turn on the dynamometer to impose load on the engine running at no-load maximum speed and increase the load until the speed falls to 2400 rpm. Read the dynamometer indication right then. Determine the formal output power by multiplying the reading by this correction factor K:

$$K = \frac{760}{H - Hw} \sqrt{\frac{273 + t}{293}}$$

where H : barometric pressure in mmHg

Hw : partial pressure of H2 O vapor in mmHg

t : room temperature in °C.

# DISASSEMBLY AND REASSEMBLY

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Hints for facilitating disassembly-reassembly work Engine disassembly

1

- (a) Orderliness is important. Have work benches and parts trays in good condition, clean and tidy. Washing sinks and pans should be neat and ready for use. Have the disassembled parts placed in respective trays, keeping a group of associated parts in the same tray for easy identification.
- (b) Before separating two parts, be sure to make match marks as necessary. Even for those parts to which positional matching is not critical, such marks will facilitate reassembly work.
- (c) Signs of some defects or flaws are visible during disassembly but may disappear when the disassembled parts are washed clean. Leave your findings on record when such signs are noted.
- (d) Use the right kind of tool for each disassembling job, in order to protect the parts and to speed up the work.
- (e) Handle bearings and bushings and the like with care. They are critical parts: a little nick could make them unfit for re-use.

Engine reassembly

- (a) Make it a rule not to re-use dirty parts in reassembly. Oil seals and bearings must be particularly clean. Before installing them, be sure to clean the bores for admitting them.
- (b) It is a good practice not to re-use those gaskets and sealing members removed from the engine that has been in long service. Use of replacement parts is more economical as far as gaskets and the like are concerned.
- (c) Before fitting a running part, be sure to oil its sliding surfaces. Use clean, fresh engine oil.
- (d) Have the specified sealing compound on hand. Use of the compound is prescribed for most of sealing parts.
- (e) Torque limits are specified for some bolts and nuts. Be sure to use torque wrenches and to refer to the specified values of torque limits.

Engine dismounting and mounting

The engine and transmission are to be taken down together from the machine. The two must be combined on the bench and then remounted as a unit. The dismounting and mounting procedures are set forth in another manual.

Rocker arms and rocker shaft

Rocker shaft assembly removal

- (1) Remove rocker cover and gasket. Remove oil pipe eye bolt (1) and disconnect pipe (2).
- (2) Loosen short bolt (3) first and long bolt (4) next on each bracket and, after freeing all four brackets, lift the rocker shaft assembly (5) off the cylinder head.



(3) Draw out push rods (6), and remove valve caps (7).



6-Push rod 7-Valve cap

Rocker shaft assembly installation

- (1) Insert push rods into respective tappet holes. Mount valve caps on the heads of respective valves. Position the rocker shaft assembly on the cylinder head and make the short bolts and long bolts fingertight.
- (2) Tighten the 8 short and long bolts to a torque value between 1.5 and 2 kg-m (10.8 and 14.5 ft-lb). Give the final torquing to long bolts first and to short bolts next, making sure that all bolts are tightened equally.

(3) Reconnect the oil pipe and secure the connection by tightening the eye bolt.

Upon starting the reassembled engine for the first time, inspect the rocker shaft assembly, checking for evidence of any loose bolts on the brackets and making sure that the rocker arm mechanism is properly lubricated. Check, also, for abnormal noise, such as valve chatter due to improper valve clearances.

As to the valve clearance, refer to the part dealing with valve clearance adjustment in the latter part of this section. For both exhaust and intake valves, this clearance is prescribed to be 0.25 mm (0.0098 in.) (cold).

Rocker shaft disassembly and reassembly



Rocker shaft assembly - Exploded view



Rocker shaft assembly

- To break the rocker shaft assembly, taken off the cylinder head, into its component parts, the first step is to remove snap rings (1), one at each end of the shaft. Pliers must be used to pick out the rings.
- (2) Remove washer (4). This permits the assembly to be broken into components: brackets (10), rocker arms (5), springs (15) and shaft (16).



Removing snap ring



Reverse the above sequence of disassembly to rebuild this assembly. Make sure that each rocker arm in place is capable of smooth rocking motion.

# Cylinder head

#### Removal

 Disconnect bypass hose (1) and pipe (2). To do so, the thermostat elbow and water pump clamp must be displaced.



1-Bypass hose 2-Pipe Removing cylinder head (1)

- (2) Remove bracket (3) and fuel filter (4).
- (3) Disconnect injection pipes (5), leak-off pipe (6) and return pipe (7).



3-Bracket 4-Fuel filter 5-Injection pipe

6-Leak-off pipe 7-Return pipe

Removing cylinder head (2)

- (4) Remove rocker cover, and take out the rocker shaft assembly, push rods and valve caps, as described previously.
- (5) Remove cylinder head bolts (8). Lift cylinder head (9) straight up to remove it from the block of cylinders and crankcase.



8-Cylinder head bolt 9-Cylinder head Removing cylinder head (3)



Cover up the open ends of injection pipes and air intake pipe to avoid entry of dirt. When removing the gasket from cylinder head, be careful not to nick or mar the gasketed surfaces of head and block. Read the torque needed to loosen each cylinder head bolt: these readings might help locating the cause of the trouble reported.

## Installation

 Place new gasket (1) on the block (2), and lower cylinder head (3) squarely onto the gasket. Be sure to have two guide bolts (4) installed so that the gasket in place will not shift.



1-Gasket 3-Cylinder head 2-Crankcese 4-Guide bolt (2 pcs) Installing cylinder head (1)

(2) Using a torque wrench, tighten the cylinder head bolts to 12 ± 0.5 kg-m (87 ± 3.6 ft-lb) in the sequence indicated by the ascending order of numbers, starting with "1."





Installing cylinder head (2)

- (3) Insert push rods, fit valve caps and install the rocker shaft assembly as described previously.
- (4) Reconnect injection pipes, bypass hose and others.
- (5) Adjust the valve clearance as prescribed, and mount rocker cover.

#### Disassembly

(1) Remove nozzle holders (1) and glow plugs (2).



1-Nozzle holder 2-Glow plug Disassembling cylinder head (1)

- (2) Remove nuts (3) securing exhaust manifold (4) to the block. Take off manifold (4).
- (3) Similarly remove intake manifold (5).



3-Nut 5-Intake manifold 4-Exhaust manifold Disassembling cylinder head (2)

(4) Remove thermostat elbow (6) and take out thermostat (7).



6-Thermostat elbow 7-Thermostat Disassembling cylinder head (3)

(5) Remove exhaust and intake valves as follows: Compress valve spring by operating the valve lifter, pick out valve cotters (8) and retainer (9), release spring (10) and remove spring and valve (11).



Removing valve cotters



(6) After disassembling the head, de-carbon it thoroughly and clean the threaded holes for glow plugs and nozzle holders by washing. Clean the plugs and holders similarly. Use compressed air to dry washed parts. When washing the nozzle holder units, be careful not to damage their nozzle tips.

# NOTES

1

- a) Intake and exhaust valves are not marked to identify the respective cylinders they serve. Upon removing each valve, be sure to mark it or otherwise identify it to ensure that it will be restored to the original place of service.
- b) The two halves of each valve cotter must be handled as a matched pair for the valve from which it was removed in disassembly.
- c) Leave the valve guides in place unless they need replacement.
- d) Observe the carboned condition of the combustion chamber surfaces, intake and exhaust ports and valve heads before cleaning them. What is observed is an important symptom for troubleshooting.
- e) To remove the valve guide, be sure to use the valve guide remover (A).



Removing valve guide

f) Leave the precombustion chamber jets in place unless their replacement is necessary. To remove a jet, as when cracks are noted on it, ease the jet out by driving with a flat-faced drift pin inserted through the glow plug hole, as shown:



Removing precombustion chamber jet

Before installing the jet, wash the precombustion chamber cavity clean, and drive the jet into position, with its orifice pointing to the center of the cylinder.





Make sure that all component parts have been serviced or otherwise checked to be in re-usable condition; have them all clean, free of greasy matter. Refer to this cross section in executing the reassembly work:





Cylinder head - Cross section

1-Lock nut 2-Adjusting screw 3-Push rod 4-Cylinder head bolt 5-Rocker bracket 6-Valve guide 7-Exhaust valve 8-Valve seat 9-Cover 10-Rocker arm 11-Rocker shaft. 12-Spring retainer 13-Valve cap 14-Valve cotters 15-Spring 16-Stem seal 17-Glow plug 18-Leak-off pipe 19-Nozzle 20-Cylinder head 21-Precombustion chamber jet

 Assuming that the valve guides have been removed, install each guide (2), as shown, with the use of the guide installer (A). After driving the guide in, check to be sure that the dimension (B) measures 17 mm (0.669 in.).



1-Cylinder head 2-Valve guide

A-Valve guide installor B-Specified length: 17 mm (0.669 in.) Installing valve guide



- (2) Mount valves (7), springs (15) and retainers (12). Install valve cotters (14) on each valve with the spring compressed by means of the valve lifter. Leave out caps (13), which are to be put on when installing the rocker shaft assembly.
- (3) Restore to the cylinder head the thermostat, thermostat cover, nozzle holders, leak-off pipes, glow plugs, connection wires, exhaust manifold and intake manifold.

# Timing gears

Removal

1

 Remove bolts (1) and take down the fan (2). Loosen bolt (3) securing the adjusting plate to the alternator, and take off fan belt (4).



(2) Remove nut (5) on water pump shaft, and nut (6) on crankshaft. From the pump shaft, remove fan spacer and, by using the fan puller (A), draw pump pulley (7). Draw pulley (8) from crankshaft with the puller.



5-Nut and washer 7-Pump pulley 6-Nut and washer 8-Crankshaft pulley Removing pump pulley and crankshaft pulley



7-Pump pulley A-Puller Drawing pump pulley off



8-Crankshaft pulley A-Puller Removing crankshaft pulley

(3) Remove bolts (9) securing the cover (10). Remove bolts (10) securing the timing gear case (12). Take off the cover and gear case.



9-Bolt and washer (5 pcs each) 11-Cover 10-Bolt and washer (9 pcs each) 12-Gear case Removing timing gear case

(4) Remove idler bolt (13), thrust plate (14) and injection pump gear nut (15).



13-Bolt 14-Thrust plate 15-Nut and washer 16-Idler gear 17-Camshaft gear 18-Injection pump gear 19-Crankshaft gear

Timing gears

(5) Draw idler gear (16) while twisting it in the direction of its helix.



16-Idler gear Removing idler gear

Leave camshaft gear (17) and injection pump gear (18) in place, unless they have to be serviced or replaced: each gear is to be removed complete with its shaft.



Before removing any of the timing gears, be sure to turn over the crankshaft to bring these gears into the position at which the timing marks provided on them meet each other. Removed gears, if any, are to be fitted to take the same angular position, which is illustrated below:



# Installation

The procedure is generally the reverse of removal. It is assumed here that injection pump gear (4), camshaft gear (1) and idler gear (2) have been removed.

- Mount pump gear (4) and camshaft gear (1). Turn these gears while fitting idler gear (2), so that the match marks will meet as shown above.
- (2) Position timing gear case (1) in place, as governed



1-Timing gear case 3-Bolt and washer 2-Cover (13 pcs each) Securing gear case

by the locating pins provided on the front plate, and secure the case by tightening a total of 13 volts. Install injection pump gear cover (2).

(3) Install water pump pulley (5) and crank pulley (6) by driving them onto respective shafts with installers (B) (C).



5-Water pump pulley B-Installer Installing pump pulley by driving



5-Crank pulley C-Installer Installing crank pulley by driving

- (4) Fit the washer to crank pulley and fasten down the pulley by tightening the nut.
- (5) Pass the fan belt around the pulleys and install the cooling fan. Adjust the belt for proper tension as described in the part titled "Fan belt tension adjustment." Belt tension is specified in terms of "belt deflection" and is prescribed to be 12 mm (0.472 in.) (deflection) under thumb pressure.

# Lubrication system

## Oil filter removal and installation

The oil filter is bolted to the crankcase. Removing the four bolts (1) allows the filter (2) to be detached from the engine for removal. When installing the filter, make sure that the seating face is clean and that the packing is in good condition.



1-Bolt and washer (4 pcs each) 2-Oil filter assembly Removing oil filter

#### Oil filter disassembly and reassembly

To disassemble the oil filter, use the oil filter wrench to detach element (1) from filter bracket (3). Removing relief valve (4) from the bracket completes disassembly. To reassemble the oil filter, reverse the disassembling sequence.





#### Oil pump removal

To remove the oil pump, the oil pan must be separated from the crankcase. Before removing the oil pan, be sure to drain it completely. Removing the mounting bolt (1) allows the pump (2) to be pulled out of the crankcase. Reverse this sequence of removal to install the oil pump.



Removing oil pump

# NOTE

The oil pump will not come off if oil pump drive gear is firmly meshed with camshaft skew gear. While giving a pull to the oil pump, turn over the crankshaft a little to ease the drive gear from the skew gear.



When installing the oil pump, examine the gasket for the mounting bolt. Replace the gasket if it is nicked or otherwise defective.

#### Oil pump disassembly

 From the pump, remove oil strainer (2) and pump cover securing bolts (1). Separate the cover.



1-Bolt and washer (4 pcs each) 2-Oil strainer Removing oil strainer

(2) Invert the pump case and catch outer rotor (3), which will slide out of the bore by its own weight.



3-Outer rotor Removing outer rotor

(3) Drive out tapered pin (4) by using a drift, as shown, and pull drive gear (5) off main shaft (6). Draw the main shaft out of the pump case.



Removing tapered pin

(4) Drive out inner rotor pin (7), and separate main shaft (6) from inner rotor (8).



7-Inner rotor pin Removing inner rotor pin

Oil pump reassembly

After securing inner rotor to shaft by driving in the pin, insert the shaft into the pump case, and mount the gear on the shaft, locking the gear by driving in the tapered pin.





Mounting drive gear



Fitting cover to case by matching marks

# NOTES

- a) If main shaft or drive gear has been replaced, a new pin hole must be made by drilling through the gear mounted on the shaft.
- b) After putting on the cover, check to be sure that the match marks are correctly indexed. If the cover is in a wrong position relative to the case, the pump will not draw in oil. Tighten the bolts after checking to be sure that the marks are correctly matched.
- c) After reassembling the pump complete with its strainer, immerse the strainer in a pool of oil and run the drive gear by hand to make sure that the pump is capable of sucking oil in.

# Cooling system

(water pump complete with thermostat)

Water pump removal

(1) Remove four bolts (1) and take off fan (2). Loosen bolt (3) and remove fan belt (4).



Removing fan

- (2) Loosen clamp (5) and disconnect bypass hose (6).
- (3) The oil pipe for pressure-feeding lube oil to the water pump is connected to the pump by means of union nut (7). Loosen this nut and disconnect the pipe.



(4) Remove the nut securing pulley (8), and draw the pulley off the shaft by operating puller (A).

Disconnecting hose and pipe

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8-Pulley A-Puller Removing pulley

(5) Remove mounting bolts (9) and take off the water pump assembly from the crankcase.



9-Bolt and washer (4 pcs each) 10-Pump essembly Removing water pump

(6) Remove two bolts (11) securing elbow (12), and take off thermostat (13), as outlined previously.



Installation is reverse of removal and can be effected by carrying out the foregoing steps in the reverse order. After installing the water pump, be sure to adjust the belt for proper tension.



Handle the fan belt with care, keeping it free of any greasy stains. After removing the water pump from the crankcase, be sure to close the water opening of crankcase to avoid entry of dirt,

Water pump disassembly





1-Bolt and washer 2-Cover Removing pump cover

(2) Hold the pump shaft rigidly, and unscrew pump impeller (3) to remove it from the shaft.



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Impeller (3) is mounted threadedly on the shaft. The screw threads is of right-hand screw. To remove impeller, turn it in the direction of the arrow.

#### Removing impeller

(3) Ease out oil seal (5) from pump case (10).



 Remove cover (2), which is secured to the pump case by bolts (1).

(4) Using pliers, pick out snap ring (6) from around the shaft.



(5) Draw shaft (8) out from the pulley side of the pump case. Separate the two bearings (7) (9) from the shaft.



7-Bearing 8-Shaft 9-Bearing Removing shaft

Water pump reassembly



- (1) Fit oil seal (10) into the case (4).
- (2) Mount bearings (2) on shaft (1), put on spacer (3) and insert the shaft into the case.
- (3) To the pulley side of the pump case, fit snap ring (5).
- (4) Attach unit seal (7) to impeller (8), and mount the impeller on the shaft by running the impeller onto the shaft.
- (5) Put on cover (9) and, after fastening it down to the case, check to be sure that the impeller does not rub the cover.

# Flywheel and ring gear

Ring gear separation from flywheel

It is assumed here that the clutch has been removed.

(1) Straighten lock washers (1) and remove bolts (2).



1-Lock washer (2 pcs) 2-Bolt (4 pcs) Removing flywheel (1)

(2) Hold the flywheel with both hands, as shown, and pull it off crankshaft. Lay the removed flywheel on the bench top.



Removing flywheel (2)

(3) Using a drift, drive pilot bushing (3) out.


3-Pilot bushing Removing pilot bushing

(4) Immerse the flywheel in a hot bath of oil at 100°C (212°F). Keeping the flywheel in this bath for 3 minutes will heat the flywheel uniformly to this temperature. Take out the flywheel, place it on a firm, level working surface, and ease ring gear (4) off flywheel (5) by driving with a drift bar.



Flywheel reassembly and mounting

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- Clean the shouldered periphery of the flywheel for receiving the ring gear. Clean the ring gear similarly.
- (2) Heat the ring gear to about 100°C (212°F) in a hot bath of oil. Keeping the gear in the 100°C (212°F) bath will heat it uniformly to this temperature. Upon removing the ring gear from the oil bath, fit it to the flywheel, making sure that the ring is seated firmly.
- (3) Drive the pilot bushing into the flywheel.



Installing pilot bushing

(4) Screw two guide bolts (A) into the crankshaft end. Hold out the flywheel squarely against crankshaft, positioning it as guided by dowel pins.



A-Guide bolt (2 pcs) Remove guide bolts after correctly positioning the flywheel. Fitting flywheel to crankshaft

(5) Put on washers and tighten mounting bolts to secure flywheel to crankshaft. Be sure to torque these bolts to 8.5±0.5 kg-m (61.5±36 ft-lb). Lock the tightened bolts by bending the washers firmly against each bolt head,



Securing flywheel to crankshaft.

## Fuel filter

Removal and installation

- (1) Close the fuel supply valve under the fuel tank.
- (2) Disconnect fuel feed pipe (2) from filter.
- (3) Remove two mounting bolts and take off the fuel filter assembly (3).



Removing fuel filter

Reverse the foregoing sequence of steps to install the fuel filter. After installing the fuel filter, run the engine and inspect for fuel leakage from pipe connections.

## Disassembly and reassembly

The two views of the fuel filter given here are selfexplanatory and will serve to explain the methods of disassembly and reassembly:





Parts are to be taken off one another in the ascending order of reference numbers.

1-Bolt 5-Spring seat 2-Filter cover 6-Spring 7-Filter case 3-Gasket 4-Element Fuel filter - Exploded view

## Injection nozzles

Removal and installation

- (1) Loosen connector (1) and disconnect injection pipe (2) from the nozzle holder.
- (2) Loosen the union nut and disconnect return pipe (3) from the holder.
- (3) Remove nut (5) on each nozzle holder and disconnect leak-off pipes (4) interconnecting the holders.



1-Connector (4 pcs) 2-Injection pipe 3-Return pipe 4-Leak-off pipe 5-Nut (5 pcs)

Removing injection nozzle

(4) Put the wrench to the retaining nut, and unscrew the injection nozzle assembly to remove it from the cylinder head. Take off the packing remaining behind on the seating face by plucking with a screwdriver tip. Examine the removed packing to see if it can be re-used.

## NOTE

Reverse the foregoing sequence of removing steps to install the injection nozzles, making sure to tighten the nozzle holder in place by torquing its nut to  $5 \pm 0.5$  kg-m ( $36.2 \pm 3.6$  ft-lb). After removing each nozzle assembly, be sure to plug up the hole with a wad of cloth to avoid entry of dirt into the cylinder.

### Disassembly

- Before disassembly, collect data on the nozzle by testing it for injection pressure (beginning of injection), spray pattern and internal oil-tightness, all in the manner already described. Throughout the disassembly, cleaning and reassembly work, handle each nozzle assembly with care to protect, in particular, the nozzle tip.
- (2) Clamp retaining nut (1) between the jaws of a vise, as shown, put the wrench to the holder and loosen it to separate it from the nut (1).
- (3) Take out of the removed holder these parts: nozzle tip (3), distance piece (4), pressure pin (5), spring (6) and washer (7).

Wash the disassembled parts clean with clean kerosine or diesel fuel oil, and dry them with compressed air. Using a wooden scraper, remove carbon: after decarboning, wash the decarboned parts with a more powerful cleaning fluid such as gasoline.



Removing nozzle holder



Injection nozzle assembly - Exploded view

### Reassembly

The reassembling steps are the same as the disassembling steps except that the sequence is reversed, and that the job of fitting a part to another must be carried out in a pool of clean kerosine.

If the needle valve and nozzle proper have to be replaced, be sure to wash the replacement parts in the pool of kerosine after removing their protective film of plastic: wash off the rust-preventive oil from the nozzle proper by stroking the needle valve back and forth in the needle valve stem bore.



## Injection pump and governor

## CAUTION

Unless the circumstances require disassembly of the injection pump and its governor, these components should not be disassembled. To overhaul them, special equipment complete with testing devices and special tool is needed. Furthermore, the overhauling work must be performed by a person skilled in this service and in a place specially kept clean.

### Removal

 Disconnect control wire (2) from adjusting lever rod (1); and disconnect fuel feed pipe (4) from the pipe joint at injection pump. Similarly disconnect the pipe between fuel filter and injection pump. Remove fuel return pipe (3) by undoing its pipe connections.

At each injection pipe connection on the pump unit, undo connector (5) to disconnect injection pipe (6). Remove oil pipe (7).



1-Adjusting lever rod 2-Control wire 4-Fuel feed pipe

5-Connector 6-Injection pipe (4 pcs)

Removing injection pump (1)



3-Return pipe 7-Oil pipe Removing injection pump (2)

(2) Remove the injection pump gear cover. Turn over engine crankshaft to bring the piston in No. 1 cylinder to top dead center on compression stroke. Look into the timing gear case to check to be sure that the match marks provided on the idler and pump gear are meeting each other correctly.



8-Pump gear Injection pump gear

(3) Remove bolts (9) securing the mounting flange of the injection pump to the engine front plate, and take down the injection pump unit.



9-Bolt and washer Removing injection pump (3)

Installation

- (1) Alignment marks (line marks) are provided on the pump body and mounting flange. Make sure that these marks are lined up. With the pump gear and idler properly positioned in their meshed condition inside the timing gear case, that is, the match marks on these gears indexed to each other, mount the injection pump unit on the engine front plate and secure it by tightening the mounting bolts.
- (2) Install fuel feed pipes and lube oil pipe, and reconnect all but No. 1 fuel injection pipe.
- (3) Turn over engine crankshaft slowly until the plunger in No. 1 pumping element comes to the position for "beginning of injection." Check to be sure that the timing mark on crank pulley is matched to the mark on the timing gear case; if not, adjust the mounted position of the pump in the following manner:



Setting injection timing

- (4) Tilting the pump toward the engine advances the timing, and vice versa. Refer to the graduation marks provided on the edge face of the mounting flange: one division is equivalent to 6 deg. of crank angle.
- (5) Having made sure that all timing marks are matched as prescribed and that the beginning of injection is correctly timed (in reference to No. 1 cylinder), reconnect the injection pipe (No. 1). Prime the fuel circuit in the manner previously described: make sure that no air remains trapped in any part of the circuit.



Whether the injection pump is correctly installed must be checked by actually running the engine. Run the engine in all speed ranges; listen in for abnormal noise and examine the color of exhaust smoke. Evidence of malconditions noted could be due to mistimed fuel injection.

Feed pump removal and installation

- (1) Disconnect fuel feed pipe (1) and return pipe (2).
- (2) Remove nuts (3) securing the feed pump to the injection pump body, and take off feed pump assembly (4).



1-Fuel feed pipe 3-Nut and washer (3 pcs each) 2-Return pipe 4-Feed pump assembly Removing feed pump

Feed pump disassembly and reassembly



 Remove priming pump (2), and take out check valve (3).



1-Valve holder 3-Check valve 2-Priming pump Removing priming pump NOTE

Priming pump is not meant for disassembly: its cylinder and valve holder are integrally combined by using a bonding compound.

(2) Loosen piston chamber plug (4), pick out piston spring (6), and draw out piston (5).





4-Piston chamber plug 6-Piston spring 5-Piston Removing feed pump piston

(3) Remove ring (7), and take out tappet (8) and push rod.



7-Spring 8-Tappet Removing feed pump tappet



Assembly is reverse of disassembly. Be sure to correctly assemble by referring to the cross section.



#### Governor disassembly

The following tools are needed to disassemble and assemble the RSV governor of the injection pump unit:

- (a) Screwdriver
- (b) Wrench set
- (c) Long-nose pliers
- (d) Special wrench for torquing governor weight round nut
- (e) Flyweight extractor
- (f) Overhauling tool set

Before starting to disassemble the governor, wash the exterior surfaces of injection pump unit and set up the pump on the bench. Drain lube oil.

- (1) Detach and remove the governor cover as follows:
  - (a) Loosen idle lock nut (1), remove idling subspring (2), loosen screws securing cover piece (3), and take off this cover piece.
  - (b) Remove the six screws (4) securing the governor

cover to the housing, and detach the cover by pulling it a little. Insert the screwdriver and move the shackle upward or downward with the tip of screwdriver to undo the pinned connection between control rack and shackle.



Disconnecting shackle from control rack

(c) Using the long-nose pliers, unhook the start spring. Remove the governor cover complete with the lever mechanism.



Disconnecting start spring

- (2) Remove the flyweights as follows:
  - (a) Remove round nut (5) by loosening it with the round nut wrench.



Removing round nut



Governor - Exploded view

(b) Run the flyweight extractor into the threaded hole, as shown, and force the flyweights off the camshaft by jacking action.



Removing flyweights

- (3) From the removed governor cover, take out the lever mechanism parts, as follows:
  - (a) Remove two screw plugs (2), and draw out lever supporting shaft (8), on which tension lever (7) is hinged.
  - (b) Raise the swiveling lever, as shown, take out tension lever (7) and remove control spring.
  - (c) Take out guide lever (9) together with control lever (10) and governor sleeve (11).
  - (d) To remove swiveling lever (12), remove adjusting lever (13), pick out snap ring (14), drive out lever bushing (15) to outer side. This permits the lever (12) to come out of the cover.



Removing tension lever

#### Governor reassembly

Carry out the sequential steps of disassembly in reverse order, adhering to the following instructions:

(1) Flyweight mounting

After positioning the flyweights on camshaft, secure it by tightening the round nut to 6 kg-m (43.4 ft-lb). Be sure to place a spring washer under this nut.

(2) Combining governor sleeve and guide lever (floating lever)

If these two parts have been separated, combine them in the following manner:

Referring to the sketch below, press the ball bearing into the governor sleeve, and press the control block into the bearing, making the flange of the block seat firmly against the inner ring of the bearing. Fit the sleeve (complete with the control block) onto the flyweight support, and measure the distance between the end face of governor housing and the flange of control block. This distance is prescribed to be from 15 mm (0.591 in.) to 15.2 mm (0.598 in.); if not, adjust it by shimming. The shim stock for this purpose is available in three sizes: 0.2 mm (0.0078 in.), 0.3 mm (0.0118 in.) and 0.4 mm (0.0157 in.).



1-Governor sleeve 3-Governor housing 2-Control block

Position of control block



Pressing ball bearing into governor sleeve



1-Control block 2-Inner ring supporting tool Pressing control block into ball bearing (3) Check to be sure that the adjusting screw in the swiveling lever is positioned as prescribed; if the screw is too far in or out, adjust it by referring to the value indicated in the list of adjusting standards so that the governor adjusting work to be carried out after reassembly will be made easier.



Checking position of adjusting screw in swiveling lever

(4) Make sure that each lever and link set in position moves smoothly without unduly heavy resistance. After securing the governor cover to the housing, check to be sure that tensioning and relaxing the start spring cause the control rack to slide outward and inward smoothly.

Injection pump disassembly, inspection and reassembly

- (1) General rules on work
  - a. Make a batch of washing fluid available for ready use. For the washing fluid, use clean kerosine or diesel fuel oil.
  - b. Work in a clean place. The injection pump is a precision-machine component and, as such, abhors dust. After removing it from the engine, wash its exterior clean and inspect for damage: this should be accomplished before starting to disassemble the pump.
  - c. Some jobs are prescribed to be effected with the use of special tools. Use of common tools is not permitted.
  - d. Handle each plunger and its cylinder (barrel) as a suit, and each delivery valve with its seat as another suit. Upon removal of these parts, have them set aside as distinct suits identified for the cylinder numbers.
  - e. When installing the delivery valve holder, be careful not to overtighten it or the pump housing will break. A sticky control rack is often due to an overtightened valve holder. Use a torque wrench, and tighten it to this torque limit:

Unit: kg-m (ft-lb)

ltem	Limit
Delivery valve holder	2.5 ~ 3.5
tightening torque	(18 ~ 25)

f. Each plunger has its part number punched on its flange. When reassembling the injection pump, be sure to position each plunger so that its punched part faces the front; this means that, with the plunger so positioned, its control groove meets the feed hole. Remember, the adjustability of injection quantity presupposes that all plungers are so positioned in their barrels.



1-Feed hole 2-Control groove Pumping element parts

- g. It is highly essential that the control rack should slide smoothly in place. If any stickiness is noted after reassembly, the pump must be disassembled and reassembled once again. Stickiness of the control rack is often caused by nicks or dents sufferred by the rack, defective rack teeth or pinion teeth, interference between pinion and pump housing, or overtightened delivey valve holders.
- h. Axial play is specified for the camshaft. Make sure that it is between 0.03 and 0.05 mm (0.0012 and 0.0020 in.). If too much or too little a play is noted, adjust it by shimming.
- (2) Disassembling procedure
  - (a) Drain lube oil. Examine the drained oil for evidence of malcondition. Set the injection pump firmly on the work bench, by means of the holding fixture.

Remove the cover plate and look in to see if there are signs of malcondition.

## NOTE

Removing one part after another without examining each part critically and heeding to the story each part wants to tell is a wasteful practice and prevents you from taking proper measures necessary for restoring the pump as close to the original condition as possible.



Setting up pump by means of holding fixture

- (b) Remove the governor from the pump proper, as in Governor disassembly.
- (c) Turn the pump camshaft by hand to "feel" the resistance of camshaft: abnormal resistance means that there is something wrong in the pump. Using a spring balance, measure the sliding resistance of the control rack, and write down the reading for reference. A limit is specified on this resistance:

Unit: gram (ounce)

ltem	Limit
Sliding resistance of control rack in standstill pump	150 (5.3)

## NOTES

- a) A control rack not sliding smoothly could mean that tappets or control sleeves are in bad condition. Check these parts carefully when taking them out in disassembly.
- b) Move the control rack all the way in each direction, and note the position of each control pinion for the extreme positions of the rack. Restore these positions at the time of reassembly.
- (d) Isolate the four tappets from the cams by holding them in lifted condition. This is accomplished by turning the camshaft by hand to raise each tappet to the highest position and locking the tappet in that position with a tappet insert.



Locking tappets

(e) Lay down the pump and remove the screw plugs, as shown:



Removing screw plugs

(f) Remove the bearing cover, and draw out the camshaft gently.



When removing the bearing cover, be careful not to nick or mar the seating faces or oil leakage will develop during operation.



Removing camshaft

(g) Remove the tappet inserts locking the tappets in raised position. To do so, the roller clamp (95905-06030) must be used: insert the clamp through the screw plug hole to pinch the tappet roller, and give a push to the tappet with the clamp to allow the tappet insert to be pulled out.



Taking out tappet inserts

(h) Insert the tappet clamp (95905-02030) through bearing hole to pinch the tappet, take off the roller clamp, and remove the tappet. Remove all four tappets in this manner.



Tilt down the pump so that the plungers and springs will not fall off.



Removing tappets

(i) Remove the plungers and lower spring seats by using the plunger clamp (95905-09030): pinch the lower portion of the plunger with this tool, and draw out the plunger together with its lower spring seat.



This removal operation must be carried out with great caution to avoid scratching the plunger: pull the plunger straight out.



Removing plungers

(j) Draw out each plunger spring, and remove the control sleeve and upper spring seat.



Removing plunger springs and upper seats

(k) Raise the pump body into vertical position. Unscrew and remove the delivery valve holders. Install the extractor and draw out each delivery valve, as shown:



Removing delivery value

# CAUTION

When installing the extractor, be careful not to run down the threaded portion of the extractor too far onto the delivery valve in place. I.

 Take out the cylinder (barrel) from top side by pushing on its bottom.



Removing cylinder

(m) Remove the rack guide screw, and draw out the control rack.



Removing control rack

(3) Inspection

Lay out the removed parts in the trays, segregating them in groups, each group for each pumping element and related parts, as identified for the respective reference numbers, No. 1, No. 2, No. 3 and No. 4. Do not disturb the original suits.

Wash each part clean, and dry it with compressed air, making sure that orifices, screw threads and pockets are all clean.

(a) Pump housing

Inspect each bore, from which the cylinder (barrel) has been drawn out, to see if there is any damage. Check to be sure that the counterbore (into which the shoulder of the cylinder

fits) is in good condition to ensure a good oiltight fit. To repair this counterbore, use the counterbore cutter, one of the special tools for servicing work. Fuel leakage into the camshaft chamber is often due to a defective fit in this counterbore.

- (b) Pumping elements
  - 1. If any scratch or scoring is noted on the cylinder bore under visual examination, replace it together with its plunger.
  - 2. Each suit of cylinder and plunger must be checked for the tightness of sliding fit by testing as follows: Hold the cylinder tilted, forming an angle of 60 deg, relative to horizontal, with its plunger inserted into its bore; pull out the plunger about 20 mm (0.8 in.); and release it to see if it slides down smoothly by its own weight. If it does in several angular positions with the cylinder so angled, the fit is satisfactory. Replace the cylinder and plunger as a suit if any stickiness or free-falling sliding motion is noted.



Testing fit of plunger in cylinder

- 3. Measure the width of plunger flange. If the flange is worn down to 6.95 mm (0.274 in.) or under in width, replace the plunger and cylinder.
- 4. Oil-tightness test

The cylinder and plunger found to be in satisfactory condition must undergo another test for checking the oil tightness of sliding fit between cylinder and plunger, after reassembling the injection pump. Before reassembling the pump, make it absolutely sure that the delivery valves are all in satisfactory condition as determined by the methods to be described subsequently.



Delivery valve test

Set up the pump on the tester, and install a test pressure gauge on one of the delivery valve holders. Prime the pump with fuel oil, and drive the pump at 200 rpm, with the control rack locked in idling position. The pressure gauge indication under this running condition tells whether the cylinder and plunger being tested is satisfactory or not: the pressure criteria are as follows:

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Item	Standard	Service limit
Oil-tightness of sliding fit between cylinder and plunger	150 ~ 200 (2133 ~ 2844) at 200 rpm	150 (2133)

Cylinder-plunger suits not capable of developing at least 150 kg/cm<sup>2</sup> must be replaced. For the test pressure gauge, use one calibrated up to 300 or 400 kg/cm2 (4266 or 5688 psi).

- (c) Delivery valves
  - 1. A delivery valve found with its piston worn excessively or valve seat showing evidence of poor seating contact or damage must be replaced together with its seat.
  - 2. Hold the suit of delivery valve and seat between two fingers, as shown, with the thumb plugging the bottom hole. Pull up the valve and let it go to see if the valve goes down smoothly but becomes arrested as its extraction land closes the bore; if it does, then the suit is satisfactory for the first test. For the second test, push the valve down till it touches the seat, and remove the push to see if the valve springs upward to the arresting position mentioned above; if it does, then the suit is satisfactory for the second test. A valve-seat suit found unsatisfactory in either test must be replaced. Remember, these two tests are meaningful only when the valve

and seat are absolutely clean and oiled adequately to form oil films on their surfaces.

Test each suit of delivery valve and seat in the foregoing manner and, when reassembling the pump, use only those found satisfactory.



Testing delivery valve and seat

3. Oil-tightness test

Test the delivery valve for oil-tightness of its seating contact after checking the cylinders and plungers for oil-tightness by running the reassembled pump at 200 rpm in the manner already described. It is assumed here that the cylinder and plunger served by a particular delivery valve are in good condition with respect to oil-tightness.

Run the pump at 200 rpm in the same testing setup, with the control rack kept in idling position, and, when the pressure gauge indication reaches 150 kg/cm<sup>2</sup> (2133 psi), move the rack into non-injection position ("0" mm) to see if it takes more than 5 seconds for this pressure to fall to 10 kg/cm<sup>2</sup> (142 psi); if it does, then the delivery valve under test is satisfactory.

(d) Control rack and pinions

A control rack in distorted condition or presenting excessive wear on its rack teeth must be replaced. After reassembling the pump, check each pinion for backlash and, as necessary, replace the pinion or control rack, or both, to reduce the backlash to the specification:

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ltem	Standard	Service limit
Rack-to-pinion backlash	0.15 (0.0059)	0.25 (0.0098)

(e) Tappets

Using a dial gauge, read the radial play of the roller of each tappet by moving the roller up and down in the manner illustrated: use a rod to push the roller up from bottom side. If a reading of 0.3 mm (0.012 in.) or more is obtained, replace the whole tappet.

Check the sliding clearance of each tappet in the bore of the housing; tappet replacement is necessary if the clearance noted exceeds the service limit.

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Item	Standard	Service limit
Tappet-to-bore	0.02 ~ 0.062	0.25
clearance	(0.0008 ~ 0.00244)	(0.0098)





(f) Delivery valve springs and plunger springs Replace badly rusted, cracked or otherwise damaged springs. Springs visibly out of square must be replaced. Check each spring for freestate length, and replace it if the limit, indicated below, is exceeded.

Unit: mm (in.)

	Plunger springs		Delivery springs	vaive
Item	Standard	Service limit	Standard	Service limit
Free length of spring	49.0 (1.929)	48.5 (1.909)	32.0 (1.260)	31.0 (1.220)

Inspect each plunger spring seat for depth of its concave; replace the seat if the depth is 0.1 mm (0.004 in.) or over.

(g) Camshaft

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Inspect the camshaft for wear of its cam surfaces, of the surfaces of sealed portions in contact with oil seals, and of screw threads at both ends. Inspect, too, for rusting, damage and for keyway deformation. Repair or replace the camshaft depending on the result of inspection.

Check the camshaft for alignment by supporting it with center pins fitted to its end faces, and by measuring the amount of deflection with a dial gauge. A camshaft exhibiting a deflection of 0.15 mm (0.0059 in.) or over at its middle section must be straightened in a press or replaced by a new one.

## (4) Assembly

(a) Install the cylinders (barrels), positioning each cylinder angularly as guided by the locating pin and positioning groove.



- a) These cylinders are meant for thumb-pressure fit. Never attempt to drive them into the bores with such as a mallet.
- b) After fitting each cylinder into the bore, making it seat firmly in place, check to be sure that it will not rotate when turned with a finger.



Installing pumping element cylinder

(b) Fit a new valve gasket to the delivery valve, and insert it into bore. Drive down the delivery valve by using a drift, as shown, making the seat meet the top of the cylinder to present an oil-tight face-to-face contact. Only light blows are needed to the end of the drift.



Take every precaution to avoid dust particles getting into between cylinder and delivery valve seat.



Driving in delivery valve seat

(c) Insert the spring, and run in the delivery valve holder and tighten the holder tentatively.



Installing holder

(d) Position the control rack in place, and install the guide screw.



Fitting guide screw



- a) Make sure that the guide screw installed is located accurately inside the groove provided in the rack. Tightening this screw located off the groove is liable to bend the rack.
- b) After installing the rack, check to be sure that it is capable of smooth movement through its entire stroke.
- (e) Lay down the pump, and fit control pinions and sleeves to barrels, making sure that each pinion is so positioned that the adjusting hole provided in the control sleeve points to the pinion clamp screw.



- a) Move the rack back and forth to rotate the control pinions similarly, and check to be sure that each pinion is accurately meshed with the rack teeth.
- b) Check, also, to see that each pinion is accurately centered to the control rack when the rack is in its center position, so that moving the rack from this position to each stroke end rotates the pinion by an equal amount.
- (f) Install upper spring seats and plunger springs. Using the plunger clamp, fit the lower spring seat to each plunger, and insert the plunger into the barrel.



Inserting upper spring seat and plunger spring

NOTES

- a) Be sure to hold each plunger true and square and insert it straight into the barrel.
- b) Remember, the driving face (on which a number is marked by punching) of each plunger comes on top side: if not, the helical groove of the plunger will not meet the feed hole and this defeats the adjustability of injection quantity.

c) After inserting the cylinder, position its lower spring seat in such a way that the seat will not fall off.



Inserting plunger

(g) Pick up the tappet with the tappet clamp, feed the tappet into the camshaft chamber and insert it into the bore.



Inserting tappet

Match the plunger driving face to the notches cut out in the control sleeve, push up the tappet and hold it there by means of the tappet insert. With the tappet so held, move the rack



Fitting tappet insert to hold up tappet

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to be sure it slides smoothly: Install all four tappets in this manner, making sure each time that the rack moves smoothly.

(h) Using the torque wrench, tighten the delivery valve holders to this torque value:

Item	Standard
Delivery valve holder	2.5 ~ 3.5 kg-m
tightening torque	(18~25 ft-lb)

## NOTE

After tightening up each delivery valve holder, move the rack to see if this tightening has adversely affected the ability of the control rack to slide smoothly.



Tightening delivery valve holder

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(i) Install the lock plate for locking the delivery valve holders in place. Using the spring balance, check the sliding resistance of the control rack. The rack is required to slide with a push or pull of not greater than 150 grams (5.3 ounces).



Checking sliding resistance of control rack

(j) Apply engine oil to the camshaft, and install it, positioning it in place with the marked end face coming on the drive side. (k) Put on the bearing cover tentatively, and check the axial play of the camshaft by using the camshaft clearance gauge (95905-01080). If the play noted is off the range indicated below, reduce or increase it into the specified range by shimming: shim stock for this purpose is available in six thicknesses, 0.10 mm (0.0039 in.), 0.12 mm (0.0047 in.), 0.14 mm (0.0055 in.), 0.16 mm (0.0063 in.), 0.18 mm (0.0071 in.) and 0.50 mm (0.0197 in.). So that the camshaft will not be so displaced by shimming to one side as to offset the cams from the tappets, try to use equal amounts of shim on both sides.

	Unit: mm (in.
ltem	Standard
Camshaft bearing axial play	0.03 ~ 0.05 (0.0012 ~ 0.0020)



Checking camshaft axial play

 Having properly installed the camshaft, apply BOND to the bearing cover and secure it permanently to the pump body, with an "O" ring set in the joint.



Applying BOND to bearing cover



a) Do not apply BOND to the "O" ring.

- b) Be sure that the ventilating hole of the cover comes on top side.
- (m) Install the screw plugs, tightening each plug to this torque value:

Item	Standard
Screw plug tightening	5.5 ~ 7.5 kg-m
torque	(39.8 ~ 54.2 ft-lb)

(n) Remove the tappet inserts one after another while turning over the camshaft. For the last time, check to be sure that the control rack is capable of smooth sliding movement, and check its sliding resistance to see and confirm that a force not greater than 150 grams (5.3 ounces) will move the camshaft.



Removing tappet inserts

## Engine proper

## Disassembly

Drain out engine oil and coolant. Wash the exterior surfaces of the engine clean, and set it up on the disassembly stand. After removing the various components attached to the engine proper, proceed as follows:

 Remove a total of 24 bolts securing oil pan (1) to crankcase, and take off the oil pan. Remove oil pump (2) as outlined in OIL PUMP REMOVAL.



1-Oil pan 2-Oil pump Removing oil pan

- (2) Remove the idler gear in the timing gear train.
- (3) Turn camshaft gear, bringing its two holes to top and bottom to expose bolts (1) securing thrust plate. Remove bolts (1), and draw the camshaft assembly out.



1-Bolt and washer (2 pcs each) Removing carnshaft (1)

NOTE

Remove the tappets after drawing out the camshaft.



Removing camshaft (2)

Using the puller (A), extract gear (2) from the camshaft. Remove thrust plate from the camshaft.

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2-Gear A-Puller 3-Thrust plate Removing gear from camshaft

(4) Remove bolts (1) securing the front plate (2) to crankcase, and take off the plate. The injection pump unit (3) comes off as mounted on the front plate.



1-Bolt and washer (2 pcs each) 3-Injection pump unit 2-Front plate

Removing front plate

- (5) Remove the flywheel.
- (6) Remove bolts (1) securing the rear housing (2) to the rear plate, and take off the housing.



Exercise caution not to damage the oil seal fitted to the rear housing.



1-Bolt 2-Rear housing Removing rear housing

- (7) Remove connecting rods, pistons and crankshaft, as follows:
  - (a) Remove 8 nuts fastening down the connecting rod caps, two nuts on each cap. Take off caps (1) complete with bearings.



1-Cap Loosening connecting rod cap nuts

(b) Lay down the crankcase, and draw out each piston and connecting rod.



2-Connecting rod 3-Piston Drawing out piston and connecting rod

(c) Raise the crankcase (bringing crankshaft to top side). Remove 10 bolts securing the main bear-



4-Main bearing cap Loosening main bearing cap bolts

CAUTIONS

- a) Handle the bearings carefully to avoid damage. Tag or otherwise identify each bearing shell and set aside the shells in identified groups so that each will be restored to its original location in reassembly.
- b) Remember, No. 1 and No. 5 bearings have two side seals each.
- (d) Lift the crankshaft (5) off and out of crankcase, as shown.



5-Crankshaft Removing crankshaft

(8) Remove the remaining main bearing shells (1). The idler shaft need not be removed unless to do so is absolutely necessary; the puller (A) must be used to draw out this shaft.



1-Bearing shell Taking off main bearing shells



(9) Using the special tool (A), extract crankshaft gear (1) from the removed crankshaft (2).



1-Gear 2-Crankshaft 'A-Puller Removing crankshaft gear

- (10) From each piston, separate its connecting rod, as follows:
  - (a) Using the piston ring tool (A), remove compression rings (1) and oil ring (2). Pick out oil ring spring with fingers.



1-Compression rings 2-Oil ring A-Piston ring tool Removing piston ring

(b) Using the snap ring tool (B), remove snap ring (3) from each end of the pin.



(c) Remove pin (4) by driving it out with a drift (special tool) (C), as shown:



(d) From the small end of connecting rod, remove the bushing.



Wash the disassembled parts clean, and decarbon them.

(11) Clean the crankcase by washing it in a bath of caustic soda solution or cleaning solvent, removing grease and grime from all surfaces in and out. Clean oil drillings and holes with a long-handle brush. After washing, dry it with steam or compressed air.



Reassembly

(1) Fit the bushing into the small end of each connecting rod by driving it in with a drift, making sure that the oil hole provided in the bushing meets the hole provided in the small end.



Combining piston and connecting rod

Have a snap ring fitted to one end of the piston pin hole, insert the small end into the piston, aligning the bushing to the pin hole, and insert the pin gently into the pin hole through the bushing from the end at which the snap ring is to be installed.

Be sure that, with the small end connected to the piston, the big end has its match mark located on the camshaft side, that is, opposite to the prechamber side. Secure the pin in place by fitting the other snap ring. The pin and pin hole are sized for loose fit and, therefore, the pin should go into the hole when given thumb pressure. This insertion will be made easier by having the piston warmed up in advance.

Using the piston ring tool, fit the piston rings (two compression rings and one oil ring) to the piston.



No. 2 ring has "R" mark on its top side. Be sure that this side is on top when the ring is in the groove.



"R" mark on No. 2 piston ring

When installing No.3 (oil) ring, be sure to combine ring (1) and expander (2) by matching the ring ends to the tube.



(2) After combining the connecting rods with their pistons, to which rings are fitted, insert woodruff keys (1) to the keyway provided in the forward end of crankshaft, and mount gear (2) on this end by driving it in with the installer (A). (This job will be easier if crankshaft gear (2) is heated hot.)



1-Woodruff key (2 pcs) A-Installer 2-Crankshaft gear Driving gear onto crankshaft

- (3) Fit the thrust plate to camshaft, and press camshaft gear onto camshaft. Have the gear heated hot to facilitate the job.
- (4) Install idler shaft (2) by driving it into the crankcase with the installer (A), as shown:



2-Idler shaft A-Installer Installing idler shaft

(5) Position the tappets in the respective tappet holes. Lightly oil the seats of main bearings, and fit the upper bearing shells to the seats, making sure that the tab formed of the shell fits snugly into the recess provided in the seat. Fit thrust plate (3) to the rear side of the seat.



3-Thrust plate Fitting upper bearing shells

- (6) Install the crankshaft, as follows:
  - (a) Lower crankshaft in level position and rest if on the five upper shells, which have been lightly oiled. Oil the five bearing caps, and fit the lower bearing shells, which have been oiled similarly. To the mating face of each cap, apply SUPER THREE BOND No. 10. Taking care not to damage thrust plates (3), put on No. 5 cap (2) and settle it firmly in place by tapping on it with a mallet (A). Put on No. 1, No. 2, No. 3 and No. 4 caps, and run down the bearing caps. Make sure that the outer end faces of No. 5 and No. 1 caps are flush with the outer faces of crankcase. (The face to be flush is indicated by the arrow.)



1-Crankshaft 3-Thrust plate 2-No. 5 bearing cap A-Mallet Fitting bearing caps

(b) Using a torque wrench, tighten the 10 cap bolts equally, tightening each just a little at a time, to the final torque value of 10 to 11 kg-m (72 to 80 ft-lb).



4-Cap bolt (10 pcs) B-Torque wrench Securing main bearing caps



Re-check to be sure that No. 1 and No. 5 caps have their outer end faces flush with crankcase end faces.

(c) Using a dial gauge, check the crankshaft end play to be sure it is within the specified range. Adjust the play, as necessary, by replacing thrust plates.

Unit: mm (in.)

ltem	Standard	Repair Jimit
Crankshaft end play	0.10 ~ 0.264 (0.0039 ~ 0.01039)	0.30 (0.012)

(7) Apply SUPER THREE BOND No. 10 to cap seals (1), and insert the seals into bearing cap grooves at No. 1 and No. 5 main bearing cap.



1-Cap seal Fitting cap seal



2-Main bearing cap 4-Rounded corner of cap seal Position of cap seal in groove



- a) Be sure to insert each cap seal with its round end foremost, bringing the rounded corner on outer side.
- b) After pushing in the cap seal, give a full thumb pressure to its end to settle it in place, taking care not to bend the seal. Never drive the seal with such as a hammer.
- (8) Lay down the crankcase, and insert the piston-androd combinations, as shown. At the crankshaft



1-Connecting rod bearing

side, fit bearing shells to the big ends and to the caps, oiling the shells as in the case of main bearing shells. Have the pistons and piston rings adequately oiled.

- (a) Distribute the ring joints, as shown, and feed the piston into the cylinder by using the piston guide (A), positioning the piston in such a way that the match mark provided on the connecting rod comes on camshaft side.
- (b) Secure the four caps by tightening the 8 cap nuts to 8.5 ± 0.5 kg-m (61.5 ± 3.6 ft-lb).



Precombustion-chamber side Configuration of piston ring joints



2-Piston A-Piston guide Inserting piston

NOTE

Before inserting the piston-and-rod combinations into the cylinders, have the cap bolts studded in the big ends. Be sure to match each cap to its big end as governed by the marks.



3-Cap 4-Nut (8 pcs) Tightening big-end cap nuts

(9) Check the side play-of each connecting-rod big end by barring it to one side and inserting a thickness gauge into the clearance. If the clearance (side play) measured exceeds service limit, the bearing shells or connecting rod must be replaced.

Unit: mm (in.)

Item	Standard	Service limit 0.50 (0.020)	
Big end side play	$0.15 \sim 0.35$ (0.0059 $\sim 0.0138$ )		

(10) Attach the front plate (1) (on which the injection pump is mounted), and secure the plate to crankcase by tightening its two mounting bolts (2) to 2.1 kg-m (15.2 ft-lb).



1-Front plate 2-Bolt (2 pcs) Securing front plate to crankcase

(11) Insert the camshaft gently into crankcase. Secure the camshaft thrust plate to crankcase by tightening its bolts with the wrench put to each bolt head through the hole provided in camshaft gear: the holes are indicated by the arrows.



Installing camshaft



Securing camshaft thrust plate to crankcase

- (12) Fit the oil pump assembly to crankcase. Make sure that drive gear and camshaft gear are correctly meshed. Secure the pump in place as outlined in OIL PUMP INSTALLATION.
- (13) Position the oil pan on crankcase, and secure it by tightening its mounting bolts evenly: there are 24 bolts to be tightened.



Oil seal in rear housing



When fitting the rear housing to the rear plate, be careful not to allow the oil seal lip to fold over, making its spring to come off. The lip portion is indicated by the arrow.

(14) Drive dowel pin (1) into crank gear. Drive ball bearing into flywheel (2). Secure flywheel to crankshaft through the crank gear by tightening the four mounting bolts to 8.5 ± 0.5 kg-m (61.5 ± 3.6 lb-ft), with the lock washers inserted under bolt heads. Lock the bolts securely by bending the washers positively.



Installing flywheel

## Starter

Removal and installation

 Disconnect from the starter the lead wire (1) connecting starter to the battery. Disconnect, also, the two wires (2) connecting starter to the starter switch.



1-Starter-to-battery wire 2-Starter-to-switch wire (2 pcs) Disconnecting electrical wires from starter

(2) Remove mounting nut (3) and bolt (4). Take down the starter assembly (5) from the engine.



3-Nut and washer

4-Bolt and washer 5-Starter assembly Removing starter

Disassembly and reassembly

## 

1-Bolt and washer (2 pcs each) 2-Switch 3-Plunger 4-Bolt and washer (2 pcs each) 5-Rear bracket 6-Yoke 7-Center bracket 8-Front bracket 9-Armature 10-Lever 11-Lever spring 12-Rubber packing 13-Overrunning clutch 14-Pole piece

Starter - Exploded view

- Remove two bolts (1) securing the magnetic switch to front bracket (8), and take off the switch (2).
- (2) From front bracket (8), remove plunger (3) complete with rubber cover.



1-Bolt and washer (2 pcs each) 2-Magnetic switch Removing magnetic switch



3-Plunger with rubber cover Removing plunger

To install the starter, carry out the above two steps in reverse order.

4

NOTE

(3) Remove two bolts (4) from rear bracket (5).



4-Bolt (2 pcs) Removing bolts securing rear bracket

(4) Detach front bracket (8) from yoke (6) by tapping lightly on the bracket. Separate rear bracket (5) from yoke (6).



Separating brackets from yoke

(5) Take out front bracket complete with rotor and pinion from yoke. Remove center bracket (7) from front bracket by removing the bolts securing it to front bracket.



7-Center bracket Removing center bracket

(6) From front bracket (8), draw out the rotor, that is, armature (9), rubber packing (12), lever (10), spring (11), overrunning clutch (13) complete with pinion.



Separating front bracket from rotor

 Remove overrunning clutch (13) from armature (9).



Removing clutch from armature

# CAUTION

Before drawing off the center bracket, tape the splined portion of the shaft so that the oil seal will not suffer damage as the bracket is moved along the shaft for removal.

(8) To reassemble the starter, reverse the foregoing sequence of disassembling steps and refer to the following cross section:

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When installing the lever, be careful not to position it the other way around. The correct position is clearly recognizable in the cross section.





Alternator and regulator unit

## Alternator removal and installation

- (1) Disconnect wires between alternator and regulator unit and disconnect ground harness (2). Loosen adjusting plate bolt (3) and remove fan belt.
- (2) Loosen alternator bracket bolts (4) and remove alternator assembly (5) from engine.
- (3) To install the alternator, follow the reverse of removal procedure.



1-Wires between alternator and regulator unit 2-Ground 3-Adjusting plate bolt and washer 4-Bolts, nuts and washers (2 pcs each) 5-Alternator assembly

Alternator disassembly and reassembly



2-Front bracket 3-Rotor

5-Rear bracket 6-Pulley

9-Brush

Alternator - exploded view

(1) Pulling out through bolts (1) will permit removal of rotor (3) (with front bracket and pulley) and stator coil (4) (with rear bracket).

1



(2) Hold rotor in a vice and remove pulley by loosening pulley clamping nut.



Assemble alternator in the reverse order of dis-(3)assembly. Place brush in brush holder and secure it by passing a pin through a small hole vacated by removing screw as shown. After assembly, be sure to pull out the pin.



(4) Apply a coating of sealer to the mating surfaces between the stator and rubber packing and those between the front and rear brackets and rubber packing.

NOTE

Care should be taken to install the rubber packing to the stator properly.



Regulator unit removal and installation

The regulator unit is mounted on the inboard side of the dashboard. To remove the regulator unit, undo the coupler (1), remove mounting screws (2) and detach it from the dashboard. Reverse these steps to install the regulator unit.



Regulator unit disassembly and reassembly

For the purpose of servicing the regulator unit, the only thing to be done is to remove its cover. Its internals are not meant for disassembly.



Regulator unit - Disassembled view

# MAINTENANCE STANDARDS

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## MAINTENANCE STANDARDS

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Unit: mm (in.)

Group		Part or iten	1	Nominal dimension	Assembly standard [Standard clearance]	Repair limit [Clearance]	Service limit [Clearance]	Remedy	Remarks
	Maximum rpm (no-load)		2640, max.				Adjust governor.		
	Minimum rpm (no-load)		650 ~ 700						
General	Compression pressure			20 kg/cm <sup>2</sup> (284.4 psi), min. (at 150~200 rpm)	16 kg/cm <sup>2</sup> (227.5 psi)				Oil and water temperatures: 20 $\sim$ 30°C (68 $\sim$ 86°F)
				3~4 kg/cm <sup>2</sup> (43~57 psi) (at 1500 rpm)	2 kg/cm <sup>2</sup> (28,4 psi)				Oil temperature:
	Engine oil pressure		1.0 kg/cm <sup>2</sup> (14.2 psi) min. (at idle speed)	0.5 kg/cm <sup>2</sup> (7.1 psi)	5 kg/cm <sup>2</sup> (7.1 psi)		70°C (158°F)		
	Valve timing		Open	30° BTDC					
		intake vaiv	Close	50° ABDC	± 3°				
		Parts and and	Open	3 74° BBDC		1	5		
		Exnaust va.	Close	30° ATDC					
	Beginning of injection		25°±1 (Crank angle)						
		LD.		94 (3.701)	+ 0.035 ~ 0 (+0.0014 ~ 0)	+0.20 (+0.008)	1,20 (0.047)	Hone sleeve to 0.25 (0.0098) or 0.50 (0.0197) oversize with prescribed tolerance. Oversize pistons and piston rings should be used.	Four sleeves should be finished to the same oversize. Hone cylinder bore to $98 \stackrel{-0.010}{-0.045} \stackrel{-0.00039}{-0.00177}$ and heat the crankcase. Press sleeves into crankcase and machine each sleeve 1.D. to assembly standard,
	Cylinde	er Out-of-r	oundness		0.015 (0.0006), max.				
parts		Tapér							
noving	Pistons, protrusion above crankcase gasketed surface				0.35 ~ 0.75 (0.0138 ~ 0.0295)			Check bearing clearance.	
Main r	pu	No. 1 compression ring			$[0.04 \sim 0.08 \ (0.0016 \sim 0.0031)]$	0.20 1 <sub>(0.0079)</sub> ]			(1) Ring side clearance Measure side clearance with ring kept flush with second land. (2) Replace oil ring together with expander.
	iston rings a			2.0 (0.079)	$0.025 \sim 0.060$ $1_{(0.0010 \sim 0.0024)}$	0.15 [(0.0059)]		Use pistons by replacing piston rings up to service limit. Replace pistons when service limit is reached.	
	Oil ring			4.0 (0.157)		(0.0023)			

137

Unit: mm (in.)

Group		Part or item	Nominal dimension	Assembly standard [Standard clearance]	Repair limit [Clearance]	Service limit [Clearance]	Remedy	Remarks
Main moving parts	60	No. 1 compression ring						
	n rin	No. 2 compression ring		0.30~0.50	I (0.059) l			When oil ring is compressed to 94 (3.701).
	Pisto gaps	Oil ring		·(0.0118 ~ 0.0197)·				
	Piston pins	Clearance in pistons	28 (1.102)	$1^{0}_{(0 \sim 0.006)}$	0.05 (0.002)		Use pistons by replacing piston pin, up to repair limit.	
		Clearance in connecting rod bushings		$\begin{bmatrix} 0.020 \sim 0.051 \\ (0.0008 \sim 0.0020) \end{bmatrix}$	$[ { 0.08 \atop (0.003) } ]$		Replace pistons or bush- ings. (Ream if necessary.)	
	Pistons, variance in weight per engine			±3g (±0.1 oz)				
	Connecting rods	Variance in weight per engine		±5g (±0.18 oz)				
		Clearance on crankpin O.D. (big end bearing I.D.) (in two directions at right angles to each other with bearing in place)	58 (2.283)	$[0.035 \sim 0.100 \\ (0.0014 \sim 0.0039)^{1}$	0.20 [ <sub>(0.008)</sub> ]		Use connecting rods by replacing bearings, up to repair limit. Regrind crankpins and use under- size bearings when repair limit is reached.	Cap must be installed with marks on cap and rod on the same side.
		End play	40 (1.575)	$[0.15 \sim 0.35 \ (0.006 \sim 0.014)]$		(0.50 (0.020)	Replace connecting rods or bearings.	
		Bent and twist		0.05 (0.002)	0.15 (0.006)			
	ankshaft	Center-to-center dimen- sion between journals and crankpins	47 (1.850)	•±0.05 (±0.002)				
		Parallelism between crank- pins and journals		0.01 (0.0004), max as runout				
		Out-of-roundness of crank- pinz and journals		Variance in dia.:	0.03 (0.0012)		31	
	ð	Taper of crankpins and journals		0.01 (0.0004), max				
		Fillet radius	3 (0.12)	± 0.2 (0.008)				
		Variance in crankpin angles		± 20 <sup>1</sup>				
crank- shaft	Runout (measure and 4th journals "V" blocks	ed with 1st held in		0.02 (0.0008), max	0.05 (0.0020)		Straighten or replace crankshaft,	
---	---	--	--	--	--	--	--	--
Main bearings	Clearance on tran journal (in two direction angles to each of bearing in place)	nkshaft 15 at right ther with	75 (2.953) $ _{(0.0020 \sim 0.0045)}^{0.05 \sim 0.115}$		0.20 1 <sub>(0.008)</sub> 1	-0.9 (-0.035) (at crankshaft journal O.D.)	Replace bearings unless repair limit is reached. Regrindcrankshaft journals and use undersize bearings 0.25 (0.0098) or 0.50 (0.0197) when repair limit is reached.	Replace crankcase and bearing caps as an assembly.
	Crankshaft end play		2,45 (0.097)	$0.1 \sim 0.264$ [(0.0039 ~ 0.01039)]	[0,30 [(0.012)]		Replace thrust bearing.	
Flywh	eel, face runout a	nd flatness		0.15 (0.006), max	0.50 (0.020)			Install dial gauge to flywheel and measure face runout with respect to flywheel housing rear face.
Crankcase, warpage on gasketed surface				0.05 (0.003)	0.2.00.0095		Regrind if warpage is	
cad	Warpage of gasketed surface			0.05 (0.002), max	0.2 (0.008)		minor.	
er he	Valve seat angle		45°					
/lind	Valve sinkage		0.7 (0.028)	±0.2 (±0.008)	1.3 (0.051)			
0	Valve seat width	1	1.2 (0.047)	±0.14 (±0.0055)	1.6 (0.063)			
Valve cylind	guides, protrusion er head gasketed s	above surface	17 (0.669)	±0.3 (±0.012)				
Cyline thickr	ler head gasket, as iess	-installed	1.4 (0.055)	± 0.05 (± 0.002)			3	
	Clearance of	Intake valves	0.0010	$0.055 \sim 0.085 \ (0.0022 \sim 0.0033)^{1}$		0.15 (0.0059)	Replace both valve guide	
~	guide	Exhaust valves	8 (0.315)	$[0.070 \sim 0.100 \\ (0.0028 \sim 0.0039)]$		[0.20 [0.0079)]	limit is reached.	
Valve	Margin		1.5 (0,059)			1.2 (0.0472) after refacing		
	Sinkage		0.7 (0.028)	±0.2 (±0.008)		1.3 (0.051)		
	Face runout of	Face runout of head		0.03 (0.0012), max (per- pendicular to valve face)				
	Valves Valves Crank Colinder head Cylinder head Shaft Shaft	Runout (measure and 4th journals "V" blocks         Set       Clearance on crajournal (in two direction angles to each of bearing in place)         Flywheel, face runout a crankcase, warpage on place         Set       Warpage of gask surface         Valve seat angle       Valve seat width         Valve seat width       Valve seat width         Valve suides, protrusion cylinder head gasket, as thickness       Clearance of valve stem in guide         Set       Margin         Sinkage       Face runout of	NumberRunout (measured with 1st and 4th journals held in "V" blocksSettingClearance on crankshaft journal (in two directions at right angles to each other with bearing in place)Flywheel, face runout and flatnessCrankshaft end playFlywheel, face runout and flatnessCrankcase, warpage on gasketed surfaceWarpage of gasketed surfaceValve seat angle Valve seat widthValve seat widthValve guides, protrusion above cylinder head gasketed surfaceCylinder head gasket, as-installed thicknessMargin Sinkage Face runout of head	Runout (measured with 1st and 4th journals held in "V" blocksSet und (in two directions at right angles to each other with bearing in place)75 (2.953)Clearance on Crankshaft journal (in two directions at right angles to each other with bearing in place)75 (2.953)Flywheel, face runout and flatness2.45 (0.097)Flywheel, face runout and flatness2.45 (0.097)Crankcase, warpage on gasketed surface2.45 (0.097)Valve seat angle45°Valve seat angle45°Valve seat angle0.7 (0.028)Valve seat width1.2 (0.047)Valve guides, protrusion above cylinder head gasket, as-installed thickness1.4 (0.055)Valve stem in guideIntake valvesSet MarginIntake valvesMargin1.5 (0.059)Sinkage0.7 (0.028)Face runout of head0.7 (0.028)	$\begin{array}{c c c c c c c c c c c c c c c c c c c $	$\frac{1}{100} \frac{1}{100} \frac{1}$	$ \frac{1}{90} $	$\frac{1}{90} \underbrace{\frac{1}{90}}{10} \frac$

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								Quint. thus fu
	Part or iter	n	Nominal dimension	Assembly standard [Standard clearance]	Repair limit [Clearance]	Service limit [Clearance]	Remedy	Remarks
	Free length		48.85 (1.923)		47.6 (1,874)			
ve springs	As-installed len	gth	43 (1.693)		44 (1.732)		Adjust by means of shim(s) when repair limit is reached,	
Val	Squareness			0,4 (0,016)/25 (0.98), max.				Squareness of each end with respect to center line.
Valve clearance		0.25 (0.0098) (cold setting)		0.12			0.25 clearance may also be obtained by warm setting if intake and exhaust valves are at the same temperature.	
Tapp	ets, fit in crankca	se	22 (0.8661)	$0.035 \sim 0.086 \ 1_{(0.0014 \sim 0.0034)}$	0.12 [ <sub>(0.0047)</sub> ]	+0.10(+0.004) (at tappet hole dia.)	Replace tappet unless repair limit is reached.	
Valve	push rods, bend			0.4 (0.016), max.				
Rock shaft	er arms, fit on roo	sker	20 (0.787)	$[0.016 \sim 0.055]{(0.0006 \sim 0.0022)}]$	[0.07 [(0.0028)]		Replace bushing unless repair limit is reached.	
	Clearance of Ne in hole	Clearance of No. 1 journal in hole Clearance of No. 2 journal in hole					Install bushing when repair limit is reached, (Ream if necessary.)	
	Clearance of Ne in hole			e of No. 2 journal		$\begin{bmatrix} 0.040 \sim 0.090 \\ (0.0016 \sim 0.0035) \end{bmatrix}$	0.15 l <sub>(0.0059)</sub> l	
ų	Clearance of No in hole	o. 3 journal	53 (2.087)					
Camsha	Clearance of th thrust journal	rust plate on	5 (0.197)	$\begin{bmatrix} 0.05 \sim 0.112\\ (0.0020 \sim 0.0044) \end{bmatrix}$	$\begin{bmatrix} 0.3\\(0.0118)\end{bmatrix}$ ]		Replace thrust plate.	
	Cam height Intake		46.916 (1.8471)	$^{+0.1}_{-0.3}$ ( $^{+0.00394}_{-0.01181}$ ) D1-D2 = 6.684 (0.26315)		$D_1 - D_2$ = 6.184 (0.24346)		
		Exhaust	45.944 (1.8088)	$^{+0.1}_{-0.3}$ $^{+0.00394}_{-0.01181}$ D <sub>1</sub> - D <sub>2</sub> = 7.344 (0.28913)		D1 - D2 = 6.844 (0.26945)	Replace camshaft.	
	Cam dia.		40.232 (1.5839)					
	Runout			0.02 (0.0008), max,	0.05 (0.0020)		Straighten or replace.	

0,1 ((0.0039)

0.35 [(0.0138)]

Replace bushing.

Replace thrust plate.

 $0.025 \sim 0.075$  $(0.00098 \sim 0.00295)$ 

 $\begin{bmatrix} 0 \sim 0.1 \\ (0 \sim 0.004) \end{bmatrix}$ 

36 (1.417)

Unit: mm (in )

140

Group

Timing gear train

Idler

Clearance of bushing on

shaft

End play

and and	1							
g gear	Idler	Clearance of shaft in bore in crankcase	30 (1.181)	$1_{(0.00035T \sim 0.00177T)}^{0.009T \sim 0.045T}$				
Timin train	Timin	g gear, backlash		$0.07 \sim 0.20$ (0.0028 $\sim 0.0079$ )	0.25 (0.0098)		Replace gear.	
		Clearance of main shaft in body	13 (0,512)	$\left[ \begin{smallmatrix} 0.032 & \sim 0.074 \\ (0.00126 & \sim 0.00291) \end{smallmatrix}  ight]$		0.15 [ <sub>(0.0059)</sub> ]	Replace pump case.	
/stem	dun	Clearance of outer rotor in body	50 (1.969)	$[0.20 \sim 0.28 \ (0.0079 \sim 0.0110)]$		[0.5 [(0.0197)]		
tion sy	Oll p	Rotor and cover end play	30 (1.181)	$[0.04 \sim 0.09 \\ (0.0016 \sim 0.0035)]$	0.15 [(0.0059)]		Reface case cover or case,	
ubrica		Inner rotor and outer rotor clearance		$[0.013 \sim 0.15 \\ (0.0005 \sim 0.0059)]$		0.25 l <sub>(0.0098)</sub> 1		
-	Relief valve	Relief pressure	3.0 kg/cm <sup>2</sup> (42.7 psi)	± 0.3 kg/cm <sup>2</sup> (± 4.27 psi)				Oil pressure varies 0.15 kg/cm <sup>2</sup> (2.133 psi) per 1 (0.04) of shim thickness.
		Clearance of outer race in	47 (1.850)	0.011L~0.025T			Replace pump case or	
	SBL	pump casing	40 (1.575)	(0.0004 L ~ 0.0010T) <sup>1</sup>			pump assembly.	
	st pump bearb	Clearance of inner race on pump shaft		$\begin{bmatrix} 0.001T \sim 0.017T \\ (0.00004T \sim 0.00067T) \end{bmatrix}$				
шe	Wat	Padial play	17 (0.669)	$\left  \begin{array}{c} 0.010 \sim 0.025 \\ (0.00039 \sim 0.00098) \end{array} \right $		,0.045	Barlasharin	Replace bearing when it
ayste		Radiai piay		$\begin{bmatrix} 0.010 \sim 0.022 \\ (0.00039 \sim 0.00087) \end{bmatrix}^{1}$		<sup>1</sup> (0.00177) <sup>1</sup>	Replace bearing.	does not rotate smoothly.
Coolir	Space	r, I.D.		0.001T~0.017T (0.00004T~0.00067T)			:	
	Water casing sides)	pump impeller, vane-to- clearance (front and rear	$0.5 \sim 1$ (0.020 ~0.039)				Replace impeller or bearing if vanes are binding.	
1	ti si	Carbon protrusion	1.5 (0.059)		0			75/
	Un	Height (free length) 21.8 (0.858)		±1 (±0.04)			-	<ul> <li>1-Floating seat (carbon)</li> <li>2-Seal ring (ceramic)</li> </ul>
	stat	Valve opening temperature	76.5°C(169.7°F)	± 2°C (±3.6°F)				
	Thermo	Valve opening temperature         76.5 °C(169.7°           Valve lift         9 (90°C)           I0.35 (194°F)						

.

Group		Part or item	Nominal dimension	Assembly standard [Standard clearance]	Repair limit [Clearance]	Service limit [Clearance]	Remedy	Remarks
Cooling system	Belt,	tension	Deflection: 12 (0.472)					
		Injection pressure	120 kg/cm <sup>2</sup> (1706.4 psi)	± 5 kg/cm <sup>2</sup> (± 71.1 psi)	110 kg/cm <sup>2</sup> (1564.2 psi), min,		Adjust by means of shim(s).	Injection pressure varies 10 kg/cm <sup>2</sup> (142.2 psi) per 0.1 (0.004) of shim thickness.
	Injection nozzles	Spray angle	0°				Test by means of hand tester, using diesel fuel at, 20°C or 68°F approx. If spray pattern is im- proper even after nozzle is washed in clean kerosene, replace nozzle tip.	Spray of fuel oil should be uniform and consist of fine droplets.
		Needle valve seat oil- tightness	Seat s 100 k	hall show no sign of leakage g/cm <sup>2</sup> (1422 psi).	Wash needle valve seat or replace nozzle tip.	Replace nozzle tip when needle surface is scratched or scored.		
		Clearance of tappet roller pin in pin hole		$(0.013 \sim 0.071) (0.0005 \sim 0.0028)^{1}$		Total play:		
ystem		Clearance of tappet roller on roller pin	7 (0.276)	$\begin{bmatrix} 0.033 \sim 0.085 \\ l_{(0.0013 \sim 0.0033)} \end{bmatrix}$		10.3 (0.0118), max.j	-	
uel s		Roller OD	15 (0.591)	0~-0.027 (-0.00106)	-0.0075 (-0.00295)			
u.	feed pump	Airtightness Leakage b not more		ll show sign of leakage, ween rod housing should be in 50 cc (3 cu în.)/mîn.	See Assembly Standard,			Close pump discharge port with a plug. Apply an air pressure of 2 kg/cm <sup>2</sup> (28 psi) to the pump, and keep the pump immersed in diesel fuel longer than 1 minutes.
	Fuel	Number of strokes for pumping (priming pump)		25, max.	30		:	Operate priming pump handle at a speed of 60 strokes/minute. Check the number of strokes requied for making the pump start dis- chaging at a head of 1 meter.
		Number of strokes for pumping		45 seconds, max.	50 seconds			Operate injection pump at 150 rpm to check the length of time required for the feed pump to start discharging.

Unit: mm (in.)

MAINTENANCE STANDARDS

Fuel feed	dund	Capacity			900 c min.	c (55 cu in.)/min	, 600 cu ir mīn.	cc (37 1.)/min,				Check displacement with injection pump operated at 1000 rpm with a discharge ptessure of 1.5 kg/cm <sup>2</sup> (21 psi)	
		Cam heij	tht	32 (1.260)	) (	$0 \sim +0.1$ $0 \sim +0.0039)$			-0.2 (-0.0079)	Replace camshaft.		Check cam surface for condition.	
	Ĩ	Axial pla bearing	y of camshaft		l <sub>(0.0</sub>	$0.03 \sim 0.05$ $012 \sim 0.0020)$	{0.1 (	0.004)		Adjust by n shim.	neans of		
	Ī	Clearanc tappet (I	e of tappet pin in nole)		0.0) [(0.0	$013 \sim 0.050$ $005 \sim 0.0020)^{1}$							
		Clearanc bushing	e of tappet floating on tappet pin	7 (0.276)	0.0) I (0.0)	$033 \sim 0.078$ $013 \sim 0.0031)^{1}$			Total play: [0.3 (0.0118), max.]	Replace tap plete.	opet com-		
		Clearanc on floati	e of tappet roller ng bushing	11 (0.433)	1 <sub>(0.0</sub>	$050 \sim 0.097$ $020 \sim 0.0038)^1$							
		Tappet r	oller OD	17 (0.669)	0 (0	$0 \sim -0.027$ (0 $\sim -0.0011$ )			-0.075 (-0.00295)				
-		Plunger	Free length	49 (1.929)	+1 ~	-0 (+0.039~0)			-0.5(-0.020)				
du	di l	springs	As-installed length	44 (1.732)	)								
ion pu	ind uor	Backlash between pinion and rack			[0	0.15 (0.0059)]			0,25 (0,0098)				
Fuel inject	L net Tuhect	Rack sliding resistance		150 g (5.3 50 g (1.8 1000 rpm	l oz) with pu oz) with pun	mp at standstill ap running at							
		Free length of delivery valve spring		32 (1.26)		: 0.5 (± 0.02)			-1 (-0.04)				
		Plunger pre-stroke		1.95 (0.07	7) ±(	0.05 (± 0,002)							
		Fuel injection quantity			Pump speed	Rack position	Strokes mm	Injectio cc (	on quantity cu in.)	Variance cc (cu in.)			
					1000	8.5 (0.335)	200	7.) (0.4	$2 \sim 7.8$ $4 \sim 0.48$	0.4 (0.02)		Mount injection pump on	
				I	1000	8.0 (0.315)	200	6. (0.3	4 ~ 7.0 9 ~ 0.43)	0.4 (0.02)		pump to inject.	
					200	6.0 (0.236)	500	5. (0.3	0~8.0 1~0.49)	1.0 (0.06)			

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I down	1000	11-1	
Unit:	mm	(111,)	

Group		Part or iter	m	Nominal dimension	Assembly standard [Standard clearance	Repair [Cleara	llmit Service limit ince) (Clearance)	Remedy	Remarks
	Fuel injection pump	Fuel injection of	pantity		Test conditions Nozzle tip: Injection pipe: Injection pressur Delivery pressur Test oil:	ND - DN( 6 x 1.6 x re: 120 kg/cn c: 2.0 kg/cn ASTM D	0SD 600 (1/4 x 1/16 x 23- m <sup>2</sup> (1706.4 psi) n <sup>2</sup> (28.44 p <b>si)</b> iesel fuel No. 2	5/8)	
		Clearance of sw lever shaft in bu	riveling ushing	. 11 (0.433)	$\begin{bmatrix} 0.016 \sim 0.07 \\ (0.0006 \sim 0.0028) \end{bmatrix}$		[0.15 (0.0059)]		Replace bushing or swivel lever. Replace "O" ring and oil seal when replacing bushing.
		Clearance of ten lever or guide le support pin in p	nsion wer pin hole	8 (0.315)	0.013~0.05 (0.0005~0.0020)		(0.10 (0.00394)		Replace pin and lever if pin hole is worn abnormally or worn to show any stepped portion.
		Clearance of control lever shaft bushing on shackle connecting pin		5 (0,197)	$[0.005 \sim 0]{(0.0002 \sim 0)}$		[0.10 (0.0039)]		Replace control lever or shackle.
		Clearance of shi in control rack	ackle pin hole		$\binom{0.015 \sim 0.056}{(0.0006 \sim 0.0022)^{1}}$		[0.08 (0.0031)]		Replace shackle or control rack.
el system	ernor	Guide lever and control block							Replace control block or guide lever if chromed tip of control block is worn down.
μŢ	cal gov	Control spring							Replace spring if hook is badly worn.
	Mechani	Flyweight roller O.D.		16 (0.630)	-0.11~0 (-0.0043~0)		-0.25 (-0.0098)		
	4	Clearance of flyweight roller on roller pin		0.0.2121	$(-0.0043 \sim 0)$ $0.025 \sim 0.062$ $(0.00098 \sim 0.00244)^{1}$	1	0.10		Replace Hyweight.
		Clearance of to lever bushing or	rque cont 1 support	rol pin	$0.026 \sim 0.056 \\ 1_{(0.00102 \sim 0.0022)}$		<sup>1</sup> (0.00394) <sup>1</sup>		Replace lever.
			Step	ltem	Adjusting lever angle	Pump rpm Np. rpm	Co	ntrol-rack position RW, mm	
			1	High-speed control	47°± 5°	1100 1230 1290	$\begin{array}{c} 8.2 \pm 0.1 \ (0.3228 \pm 0.1) \\ 7.0 \pm 0.1 \ (0.2756 \pm 0.1) \\ \leq 4.0 \ (0.1575) \end{array}$	0.0039) at full-load stopp 0.0039) at initial lever set	er Iting
			2	Control by adaptor action	51°±5°	400 650	10.4 ± 0.1 (0.4094 ± 8.8 ± 0.1 (0.3465 ±	0.0039) 0.0039)	

	1.	•		3 1	Low-speed (	control	High-speed control lever angle MINUS 26° ± 3°	200 275 330	≧ 11 5.5 ±0 5.0 ±0	$ \stackrel{\geq}{=} 11 (0.4331) \\ \stackrel{\scriptstyle{(0,4331)}}{\scriptstyle{(5,5 \pm 0,1 (0.2165 \pm 0.))}} \\ \stackrel{\scriptstyle{(0,1069 \pm 0.)}}{\scriptstyle{(0,1969 \pm 0.)}} $		iitial lever 1 sub-sprin	setting ng	
2	sor				•		Adjusting lever angle	Pump tpm Np. rpm	Total stroke	il injection qt. cc (cu in.)/ 500 ces, 4 cyl.				
uel system	anical govern			4 N	Matching w	th engine	51 <sup>°°</sup> ± 5 <sup>°</sup>	600 1200	81 ± 3 62 ± 3	2 (3.1890 ± 2 (2.4409 ±	0.0787) with a 0.0787) with t	ring		
ιL.	Mech	×		5 5	Stop lever o	peration	The stop lever must control lever at VH.	operate piecise Np = 1200 rpr	dy, When ( n, the rack	the stop leve k must be dr	er is pulled with rawn up to Rw	h the high = 0 mm	-speed	
					No-lo	ad operation	1	Loa	d operatio	on	Magnet	ie switch o	operating voltage	
				Volta	age (V)	Current (A)	AS00	ige (V) Cu	rrent (A)	A kerm (	De Switt	ch IN	When circuit	
	1			2	23	50, max.	min.	9 71	10, max.	ft-lb), m	in. 16V.	max.	is opened,	
				Rea	ar	Rear	Intermediate metal	Commutate	-00			Daugh		
	et		Photo In		1 5.8 - 1 - 1	1		A STREET STREET STREET	11			Dinzu		
	tter		Shaft	Fro	ont Meta	Front	Bracket shaft	service limi	t	He	right	Wear	Spring compression,	
trampinpa	Starter		Shaft 14.2 (0.559	-0.050 -0.077 -0.002 -0.002	0 7 14. 20 30) (0.55	H Front 2 +0.027 -0.016 9 +0.0011 -0.0006	Bracket shaft 20.6 +0.02 -0.08 (0.811 +0.0008 -0.0031)	43.2 (1.701	t	He 19 (0.75)	right	Wear	Spring compression, kg (lb) 2 (4.4)	
Electrical equipment	Starter		Shaft 14.2 (0.559 12.2 (0.480	-0.050 -0.077 -0.002 -0.003 -0.003 -0.050 -0.077 -0.003 -0.003	Ont         Meta           0         14.           20         (0.55           0         12.           20         (0.48	$\begin{array}{c c} & & & \\ & & & \\ & & \\ 2 \end{array} + 0.027 \\ & -0.016 \\ 9 \end{array} + 0.0011 \\ & -0.0006 \\ 2 \end{array} + 0.027 \\ & -0.016 \\ 0 \end{array} + 0.0011 \\ & -0.0006 \\ \end{array}$	Bracket shaft 20.6 +0.02 -0.08 (0.811 +0.0008 -0.0031) 20.3 0 -0.033 (0.799 0 -0.0013)	service limi 43.2 (1.701 2 (0,079	)	He 19 (0.75) Service lim	ight ilt: 13 (0.51)	6 (0.236)	Spring compression, kg (lb) 2 (4,4) Service limit: 1.5 (3.3	
Electrical equipment	Starter		Shaft 14.2 (0.559 12.2 (0.480	-0.050 -0.07 -0.001 -0.001 -0.001 -0.001 -0.001 -0.001 -0.001 No-lo	Ont         Meta           0         14.           20)         (0.55           0         12.           20)         (0.48           20)         (0.48	$\begin{array}{c c} & Front \\ \hline Front \\ 2 + 0.027 \\ -0.016 \\ 9 + 0.0011 \\ -0.0006 \\ \hline 2 + 0.027 \\ -0.016 \\ 0 + 0.0011 \\ -0.0006 \\ \hline \end{array}$	Bracket shaft 20.6 +0.02 -0.08 (0.811 +0.0008 -0.0031) 20.3 0 -0.033 (0.799 0 -0.0013)	service limi 43.2 (1.701 2 (0.079	) )	He 19 (0.75) Service lim	ight ilt: 13 (0.51)	6 (0.236) Rear	Spring compression, kg (lb) 2 (4,4) Service limit: 1.5 (3.3	
Electrical equipment	lator Starter		Shaft 14.2 (0.559 12.2 (0.480	-0.050 -0.07 -0.003 -0.003 -0.003 -0.003 -0.003 -0.003 -0.003 -0.003 -0.003	Ont         Meta           0         14.           20         (0.55           0         12.           20)         (0.48           20)         (0.48           cond operation         connecter	H Front 2 +0.027 -0.016 9 +0.0011 -0.0006 2 +0.027 -0.016 0 +0.0011 -0.0006 0 +0.0011 -0.0006	Bracket shaft 20.6 +0.02 -0.08 (0.811 +0.0008 -0.0031) 20.3 0 -0.033 (0.799 0 -0.0013) [battery + resist	2 (0,079 coad operation ance load 12.5	) ) ) ) ) ) ) ) ) ) ) ) ) ) ) ) ) ) )	He 19 (0.75) Service lim	ight ilt: 13 (0.51) Ball bearings	6 (0.236) Rear Front	Spring compression, kg (lb) 2 (4.4) Service limit: 1.5 (3.3 Field resistance 20°C (68°F)	
Electrical equipment	dternator Starter		Shaft [4,2 (0.559 [12,2 (0,480 (0,480	Fro -0.05( -0.07 -0.002 -0.000	Ont         Meta           0         14.           20         (0.55           0         12.           20)         (0.48           20)         (0.48           cond operation         Current           (A)         Current	H Front 2 +0.027 -0.016 9 +0.0011 -0.0006 2 +0.027 -0.016 0 +0.0011 -0.0006 0 m d) Rpm	Bracket shaft 20.6 +0.02 -0.08 (0.811 +0.0008 -0.0031) 20.3 0 -0.033 (0.799 0 -0.0013) (battery + resist Terminal voltage(V)	service limi 43.2 (1.701 2 (0,079 coad operation ance load 12.5 Current (A)	t ) ) ) SA connec	He 19 (0.75) Service lim	ight it: 13 (0.51) Ball bearings #620	Rear Front	Spring compression, kg (lb) 2 (4.4) Service limit: 1.5 (3.3 Field resistance 20°C (68°F) 27 (Ω)	

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								Unit: mm (i
Group		Part or item	Nominal dimension	Assembly standard [Standard clearance	Repair limit [Clearance]	Service limit [Clearance]	Remedy	Remarks
					Voltage regulato	r Lamp rei	ay	
ent				Air gap	1.0~1.4 (0.039~0.055)	0.9 ~ 1 (0.035 ~ 0	.4 .055)	
quipm	or unit			Contact point gap	0.3 (0.012), min	0.75~1 (0.030~0	,1 .043)	
trical e	Regulato			Back gap	0. (0.0	75 ~ 1.1 30 ~ 0.043)		
Elect				No-load voltage (V)	27.5 ~ 30.5 at 300 rpm (alternator spe	0 ed)		
1				Cut-out voltage (V)		6.5 ~ 7	.5	
				Cut-in voltage (V)		5, max.		
							:	

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	1101116:41	io ionqui	Unit: kg-m (ft-lk		
Item	Torque	Item	Torque		
Cylinder head bolts	12 ± 0.5 (87 ± 3.6)	Oil pan drain plug	10.0 ± 0.5 (72.3 ± 3.6)		
Main bearing cap bolts	$10.4 \pm 0.5$ (75.2 ± 3.6)	Oil pump connector	5.5 ± 0.5 (39.8 ± 3.6)		
Connecting rod cap bolts	8.5 ± 0.5 (61.5 ± 3.6)	Nozzle holder retaining nuts	5.0 ± 0.5 (36.2 ± 3.6)		
Flywheel bolts	8.5 ± 0.5 (61.5 ± 3.6)	Injection pump delivery valve holders	3.0 ± 0.5 (21.7 ± 3.6)		
Camshaft thrust plate bolts	1.8 ± 0.5 (13 ± 3.6)	Flywheel housing bolts	3.5 ± 0.5 (25.3 ± 3.6)		
Front plate bolts	$1.8 \pm 0.5$ (13 ± 3.6)				
Timing gear case bolts	3.5±0.5 (25.3±3.6)		1		
Crankshaft pulley nut	40 ± 0.5 (290 ± 3.6)				
Idler thrust plate bolt	3.5 ± 0.5 (25.3 ± 3.6)				
Oil pan bolts	0.7 (5.1)				

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# TIGHTENING TORQUE

Unit: kg·m (ft-lb)

Apply to:	Mating part	Sealer
Main bearing caps	Crankcase bearing caps	SUPER THREE BOND No. 5
Side seals	Crankcase bearing caps	SUPER THREE BOND No. 5

### APPLICATION OF SEALERS

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# TROUBLESHOOTING

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A loss of a section

# TROUBLESHOOTING CHART (1)

1			-	Engine	will not s	tart			A	E	ingine l	acks	A	bnorma lust sm	al ex- oke		1000		Eng	tine 11s
	Complaint	E) by	ngine to ut does ot start	urns	not	Engin not tu	e does irn	low red	too carl	ke		aust		Whe	n ating	sively				
F	Possible cause	No exhaust smoke	A little exhaust smoke	Too much exhaust smoke	Starting motor does turn sufficiently to crank engine	Engine can be cranked manually	Engine cannot be cranked manually	Glow plugs do not g	Glow plugs glow red	A little exhaust smo	Too much whitish exhaust smoke	Too much black exh smoke	When idling	Whitish exhaust smoke	Black exhaust smoke	Engine knocks exces	Engine is noisy	Engine runs rough	When idling	When operating
	Insufficient fuel supply to injection pump	0	0	-	-	-	-	-	-	0	-	-	-	-		-	-	-	-	-
	Greater variance of injec- tion quantity	-		o		-	-	-	-	-	-	0	0	-	0	0	Ŧ	0	0	0
	Defective injection pump seals	-	-	-		-	-	-	-	-	-		-	-	-	-	-	-	-	-
	Insufficient injection quantity	0	0		-	-	-	-	-	0		-	-	-	-	-	-	_	-	-
	Excessive injection quantity	-	-	-	-	-		-	-	-	-	-		-	0	-	-		-	-
	Improper fuel spray from injection nozzles	-		0	-	·	-	-	-	-	-	o	o	-	0	0	-	0	0	0
EI0	Excessive fuel return from injection nozzles	-	0		-	-	-	-	-	0		-	-	-		-	-	0	0	-
el syst	Injection timing too advanced	-	·**	0	-	-	-	-		-	-	0	-		0	0	-	-	=	-
Ĩ.	Injection timing too retarded	-	-	0	-		-	-		-	0	-	0	٥	-	-	-	0	0	1
	Defective auto timer	-	-	-	-	-	-	-	-	-	-	0	0	-	D	0	0	-		-
	Defective governor control spring	-	-	-		-		-	-	-		-	-	-	-	-	-	-	-	-
	Maladjusted governor damper spring	-	-		-	-	-	-	-	-	-		-	-	-	-	-	-	0	0
	Engine speed too low	-	-	-		-	-	-	-	-	-		-	-	-	-	-	-	0	-
	Failure of engine to stop properly	-	-		-	-	-	-	-	•=		-	-	-	-	-	-	-	-	-
	Poor grade of fuel oil	-		0	-	-	-	-	-	-	0	0	0	0	0	0	-	-	-	
	Fuel viscosity too high	0	0	-	-	=		-		-	-	-	-	-		-	-	-		-
	Poor grade of nil	-				-	-			-	-	- 1	-	-	-	-	-	-	-	-
	Oil viscosity too high	-	-	-	-	0	-	-	-		-	-	-	-		-	-	-		-
	Oil viscosity too low	-		-		-	-		-	-	-	-	-	0	-	-	~		-	-
tem	Low oil pressure	-	-	-			-	-	-	-	-	-	-	-	-	-	-	-	-	-
Ast	Excessive oil leakage			-	-	-		-	-	-	-		-	-	-	-	-	-		
ation	Pumping up of oil	-	-	-	-	-	-	-	-		-	-	-	0	0	0	-			
Drice	Clogged oil filter	-		-		-	-		-	-	-		-	-	-	-	-	-	-	
La	Defective oil bypass alarm or lamp	-	-		-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-
	Defective oil indicator switch or lamp	-	-	-	-	-	-	-	-	-	-	-	-	=	-	-	-	-	-	
E	Insufficient air		-	0		-		-	-	-	-	0	-		O	-	-	-	-	
rstor	Poor compression	-	-	0	-	-		-	-	-	0	0	0	G	O	0		0	0	0
Air sy	Low pressure at high atomospheric temperature (or altitude)	-	-	-	-	-	-	-	-	-	-	0		_	Q	-	-	-	-	-

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TROUBLESHOOTING

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	Remedy	0	0	Check and replace if necessary.	0	٥	6	Replace nozzle tips.	٥	0		uncok and replace spring it nocessary. Then adjust governor cetter on heach	Adjust.	Adjust idling set bolt. Hald lever in STOP position.		٥	Use fuel for cold weather.	Use good quality oil.	Use proper viscosity oil.	Use proper viscosity oil.		keugnten and replace packing if necessary.		Change element and oil.	Check and replace if necessary.	Check and replace if necessary.	0	0	(Adjust full-load setting of governor.)
ive tor	Indicator lamp comes on at high-speed operation	1	I	1	1	ł	I	ļ	1	l	1	ł	ł,	I	I	I	1	I	I	I	I	I	I	1	ť	1	1	1	1
Defect alterna	Indicator lamp does not come on when engine stops	I	1	ł	I	1	I	I	ł	l	T	I	1	1	1	I	1	1	I	١	L	ł	I	l	t	1	1	1	ł
mal tem-	Water temperature too	1	T	I.	1	t	1	I	I	1	1	I	T	L	1	I	t	T	I	1	1	t	I	ł	1	I	1	1	I
Abnor water	Water temperature too high (Engine overlicats)	1	ľ	l	Ţ	0	I	I	I	0	ł	1	1	ţ	t	t	1	1	ţ	ţ	Ĭ	ł	Ĩ	1	1	1	I	1	0
-Ū-	no somoo qmal gainteW noitstoqo booqe-ilgid ta	4	I	1	1	I	I	1	ŀ	1	1	ł	T	ł	1	1	Ţ	1	0	I	Ĩ	1	1	ø	0	ĺ	1	1	1
ive lub system	Warning famp comes on at low-speed operation	1	1	1	1	1	ŧ	t	I	ł	ł	1	1	o	1	I	1	1	4	0	ó	I	I	1	I.	0	1	1	1
Defection	When engine stops, warning lamp docs not come on	1	i	1	1	1	ł	I	1	I	ł	I	1	I	I	I	I	1	1	1	i	ţ.	I	I	I	0	1	1	1
	Excessive blow-by	1	1	Ļ	1	I	1	1	1	1	1	1	1	I.	1	I	1	1	L	0	1	1	1	l	1	1	1	J	1
	lio ni 1936W	1	I	1	1	I	1	1	1	1	1	1	1	T	1	1	T	1	1	1	1	1	1	ł	I	T.	1	1	1
	Isul yd borulib si llO	1	I	0	1	0	0	ŧ.	1	1	1	1	1	1	1	1	1	1	1	1	1	ï	1	Ţ	I	1	1	1	1
	Excessive oil consumption	1	1	1	-1	1	I	I	1	1	I	ï	I.	1	1	1	1	1	1	0	L	0	0	1	1	ł	1	1	1
1	Excessive fuel consumption	1	Ó	1	1	Ò	0	0	0	O	0	1	1	1	1	O	1	1	I	1	1	L	1	. 1	1	1	Ö	0	0
uojio	Engine turns in reverse due	1	1	1	I	1	1	T	1	I	1	1	1	1	0	1	1	1	1	T	1	1	1	Î	1	1	1	1	i
spo	Engine operates at high spe- but fails to stop	1	1	I	i	1	Ţ	1	I	1	1	ľ	1	1	1	Ţ	1	1	I	1	ľ	1	١	1.	ł	1	1	ł	1
paa	Engine does nut pick up spe	o	1	1	0	I	1	1	L	1	1	0	1	1	1	t	1	1	Ĩ	1	1	1	1	1	1	1	10		1
uor	Poor response for decelerat	1	1	1	1	1	1	1	1	I	I	0	0	1	1	1	1	1	1	1	ł	1	1	1	1	T	1	1	1
	Engine stalls	0	0	1	0	1	0	0	0	0	1	ł	0	O	1	0	1	1	1	1	1	1	1	1	1	1	0	0	1
	guthus Approves excessively	1	0	I	I.	1	0	1	0	1	1	1	1	0	- E	1	1	T	1	1	1	1	1	1	1	1	1	1	T

-

1	V-			Engine	will not s	tart		P	why	E	ngine i ower	lacks	A	bnorm tust sm	al ex- loke				Eng	gine ats
	Complaint	E: bi	ngine ti ut does ot start	urns	s not	Engin not tu	e does m	glow re	d too e	ake .	ł			Whe	n rating	ssively				Γ
Ρ	ossible cause	No exhaust smoke	A little exhaust smoke	Too much exhaust smoke	Starting motor doe turn sufficiently to crank engine.	Engine can be cranked manually	Engine cannot be cranked manually	Glow plugs do not	Glow plugs glow re	A little exhaust sm	Too much whitish exhaust smoke	Too much black exhaust smoke	When idling	Whitish exhaust smoke	Black exhaust smoke	Engine knocks exce	Engine is noisy	Engine runs rough	When idling	When operating
	Engine is too cold.	-	-	0	0	-	-	-	-	-	-	-	0	-	-	0	-	-	-	-
E	Radiator dissipates heat excessively	_	-		-	_	<u>a</u>	_	-	-	0	-	0	0	-	0	-	-	-	-
yste	Insufficient coolant	-			_	-	_	-	-	-	-	0		-00	o	-	-	-	-	-
aling s	Failure of radiator to dis- sipate heat properly	-	_	-	-	-	-	-	-	-	-	٩	-	-	0.	-	-	-	3000.	
ö	Water leak through cylinder head gasket	-	-	-	-	-	-	-	-	-	0	-	0	0		-		0	-	-
	Cracks in crankcase water jacket	-	-		-	-	-	_	-	-		-		-	-	-	-	-	-	-
	Defective starter switch	+	-		-	0	-	0	-	-	-		~	-71	-		-	-	-	-
	Defective starter magnetic switch	-	=	-	- 1	0	-		and t	-	-	-			-	-	-	-	-	-
	Defective starting motor	-	-	-	0	O	-	-	-	-	-	-	-	-	-100	-	-	-	-	
	Defective starting motor free wheel	-	-	-	0	о	-	-	-	-		-			-	-		-	-	
system	Defective flywheel ring gear and pinion	-	-		-	0	-	-	-	-	-	-				-	-	-	-	-
ri ça	Battery voltage drop	-	-	Ø	0	0	-	0	-						-	-	-	-	-	-
Elect	Open circuit in glow plugs or pilot lamp	-	-	C		-	-	0	-		57 <b>-</b> 56	-	-		-	-	-	-	-	<del></del>
	Short circuit in glow plugs	-	-	0	-	-	-	-	0		-	-		-	-	-	-	-	-	
	Defective alternator	-	-	-	-	-	-	-								-	-	-		
1	Defective alternator relay	-	-	-	-	-	-	-		-		-	-	-	-200	-10			-	-
	Improper wiring		-	0	0	0	-	-	0	-	-	-				-	-	-	-	-
	Jammed moving parts	17	-		0	-	0	-			-	0	-		0	-	0	0		-
	Worn cylinders, pistons or piston rings	-	-	0	-			-7	-	-	0	-	0	0		0	0	-	-	
1	Sticking piston rings	-	-	0	-	-	-	-	-		0	-	0	0	-	0		-	-	
aerts	Excessive main bearing clearance	-	-				-	-	-	-	-	-			-	-	0	-	-	$\left -\right $
oving p	Loose connecting rod cap bolts		-	-	-	-	-	-	-	-	-	-	-	-	<u>1942</u> 2	-	0	<u>ب</u>	-	-
lain m	Interference between valve and piston	-	-	-	-	-	0		-	-	-	-	-	-		-	0	-	-	-
N	Broken valve springs	-		0		-		-		-	0	-	0	0	-	-	0	0	0	0
	Excessive valve clearance	-		-	-		-		-	-		0	-	-	0	-	0	0	0	0
	Foreign substances in cylinders	-	-	-	-	-	D	-	-	-	-	-	-	-	-	-	O.	-	-	-
	Excessive gear backlash	-				-	-		-	-	-		-		-	-	0	-	-	-

### TROUBLESHOOTING

( . For detailed information refer to the separate chart.)

×		ation	peed	spaar	rection	uo	-				Defec cation	tive lu 1 syster	bri- n	Abnor water	tem-	Defect alterna	ive itor	
Engine vibrates excessivel	Engine stalls	Poor response for deceler	Engine does not pick up a	Engine operates at high si but fails to stop	Engine turns in reverse du	Excessive fuel consumpti-	Excessive oil consumption	Oil is diluted by fuel	Watet in oil	Excessive blow-by	When engine stops, warning lamp does not come on	Warning lamp comes on at low-speed operation	Warning lamp comes on at high-speed operation	Water temperature too	Water femperature too low	Indicator lamp does not come on when engine stops	Indicator lamp comes on at high-speed operation	Remady
-	0	-	Ļ	-	-	-	-	-	-	-	-	-	0	-	-	-	_	Heat oil pan from bottom side at starting. After starting, warm up engine thoroughly.
-	-		-	-	-	0	-	-	-		-	-		-	0		~	0
			-	-	-	-	0	-	-	0	-	0		0	-	-	-	0
-	-		-	-	-	-	0	-		0	-	0	-	o		-		0
-		-	_	-	-	-	-	-	0	-	-		-	-		-	-	Retighten and replace gasket if necessary.
-	-	-	-		-	-	-	_	0		-		-	-		-		Replace crankcase.
-	-	-	-		-	-	-	-	-	-	-	-	-	-	-	-	-	Check for connections and repair. Replace if necessary.
-	-	-	-	-	0	-	-	-	-		-		-			-		Repair or replace if necessary.
-	-	-	-		-	-	-	-	-	-	-		-		-	-	-	Repair or replace.
-	-	-	-	-	-	-	-	-	-	-	-	_	-		-	-		Repair or replace starter if necessary.
-	_	-	-	-	-	-	-	-	-	_		-	-		-	-	-	Repair or replace ring gear. Replace pinion.
-	-	-	-	-	0	-	-	-	-	-	o	-	-		-	0	-	Recharge or replace battery. If necessary heat it.
-	-	-	-	-	-	-	-	-	-	-	-				-	-	-	Replace.
-	-	-	-	-	-	-	-	-	-	-	-	-	•***	-	-	-	-	Replace copper packings and if necessary glow plugs.
-			-			-	-	-	-	-			-			-	0	Replace alternator if necessary.
-		-	-		0	-14	-	-	-	-		-	-	-	-	o	Q	Adjust or replace.
-	-	-	-	-	-	-	-	-	-	-	0	-		-		0	-	Connect wires properly.
-	0	-	0		-	0	0	-	-	0	-	-	-	0	-	-	-	0
-	0	-	-		-	0	0	0	-	0	-	-	-	-	-	-	-	Repair or replace.
-	o	-		-	-	0	0	0	-	0		o	0	-		-	-	Repair or replace.
-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	Check and replace bearing(s) with undersize one(s) if necessary.
-	-	-	-		-	-	-	-	-	-		-		-	-	-	-	Retighten.
-	-	-	-	-	-	-	-	_	-	-	-	-	-	-	-	-	-	Re-time timing gear train or adjust valve sinkage properly.
0	0	-	-	3.H=	-	-	-	-	-	-	-	-	-	-	-	-	-	Replace.
o	0	-	-		-	-	-	-	-		-	7	-	-	-	-	-	Adjust valve clearance to 0.25 mm (0.0098 in.).
-	-	-	-	-	-	-	-	-	-		-		-		-	-	-	Repair.
-		-	-	•••	-	-	-	-	-	-	-	-		-		-	-	Replace gears or idler bushing.

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Pos	ssible cause	Remedy
	Closed fuel tank supply cock	Open the cock.
	No fuel in tank	Fill fuel tank,
	Clogged fuel lines	Repair or clean fuel pipes with air.
	Air in fuel system	Retighten connections or replace fuel pipes.
Poor fuel supply to fuel injection pump	Clogged fuel feed pump inlet strainer	Remove and clean strainer.
	Defective fuel feed pump	Repair.
	Leaky fuel lines or connections	Retighten connections.
	Clogged fuel filter	Disassemble and clean filter.
	Air in fuel system	Bleed air out of fuel system
	Jammed plungers	Replace plungers.
	Jammed tappets	Replace tappets.
	Worn plungers	Replace plungers.
	Worn tappet cams	Replace tappet cams.
Craster variance of injection	Worn bearings	Replace bearings.
quantity	Worn or broken plunger springs	Replace plunger springs.
• *** 41 too •	Loose plunger pinions	Reinstall properly by matching marks.
	Defective delivery valves	Replace valves.
	Worn or broken delivery valve springs	Replace valve spring.
4	Governor stop lever link is binding.	Straighten link, placing lever in neutral position.
	Control rack is sticky.	Relubricate or repair.
	Jammed plungers	Replace plungers.
	Jammed tappets	Replace tappets.
neufBalant iniantian quantity	Worn plungers	Replace plungers.
insurficient injection quantity	Worn tappet cams	Replace tappet cams.
	Worn bearings	Replace bearings.
	Loose plunger pinions	Reinstall properly by matching marks.
	Improper full-load setting of governor	Adjust governor setting on bench.
	Stop lever jammed in "increase" position.	Repair lever link , placing lever in neutral position.
Excessive injection quantity	Improper full-load setting of governor	Adjust governor setting on bench.
	Loose plunger pinions	Reinstall properly by matching marks.

# TROUBLESHOOTING CHART (2)

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Pos	sible cause	Remedy						
Improper fuel spray from injection nozzles	Sticking needle valve in one or more nozzles Damaged nozzle tip seats Worn or broken nozzle springs Too low an injection pressure Carbon deposited on packings at nozzle tips	Repair and replace nozzles if necessary. Repair and replace seats if necessary. Replace nozzle springs. Adjust pressure to $120 {}^{+10}_{0}$ kg/cm <sup>2</sup> (1706.4 ${}^{+14}_{0}$ psi) on nozzle tester by inserting shim(s). Remove carbon.						
Injection timing too advanced	Improper installation of fuel injection pump Incorrect installation of timing gears	Re-time properly by tilting injection pump away from engine. Re-time timing gear train.						
Injection timing too retarded	Improper installation of fuel injection pump Incorrect meshing of timing gears Worn cams, tappets or bearings in fuel injection pump	Re-time properly by tilting injection pump toward engine. Re-time timing gear train. Replace.						
Poor grade of fuel	. Water in fuel	Use good-quality fuel. Use good-quality fuel.						
Low oil pressure	Lack of oil in oil pan Air in oil strainer Defective oil pump Clogged fuel lines Defective oil pressure regulating valve Clogged oil filter Leak in lubricating system	Add oil to prescribed level. Replace damaged pipes or packings. Retighten loose connections if any. Repair. Clean. Repair and replace if necessary Change element and oil. Repair.						
Pumping up of oil	Excessive oil in oil pan Wom cylinders, pistons or piston rings Sticking piston rings Worn valve guides Prolonged operation under no load	Drain oil to lower oil level. Repair and replace parts if necessary Repair and replace damaged rings if necessary. Replace valve guides. Overhaul engine if oil-laden gases are exhausted.						
Insufficient air	Clogged air cleaner (Paper element type) Clogged air cleaner (Oil bath type) Clogged intake manifold	Clean and replace element if Wash interior. Check and clean.						

	Possible cause	Remedy
	Defective valve seats	Repair
	Sticking valve stems	Repair and replace valve stems if necessary .
	Failure of rocker arms to lift valves	Adjust valve clearance to 0.25 mm (0.0098 in.).
Poor compression	Worn cylinders, pistons or piston rings	Repair and replace parts if necessary.
	Sticking piston rings	Replace damaged rings.
	Exhaust gases leak through cylinder head gasket	Retighten and replace gasket if necessary.
	Worn or broken valve springs	Replace springs.
	Insufficient torque of starting motor (at starting)	Replace parts.
	Insufficient coolant	Add water to prescribed level.
	Overheating	With the management
Insufficient coolant	water leaks from unit seal of water	Replace seal.
	Crack in crankcase water jacket	Replace crankcase.
	Water leaks from other parts	Check and repair.
	Air in cooling system	Bleed air by loosening air bleed plug, drain plug, or hose clamp.
Failure of radiator to	Rust and scale deposited in radiator	Flush.
dissipate heat properly	Dust and dirt around radiator	Flush.
	Slippage of fan belt	Adjust belt tension.
	Inoperative thermostat (kept closed)	Replace.
	Extremely low atmospheric temperature	
Radiator dissipates heat	Uncovered radiator	Cover radiator.
even and the state of the state	Inoperative thermostat (kept open)	Replace.
	Sticking cylinders, pistons or piston rings	Repair and replace if necessary.
Jammed moving parts	Sticking main bearings and crankpin bearings	Repair and replace if necessary.
	Sticking cam bushing and idler bushing	Repair and replace if necessary.

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# SERVICE TOOLS



Ref. No.	Part number	Tool	Qt.	Used for:
1	34491-00400	Valve guide installer	1	Valve guide installation
2	31391-10500	Valve guide remover	1	Valve guide removal
3	31391-13010	Valve seat caulking tool (intake valve)	I	Valve seat installation and caulking
4	31391-13020	Valve seat caulking tool (exhaust valve)	1	Valve seat installation and caulking
5	30691-11100	Adaptor	I	Test pressure gauge installation
6	34491-00200	Piston guide	1	Piston installation
7	MH061077-01	Idler shaft puller	1	Idler shaft removal
8	30691-11800	Cranking handle	1	Engine cranking
9	34491-00300	Socket	1	Camshaft thrust plate removal and installation
10	34491-00100	Cylinder sleeve installer	1	Cylinder sleeve installation
11	30091-01101 or commercially-available	Universal extension	1	Injection pump removal and installation
12	31391-12900 or commercially-available	Piston ring tool	i	Piston ring removal and installation
13	64309-12900 or commercially-available	Puller assembly	I	Removal of crankshaft gear, camshaft gear, crankshaft pulley and water pump pulley
14	30091-07300	Idler bushing puller	1	Idler and camshaft bushing removal and installation
15	30891-04500 30891-04600	Adaptors	2	Idler and camshaft bushing removal and installation

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# Tool application chart

1	Part No.	Used for: Taol	Installing valve guides	Removing valve guides	Installing and caulking intake valve scat	Installing and cautking exhaust valve seat	Installing test pressure gauge	Inserting pistons into cylinders	Drawing out idler shaft	Turning over crankshaft	Installing and removing camshaft thrust plate	Installing cylinder sleaves	Tightening rocker bracket bolts (short)	Installing and removing fuel injection pump	Removing and installing piston rings	Removing crankshaft gear	Removing camshaft gear	Removing crankshaft pulley	Removing water pump pulley	Removing and installing idler shaft bushing	Removing and installing camshaft bushings when necessary
1	34491-00400	Valve guide installer	I								1										
2	31391-10500	Valve guide remover		1									1								
3	31391-13010	Valve seat insert caulking tool (intake valve)			1																
4	31391-13020	Valve seat insert caulking tool (exhaust valve)				1															
Ś	30691-11100	Adaptor					1			1											
6	34491-00200	Piston guide						I													
7	MH061077-01	ldler shaft puller							1												
8	30691-11800	Cranking handle		1						1											
9	34491-00300	Socket									1		1								
10	34491-00100	Cylinder sleeve installer	Ì									1									
11	30091-01101 or commercially- available	Universal extension												1							
12	31391-12900 or commercially- available	Piston ring tool													1						
13	64309-12900 or commercially- available	Puller assembly														1	1	1	1		
14	30091-07300	Idler bushing puller																		1	1
15	30891-04500 30891-04600	Adaptors																		1	1

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### Valve guide installer and remover

1-Remover

2-Valve guide

Removing valve guide

3-Cylinder head

The installer is for use in driving the valve guide into the guide hole in such a way that the guide will take its prescribed position. The remover is for driving the guide out of the hole.



1-Installer 3-Cylinder head 2-Valve guide A-17mm (0.67 in.) Installing valve guide

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Valve seat caulking tools

These tools are for use in driving valve seat (2) into cylinder head (3) and, after fitting the valve, in caulking the peripheral edge of the valve. There are two caulking tools, one for intake valve and the other for exhaust valve. Each comes with a caulking ring, and is to be used in two directions, one for driving and the other for caulking.





Valve seat caulking tool (exhaust valve) (part number: 31391-13020)

How to use



Ref. No.	Part number		Tool
3	31391-13010	Valve seat caulking tool	(intake valve)
4	31391-13020	Valve seat caulking tool	(exhaust valve)

### Adaptor

This is a connection fitting to be used in installing the test pressure gauge in the glow plug hole for the purpose of reading the compression pressure.





### Piston guide

Use this tool when inserting the piston into cylinder. It protects pistons and piston rings against damage, and facilitates the feeding in of the piston.

Test pressure gauge installed with the adaptor





Inserting piston into cylinder

# Idler shaft puller

For drawing out the idler shaft, one of the timing gear parts.





Idler shaft puller (part number: MH061077-01)

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Set the puller over the idler shaft (4), run the stud (3) into the shaft, and drive nut (5) to force the shaft out by jacking action.

# Cranking handle

This tool is for turning over crankshaft, as in valve clearance adjustment and others. Its hexagonal hole fits the crank pulley securing nut.



Cranking handle (part number: 30691-11800)

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Turning over crankshaft

### Socket

This tool is for use in tightening the bolts to secure the camshaft thrust plate (through the holes provided in camshaft gear), and also the shorter ones of the bolts for securing the rocker brackets. It can be used in torquing the glow plugs, too.





Socket (part number: 34491-00300)



Securing the camshaft thrust plate



Tightening shorter bolt on rocker bracket

### Cylinder sleeve installer

For use in driving replacement sleeve into the cylinder.



Cylinder sleeve installer (part number: 34491-00100)

### How to use

Insert the sleeve installer (2) into sleeve (1), and push on the top end of the installer with a press arbor to force the sleeve into cylinder (3).

# Universal extension

The fuel injection pump unit is mounted on engine front plate, as secured by two bolts. This extension tool is for bringing the wrench head to these bolts.



Pressing replacement sleeve into cylinder



Universal extension (part number: 30091-01101)





9-Bolt and washer Tightening injection pump mounting bolts

### Piston ring tool

For use in fitting piston rings to and in removing them from the piston.



Piston ring tool (part number: 31391-12900)

How to use



Removing piston rings

### Puller assembly

For use in removing crankshaft pulley, water pump pulley, crankshaft gear, camshaft gear and injection pump gear.



Puller assembly (part number: 64309-12900)

# Idler bushing puller

For use in removing and installing idler bushing and camshaft bushings, when necessary.





Idler bushing puller (part number: 30091-07300)

# Adaptor

For use in removing and installing idler bushing and carnshaft bushings, when necessary.

